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# Transient analysis and energy optimization of solar heating and cooling systems in various configurations

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#### Abstract

In this paper, a transient simulation model of solar-assisted heating and cooling systems (SHC) is presented. A detailed case study is also discussed, in which three different configurations are considered. In all cases, the SHC system is based on the coupling of evacuated solar collectors with a single-stage LiBr-H<sub>2</sub>O absorption chiller, and a gas-fired boiler is also included for auxiliary heating, only during the winter season. In the first configuration, the cooling capacity of the absorption chiller and the solar collector area are designed on the basis of the maximum cooling load, and an electric chiller is used as the auxiliary cooling system. The second layout is similar to the first one, but, in this case, the absorption chiller and the solar collector area are sized in order to balance only a fraction of the maximum cooling load. Finally, in the third configuration, there is no electric chiller, and the auxiliary gas-fired boiler is also used in summer to feed the absorption chiller, in case of scarce solar irradiation.

The simulation model was developed using the TRNSYS software, and included the analysis of the dynamic behaviour of the building in which the SHC systems were supposed to be installed. The building was simulated using a single-lumped capacitance model. An economic model was also developed, in order to assess the operating and capital costs of the systems under analysis. Furthermore, a mixed heuristic-deterministic optimization algorithm was implemented, in order to determine the set of the synthesis/design variables that maximize the energy efficiency of each configuration under analysis.

The results of the case study were analyzed on monthly and weekly basis, paying special attention to the energy and monetary flows of the standard and optimized configurations. The results are encouraging as for the potential of energy saving. On the contrary, the SHC systems appear still far from the economic profitability: however, this is notoriously true for the great majority of renewable energy systems.

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Keywords: Solar energy; Evacuated collectors; Absorption chiller; Dynamic simulation; Renewable energy

#### 1. Introduction

Solar-assisted air conditioning is a very promising concept. Usually, the maximum demand for cooling coincides with the maximum availability of solar radiation, whereas conventional electric-driven systems have the problem of providing their minimum capacity in the hottest day hours.

In addition, the use of solar energy in refrigeration can be very useful in order to limit the growth of the electric energy demand in summer and for sustaining the development of technologies based on renewable energy sources.

Many institutions are presently involved in R&D activities in this field, and a lot of demonstration projects have been developed: for example, the International Energy Agency (IEA) launched a program ("Solar Heating and Cooling Programme, SHC", Task 25 "Solar-Assisted Air Conditioning of Buildings", initiated in June 1999 and completed in November 2004) aimed at improving condi-

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Nomenclature			
$c_{ m NG}$	unit cost of natural gas (€/Sm³)	$U_{\mathrm{L}/T}$	SC transmittance (second term, kJ/h m² K²)
$c_{\mathrm{EE}}$	unit cost of electric energy (€/kWh)	UA	overall transmittance (kJ/h K)
$c_p$	constant pressure specific heat (kJ/kg K)	V	volume (m <sup>3</sup> )
$c_v$	constant volume specific heat (kJ/kg K)	$\eta_{ m el}$	conventional efficiency in thermoelectric conver-
f	dimensionless design factor		sion
m	mass flow rate (kg/h)	$\eta_{ m motor}$	motor efficiency
v	ratio between tank volume and SC area (m³/m²	$\eta_{ m pumping}$	pump efficiency
	or $1/m^2$ )	$\eta_{ m SC}$	solar collector efficiency
t	temperature (°C)	$\eta_{ m AH}$	AH efficiency
A	area (m <sup>2</sup> )	$\eta_{ m comb}$	combustion efficiency
AF	annuity factor (years)	$(\tau \alpha)_n$	transmittance-absorptance at normal incidence
C	annual cost (€/year)		angle
COP	coefficient of performance	$\alpha_{\mathrm{SC}}$	SC slope (deg)
$E_{ m el}$	electric energy (kJ)	$\vartheta$	time (h)
$F_R$	heat removal factor	$\beta_i$	power coefficient
$F_{\rm sol}$	solar fraction	ho	density (kg/m <sup>3</sup> )
FFLP	fraction of full-load power	γ	control function
HDD	heating degree day (Kd)	$arphi_{ ext{P2}}$	mass flow rate per SC area (kg/h m <sup>2</sup> )
$I_T$	total radiation on SC surface (kJ/h m <sup>2</sup> )	$\Delta t_{ m n}$	nominal temperature difference (°C)
${J}_i$	capital cost of a single component (€)	$\Delta t_{\mathrm{TK1}}$	ACH re-activation set-temperature (°C)
${J}_{tot}$	capital cost of the overall plant (€)	$\Delta C$	operating cost savings
LHV	natural gas lower heating value (kWh/Sm <sup>3</sup> )		
$M_i$	mass of the TK1 <i>i</i> -th segment (kg)	Subscripts	
OF	objective function (€/year)	DL	design load
$\dot{P}$	mechanical power (kJ/h)	Н	TK1 top
P	nominal capacity (kJ/h)	L	TK1 bottom
PE	primary energy (kJ)	amb	ambient
PES	primary energy saving	c	cooling
PLR	part-load ratio	cap	capital
$R_{\rm COP}$	ratio of actual to nominal COP (electric chiller)	EE	electric energy
$R_Q$	ratio of actual to nominal cooling capacity (elec-	h	heating
	tric chiller)	in	inlet
$Q$ $\dot{Q}$ $\dot{Q}_{B,\mathrm{lsm}}$	thermal energy (kJ)	n	nominal
Q	thermal energy flow rate (kJ/h)	NG	natural gas
$Q_{B,\mathrm{lsm}}$	building maximum cooling load (kJ/h)	out	outlet
$Q_{\mathrm{fluid}}$	thermal energy flow rate transferred to the fluid	op	operating
	(kJ/h)	opt	optimal
$\dot{Q}_{ m ambient}$		rated	referred to nominal conditions
	ronment (kJ/h)	req	required
$\dot{Q}_{\mathrm{SC}}$	useful solar collector energy gain (kJ/h)	rej	rejected
SPB	simple Pay Back (years)	S	summer
$T_{ m off,ACH}$	ACH shut-down temperature (K)	set	set by the controller
U	transmittance (kJ/h m <sup>2</sup> K or W/m <sup>2</sup> K)	tot	total
$U_{ m L}$	SC transmittance (first term, kJ/h m <sup>2</sup> K)	$\mathbf{W}$	winter

tions for the market introduction of solar-assisted cooling systems. This task, and the subsequent ones (IEA, Task 38), promoted a reduction of primary energy consumption and electricity peak loads due to air conditioning and thereby developed an environmentally friendly way for air conditioning (IEA).

In this framework, a lot of experimental and theoretical research work has been done in the last few years. In par-

ticular, in the field of the analysis and optimization of solar cooling systems, many simulation codes for the design analysis and optimization were developed, usually with transient simulation environments, such as TRNSYS, ENERGY PLUS, etc. For example, Florides et al. developed a very interesting simulation model in TRNSYS for a Cypriot building (Folrides et al., 2001, 2002). Such model was based on the use of several built-in components (ther-

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