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## Modeling of driveway as a solar collector for improving efficiency of solar assisted geothermal heat pump system: a case study



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#### ABSTRACT

It is well known that rooftop solar thermal panels increase both power rates of circulation pumps and initial investment cost of solar assisted ground source (geothermal) systems. To avoid both of them it means that the unnecessary energy consumption rates of circulation pump(s) and their initial capital cost, rather than installing rooftop solar thermal panels, driveways can be used as solar collectors for improving efficiency of geothermal heat pump systems (GSHP) and declining initial capital cost of SAGSHPs. Mainly this idea was first put in the middle by Jefferson W. Tester. In this paper, we will examine modeling of driveway as solar thermal panel to enhance efficiency of solar assisted geothermal heat pump system (SAGSHP) depends on its different operating types; yet we will give only a case that is investigated theoretically for solar assisted geothermal heat pump systems.

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#### 1. Introduction

GSHP applications have been increased all over the world, since they have huge energy savings in air conditioning of buildings and these systems make it possible to evaluate both shallow

E-mail addresses: Onder.Ozgener@ege.edu.tr (O. Ozgener), Leyla.Ozgener@cbu.edu.tr (L. Ozgener). geothermal resources and shallow soils as energy resources. Moreover, they have indirect benefits on reduction in  $NO_x$ ,  $CO_2$ ,  $SO_2$ , etc emissions. These are well known environmental air pollutants.

Typical SAGSHP systems consist of solar thermal panels, heat pump coupled with horizontal or vertical ground heat exchangers. For SAGSHP applications in heating and cooling of buildings, the containment of solar thermal panels becomes an integral part of the projects, yet they can be problematic, since they require storage tanks and special attention to the design of

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Nomen	clature	$U_L$	overall loss coefficient of the collector (W/m²K)		
		U	velocity (m/s)		
$A_c$	collector surface area (m <sup>2</sup> )	V	volumetric flow rate (m <sup>3</sup> /s)		
$A_0$	amplitude of the daily mean air temperatures (°C)	T	temperature (°C)		
$A_z$	amplitude of the temperature wave at depth $z$ (°C)	$T_m$	annual mean air temperature (°C)		
$C_{p,wa}$	specific heat of the water-antifreeze solution (kJ/kgK)	$T_s$	surface temperature (°C)		
$COP_{HP}$	heating coefficient of performance of heat pump	$T_{z,t}$	soil temperature at depth z, at time t (°C)		
	(dimensionless)	t	time, a day in year expressed in hours (h)		
$COP_{sys}$	heating coefficient of performance of the overall	$t_0$	time, zero point (h)		
	system (dimensionless)	$\dot{W}_{comp}$	power input to the compressor (kW)		
D	pipe diameter (m)	$W_{pump}$	total power input to the brine and water circulating		
Εx	exergy rate (kW)	TÅZ	pumps (kW)		
$E\dot{x}_{dest}$	exergy destruction rate of heat exchanger (kW)	$\dot{W}_{fc}$	power input to the fans of the fan-coil unit (kW)		
Ex <sub>dest,grh</sub>		X	depth below the ground surface (m)		
	exchanger (kW)	Z	soil depth (cm, m)		
	mp exergy destruction rate of pump (kW)	6 11			
Ex <sub>dest,col</sub>	exergy destruction rate of collector (kW)	Greek le	Greek letters		
	exergy destruction rate of expansion valve (kW)		200		
f	friction losses (dimensionless)	$\alpha$	thermal diffusivity (cm <sup>2</sup> /h), absorptance		
$F_R$	collector heat removal factor (dimensionless) specific enthalpy (k]/kg)		(dimensionless)		
h	1 10 ( 0)	$\eta$	energy efficiency (dimensionless)		
Н	monthly average daily radiation on a horizontal surface (MJ/m <sup>2</sup> )	$\eta_c$	collector energy efficiency (dimensionless)		
и	monthly average daily extraterrestrial radiation (MJ/	$\beta$	tilt angle of drive way (collector) from the horizontal		
$H_0$	$m^2$ )		(dimensionless)		
I.	direct solar radiation arriving on a horizontal surface	γ	Inverse of damping depth (cm <sup>-1</sup> )		
$I_b$	$(MJ/m^2)$	δ	declination angle (dimensionless)		
1.	instantaneous diffuse solar radiation on a horizontal	ho	reflectance (dimensionless); density (kg/m³)		
$I_d$	surface (MJ/m <sup>2</sup> )	τ	transmittance (dimensionless)		
I	instantaneous total radiation on a horizontal surface	$\omega_{_{\mathcal{F}}}$	hour angle at sunrise (dimensionless) the particular resistance losses (kPa, Pa)		
•	(MJ/m <sup>2</sup> )	ζ	latitude angle (dimensionless)		
$I_T$	rate of incidence of total radiation on a unit area of	$\phi \ \psi$	specific exergy (kJ/kg)		
1	surface (MJ/m²)	Ψ	specific exergy (kg/kg)		
$K_T$	monthly average clearness index (dimensionless)	Abbrevi	iations		
k	thermal conductivity (W/m K)	Abbieviations			
L	length, ground heat exchanger layout zone (m)	AC	asphalt cover thickens		
ṁ	mass flow rate of refrigerant (kg/s)	BH	borehole depth		
$\dot{m}_{wa}$	mass flow rate of water (kg/s)	BZT	Bethe–Zeldovich–Thompson		
P	period (h),	EWT	entering water temperature		
$\Delta P$	Pressure, pressure loss (kPa)	FC	fan coil		
$R_{soil}$	conduction resistance of soil per unit length (Km/W)	GSHP	geothermal (ground source) heat pump		
R	ratio of the total radiation on a tilted surface to that on	HP	heat pump		
_	a horizontal surface	HE	heat exchanger		
$R_b$	ratio of the instantaneous direct solar radiation on a	VGHE	vertical ground heat exchanger		
	tilted surface to the instantaneous direct solar radia-	HGHE	horizontal ground heat exchanger		
_	tion on a horizontal surface	in	inlet		
S	entropy (kJ/kg-K)	out	outlet		
q	heat flux of density (W/m <sup>2</sup> )	NIST	The National Institute of Standards and Technology		
$\dot{Q}_U$	rate of useful heat transfer to water in the solar		P solar assisted geothermal heat pump		
Ġ	collector (kW)		PS solar assisted geothermal (ground-source) heat		
$Q_{cond}$	heat rejection rate in the condenser (kW)		pump system		
Q <sub>e</sub>	ground heat exchanger or extracted heat load (kW) heat transfer rate in the evaporator (kW)	SGR	shallow geothermal resources		
Сеvар	ambient temperature (K, °C)	Sys	system		
$T_a$ $T_i$	fluid inlet temperature, greenhouse inside design	TES	thermal energy storage		
1 į	temperature (K, $^{\circ}$ C )	x,y,z	directions		
$T_o$	fluid outlet temperature, greenhouse outside design	1,2,	successive numbers		
10	temperature (K, °C)	0	reference state		
	temperature (it, C)				

ground and solar heat exchangers for charging and discharging, other considerations include possible water leakages bad effects to building roof, cost, short and long term cycling stability, space requirements, sometimes solar thermal panels and their storage

tanks can be exposed wind that occurs the heat losses etc. Hence, shallow soils could be an alternative solution instead of solar thermal panels. In this paper, we will examine modeling of driveway as solar thermal panel to enhance efficiency of solar

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