

# Thermal cooling analysis and validation of the ytterbium doped double clad fiber laser by a general analytic method

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## ABSTRACT

Long optical fiber is usually coiled on the surface of the substrate. Spreading thermal resistance is an important factor affecting thermal properties of optical fiber in this encapsulation way. This paper adopts a concise general analytic method to calculate spreading thermal resistance of the eccentric heat source formed in the rectangular channel. The on-site testing of active doped double clad fiber (DDCF) laser is also carried out. It is observed that the output light efficiency is high after the fiber loses heat through the plate which conforms to the needs of normal fiber laser light output efficiency of 70%–80%. The analytic value is in line with the changing trend of the experimental value, with the error of the temperature measurement points within  $\pm 8.8\%$ . The third segment rules out the influence of the optical fiber coupler location and the temperature of coupling point. When the pump power ranges from 4.4 W to 31.07 W, the error range between the experiment and analytics is within  $-4.14\%$  to  $-0.82\%$ . The layout plan of the fiber has a certain effect on the temperature uniformity of the fiber laser. It is demonstrated that the analytic method and mathematical assumption are reliable in the cooling design of the optical fiber.

## 1. Introduction

High power laser, with its high reliability, efficiency, and low cost characteristics, has a wide range of applications in communication and industry in recent years [1–3]. Making use of double clad fiber doped with rare earth ions is one of the advisable approaches to optimize the output power for single-mode fiber laser [4–6]. Several kilowatts of laser power output can be implemented by effectively coupling strong pump light source into the inner cladding. Continuous single-mode output power of 9.6 kW of single fiber by using all fiber main oscillation power amplification (MOPA) structure in 2009 has been reported [7].

The strong pump light source caused severe thermal dissipation when fiber laser used slender doped fiber as gain medium to realize several kilowatts of power output [8]. The reason is that the rising temperature in the doped fiber core will reduce quantum efficiency [9]. On the other hand, thermal stress from heat diffusion of host material [10] and the change of refractive index give rise to the thermal lens effect, and even the glass melting. According to the research [11] on optical fiber damage, the outer layer will be aging or even burn to lose its role, if the work temperature of outer cladding layer exceeds 80 °C, thus causing adverse impacts on the stability and lifetime of the fiber

laser. It is shown that the problem of optical fiber cooling is extremely serious. As a consequence, effective cooling measures should be taken to reduce the thermal damage of high power fiber laser.

Many scholars have been engaged in solving fiber laser heat dissipation problems. S. Tokita et al. adopted a hybrid cooling mode, which combines water cooling in the length direction with convection cooling of fiber's end face on the erbium doped fiber ZBLAN that can effectively improve the stability of high power optical fiber [12]. L. Li et al. set a thermoelectric cooling device on the fiber and tried to use experimental methods to acquire the internal temperature distribution of high power fiber laser [13]. Simple cylindrical symmetry coordinates are usually adopted to obtain the temperature distribution inside the optical fiber [14,15]. Numerical simulations are used to study the multimode fiber or more complex fiber microstructure [14,16]. To a certain extent, these results depend on the accuracy of the boundary condition assumption and theoretical model. Thermal coupling simulations of the long fiber and cooling device are also limited by computing resources in the meantime. Experimental method needs ready-made device to obtain experimental data, and it is difficult to demonstrate the whole temperature field of the cooling system.

There is no simple and reliable thermal design method currently due

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**Nomenclature**

$a, b, c, d$	dimensions of the plate and heat source area, m
$A$	baseplate area, $m^2$
$A_0, A_1, A_2, A_3, B_0, B_1, B_2, B_3$	fourier coefficients
$A_c$	fiber core cross-section area, $m^2$
$A_s$	heat source area, $m^2$
$Error_{T_{Ana}-T_{Exp}}$	error between analytic value and experimental value, %
$Error_{T_{Ana}-T_{Simu}}$	error between analytic value and simulation value, %
$h_l$	heat transfer coefficient at lower surface of plate, $W/(m^2 \cdot K)$ or $W/(m^2 \cdot ^\circ C)$
$k_l$	thermal conductivity of plate, $W/(m \cdot K)$ or $W/(m \cdot ^\circ C)$
$k_c$	thermal conductivity of fiber core, $W/(m \cdot K)$ or $W/(m \cdot ^\circ C)$
$k_{in}$	thermal conductivity of inner cladding layer, $W/(m \cdot K)$ or $W/(m \cdot ^\circ C)$
$k_{out}$	thermal conductivity of outer cladding layer, $W/(m \cdot K)$ or $W/(m \cdot ^\circ C)$
$K_f$	thermal conductivity of fiber, $W/(m \cdot K)$ or $W/(m \cdot ^\circ C)$
$L$	length of fiber, m
$m, n$	indices for summations
$N$	number of heat sources
$N_{dc}$	doping concentration in the fiber core, $m^{-3}$
$N_2$	upper level population
$P_p^l$	forward pump power, W
$P_p^r$	backward pump power, W
$P_p^\pm(z_f)$	forward and backward pump power in the axis $z_f$ , W
$P_s^\pm(z_f)$	forward and backward signal light power in the axis $z_f$ , W
$q$	heat flux, $W/m^2$
$Q$	total power of heat source, $W/m^3$
$Q(z_f)$	heat power density in fiber in the axis $z_f$ , $W/m^3$
$r, z_f$	coordinates used for fiber, m
$r_1$	fiber core radius, $\mu m$
$r_2$	inner cladding layer radius, $\mu m$
$r_3$	outer cladding layer radius, $\mu m$
$R_1$	reflectivity of the anterior cavity mirror
$R_2$	reflectivity of the posterior cavity mirror
$R_{1D}$	one dimensional thermal conductive resistance of plate, $K/W$ or $^\circ C /W$
$R_{hl}$	convective resistance for heat transfer coefficient $h_l$ , $K/W$ or $^\circ C /W$
$R_s$	spreading thermal resistance, $K/W$ or $^\circ C /W$
$R_{th}$	thermal resistance, $K/W$ or $^\circ C /W$
$R_{total}$	total thermal resistance of network model, $K/W$ or $^\circ C /W$
$t_1$	thickness of plate, m

