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Energy saving and economic analysis of a new hybrid radiative cooling system for single-family houses in the USA

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HIGHLIGHTS

• Novel hybrid Radiative cooled-Cold storage cooling (RadiCold_{hc}) system is proposed.

• Energy and economic analysis are investigated on a highly insulated two-story house.

• RadiCold_{hc} system will save annual cooling electricity consumption by 26–46%.

• Maximum incremental costs are \$50.0/m²-\$78.9/m² for an 8-year simple payback period.

ARTICLE INFO

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ABSTRACT

Radiative cooling has received much attention as it generates "free" cooling to buildings and helps reduce energy consumption of mechanical air conditioning systems. However, most current radiative cooling materials either work for nighttime (nocturnal) cooling only or have high cost issues. A novel scalable-manufactured randomized glass-polymer hybrid metamaterial coated with silver has recently been developed and reported a 110 W/m^2 cooling power on daily average. This metamaterial potentially provides passive cooling for both nighttime and daytime. Proposed is a hybrid diurnal radiative cooled-cold storage cooling system using this metamaterial for air conditioning purposes in single-family houses. Because single-family houses have a relatively low cooling load but high ratio of roof area to floor area, they are excellent end users of the hybrid radiative cooled-cold storage cooling system. The potential energy savings of the hybrid radiative cooled-cold storage cooling system in a typical two-floor single-family house with floor area of 204 m^2 have been modeled using EnergyPlus for four locations in the U.S., including Orlando, FL, San Diego, CA, San Francisco, CA, and Denver, CO. In comparison with the electricity consumption of a split air conditioner alone, the hybrid radiative cooled-cold storage cooling system could save annual cooling electricity by 26% to 46% for the modeled locations, under a restriction of 8year payback period. The corresponding simple payback periods for adoption of the hybrid radiative cooled-cold storage cooling system fall in a range of 4.8-8.0 years and the maximum acceptable incremental costs are \$50.0/ m^2 -\$78.9/m². The diurnal working hybrid radiative cooled-cold storage cooling system may provide a costeffective solution for radiative cooling technology in residential building applications.

1. Introduction

As a passive cooling technology, radiative cooling surface transfers heat to sky by infrared radiation whenever the effective sky temperature is lower than the radiative surface temperature [1-4]. Although the radiative cooling technique has been developed for many years [5-9], the application has been limited to nighttime due to solar absorption of most materials in the daytime [10-14]. With the development of micro/ nanotechnologies, daytime radiative cooling has become possible, and applications of diurnal cooling to buildings could be realized [15,16]. In particular, Zhai et al. [17] has recently developed the radiative cooling film, of a randomized, glass-polymer hybrid metamaterial with silver coating, and characterized with effectively rejecting solar irradiance and strongly emitting in the infrared. This film can be scalable-manufactured at relatively low-cost. It provides high technical and economic potential for the development of diurnal radiative cooling

