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Thermal integration of trigeneration systems

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Abstract

Trigeneration can be considered as a special case of the application of cogeneration systems where a fraction of the shaft work or residual heat is used for running a refrigeration system. This work focuses on trigeneration schemes where a gas turbine is used as a prime mover for power production and cooling is generated by a typical compression-refrigeration system. In most applications, a gas turbine will meet either the process power requirements or the heating needs, but it is unlikely that both would be satisfied simultaneously in the most efficient manner. The selection of the gas turbine that minimizes the heat losses to the ambient while supplying the required power can be readily accomplished by superimposing the turbine exhaust gas temperature profile to the process streams profile in a T vs enthalpy curve. This is because the maximum overall efficiency depends on the process heat and power demands and on the shape of the heat demand profile of the process. The use of the thermodynamic model helps to simulate the main components of the system and permits a fast and interactive way to design the optimum trigeneration scheme using the performance data of commercial gas turbines. © 2004 Elsevier Ltd. All rights reserved.

Keywords: Trigeneration; Thermal integration; Pinch analysis; Gas turbines; Cogeneration

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Nomenclature

C	1
Cp	neat capacity at constant pressure
$C_{\rm v}$	heat capacity at constant volume
COP	coeficient of performance
F	fuel consumption
H	enthalpy
$H_{\rm w'1}$	adiabatic compressor outlet stream enthalpy
H.,	compressor inlet stream enthalpy
K	$C_{\rm m}/C_{\rm r}$
mC	heat capacity mass flow rate
mep me	refrigerant mass flow rate
D D	pressure
	hast loss to surroundings
Q_{amb}	heat loss to surroundings
$Q_{\rm C}$	condenser neat duty
$Q_{\rm cmin}$	minimum process cooling needs
$Q_{\rm E}$	evaporator heat duty
Q_{excess}	heat in excess from turbine
$Q_{ m hav}$	heat available in hot gases
$Q_{ m hmin}$	minimum process heat needs
$Q_{\rm SK}$	heat delivered to sink of heat engine
$Q_{\rm SR}$	heat available from source of heat engine
r _c	air compression ratio
$r_{\rm T}$	turbine inlet/outlet pressure ratio
Т	temperature
T_1	compressor air inlet temperature
T_3	turbine gases inlet temperature
T_4	turbine gases outlet temperature
T_{4s}	adiabatic turbine gases outlet temperature
$T_{\rm E}$	refrigerant evaporating temperature
T_{C}	refrigerant condensing temperature
T_{SK}	heat engine sink temperature
TSR	heat engine source temperature
Wad	adiabatic compression work
W _a	process power needs
Waar	power available for the process
W	power produced in excess
W.p	refrigeration power consumption
W _T	turbine power production
W.	exhaust gases mass flow rate
и И/	specific net power from turbine
ν _n	anthalmy ahanga or heat duty
ΔH	enthalpy change or heat duty

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