$T(x, y, z)$	plate temperature, K or $^\circ C$
$T_{Exp}$	experimental temperature, K or $^\circ C$
$T_{fa}$	free flow temperature or ambient temperature, K or $^\circ C$
$\bar{T}_{Ana}$	average analytic temperature, K or $^\circ C$
$\bar{T}_{Simu}$	average simulation temperature, K or $^\circ C$
$\bar{T}_s$	average temperature of heat source, K or $^\circ C$
$T(r, z_f)$	temperature in fiber, K or $^\circ C$
$\bar{T}_w$	average temperature of the lower wall, K or $^\circ C$
$x, y, z$	coordinates used for plate, m
$xc, yc$	heat source centroid, m

**Greek symbols**

$\lambda_p$	pump wavelength, nm
$\lambda_s$	laser wavelength, nm
$\sigma_{ap}$	pump absorption cross-section, $m^2$
$\sigma_{as}$	signal absorption cross-section, $m^2$
$\sigma_{ep}$	pump emission cross-section, $m^2$
$\sigma_{es}$	signal emission cross-section, $m^2$
$\alpha_a$	the absorption of pump light, $m^{-1}$
$\alpha_p$	the loss of pump light, $m^{-1}$
$\Gamma_p$	pump power filling factor
$\Gamma_s$	laser power filling factor
$\beta$	eigenvalues, $\sqrt{\delta^2 + \lambda^2}$
$\delta$	eigenvalues, $n\pi/b$
$\lambda$	eigenvalues, $m\pi/a$
$\varphi$	spreading function
$\zeta$	dummy variable, $m^{-1}$
$\theta$	temperature excess, $\equiv T - T_{fa}$ , K or $^\circ C$
$\nu_p$	pump frequency
$\nu_s$	signal frequency
$\tau$	spontaneous lifetime
$\eta$	efficiency of absorption

**Subscripts**

$1D$	one-dimension
$Ana$	analytic
$Exp$	experimental
$f$	fiber
$fa$	ambient or cold source
$i, j$	indices for segment
$l$	lower surface
$s$	heat source
$Simu$	simulation

to the complexity of slender fiber itself, and such fiber thermal model is seldom available in the literature. Based on an example of 4.4–31.07 W ytterbium DDCF laser, this paper employs separation variable method to calculate spreading thermal resistance [17,18] of the eccentric heat sources on the rectangular channel and simulates the temperature field of the coupling of long fiber and substrate. This study focuses on the analysis of characteristics of optical fiber heat dissipation through the plate, and the results are verified experimentally.

**2. Method and model**

**2.1. Heat source function**

The simplified model of optical resonator sketch is shown in Fig. 1. Heat source of double clad fiber is generated due to the pump light absorbed by  $Yb^{3+}$  doped in the fiber core. The length of fiber is much larger than the diameter of radial cross-section. Therefore, the thermal

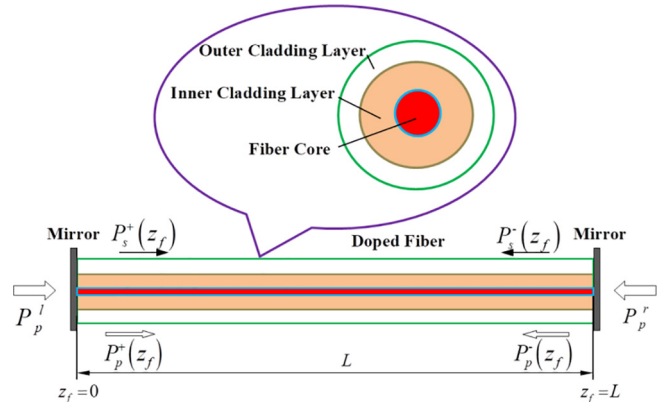


Fig. 1. Simplified model of optical resonator sketch.

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