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Nomenclature (W/m ²)			(W/m ²)	
		$q_{ m sol}$	solar radiation absorbed by the radiative cooling surface a_{1}	
A _S	area of the radiative cooling surface (m ²)	P	(W/m^2)	
<i>AIC</i> _{max}	maximum acceptable incremental cost per unit area of $\frac{1}{2}$	R _f RSAC	roughness multiplier of radiative cooling surface	
ACDD	radiative cooling surface $(\$/m^2)$		reduced-size split air conditioner	
ASPP	Acceptable simple payback period (year)	SACL	split air conditioner lonely	
C_{Radi}	cost of the RadiCold system (\$)	SPP	simple payback period (year)	
$C_{\rm RSAC}$	cost of the reduced-size split air conditioner in RadiCold _{he}	t _{charging}	total charging time (s)	
~	system (\$)	T_{amb}	ambient air temperature (K)	
$C_{\rm RC}$	cooling capacity for a of RadiCold module (kW)	$T_{\rm d}$	ambient dewpoint temperature (K)	
C_{SACL}	cost of the split air conditioner lonely (\$)	$T_{\rm sky}$	effective sky temperature for longwave radiation (K)	
cp	specific heat capacity of water (J/kg·K)	$T_{\rm surf}$	temperature of radiative cooling surface (K)	
CL _{avg-hou}		$T_{\mathrm{w}, \tau}$	RadiCold module water temprature of current time step	
$S_{\rm RSAC}$	size of reduced-size split air conditioner (kW)	_	(K)	
$S_{\rm SACL}$	size of split air conditioner lonely (kW)	$T_{w, \tau} + 1$	RadiCold module water temperature of next time step (K)	
$E_{\rm Radi-hc}$	cooling electricity consumption of the RadiCold _{hc} system	T_{w}	average water temperature in RadiCold module (K)	
_	(kWh)	ΔT_{tank}	variation of the temperature in the tank (K)	
E_{SACL}	cooling electricity consumption of the split air conditioner	V	volume of water in the RadiCold module (m ³)	
_	lonely (kWh)	V _{tank}	volume of tank (m ³)	
$h_{ m f}$	forced convection component of radiative cooling surface	$V_{ m z}$	local wind speed (m/s)	
	$(W/m^2 \cdot K)$	$W_{ m f}$	wind direction modifier	
$h_{ m n}$	natural convection component of radiative cooling surface			
	$(W/m^2 \cdot K)$	Greek let	k letter	
$h_{ m RS}$	convective heat transfer coefficient over the radiative			
	cooling surface (W/m ² ·K)	α	surface tile angle	
I _{sol}	incident solar radiation (W/m ²)	$\varepsilon_{\rm sky}$	average spectral and directional longwave emissivity/ab-	
Ν	number of RadiCold modules employed in the residential		sorptivity of the atmospheric	
	building	$\varepsilon_{\rm surf}$	radiative cooling surface emittance	
Р	perimeter length of radiative cooling surface (m)	η	efficiency of discharging	
P_E	price of electricity (\$/kWh)	λ	absorption coefficient	
$q_{ m conv}$	convection heat transfer between radiative cooling surface	ρ_w	density of water (kg/m ³)	
	and ambient air (W/m^2)	ρ_{surf}	solar absorption rate of the radiative cooling surface	
$q_{ m RS}$	net radiative cooling power (W/m^2)	σ	Stefan-Boltzmann constant (J/(m ² ·s·K ⁴))	
$q_{ m rad}$	heat transfer between radiative cooling surface and sky	$\Delta \tau$	length of the time step (s)	

systems for building applications. Direct thermal measurement of the radiative cooling film demonstrated approximately 110 W/m^2 cooling power on a daily average in hot weather [17].

Various strategies have been proposed utilizing the radiative cooling technology in buildings, including passive systems directly

applying selective-emittance material for roofs, and active systems such as water based, air based, and hybrid systems (Fig. 1) [2,4].

The water based cooling systems can be categorized into two forms: open loop [18–22] and closed loop [23–26]. The open loop water system commonly consists of a shallow rooftop pond, where the heat is

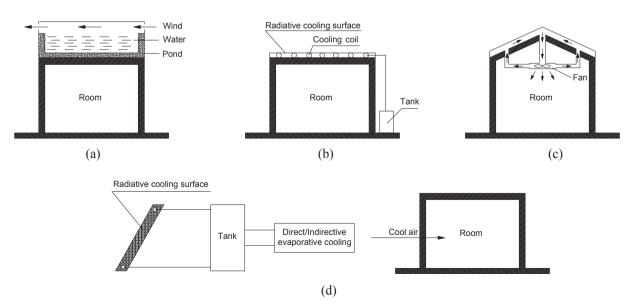


Fig. 1. Sketch of active radiative cooling systems: (a) water based system (open loop); (b) water based system (closed loop); (c) air based system; and (d) hybrid system.

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