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# Parameter sensitivity analysis and configuration optimization of indirect evaporative cooler (IEC) considering condensation



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# HIGHLIGHTS

- Numerical model of IEC considering condensation were established and validated.
- Sensitivity analysis was conducted among seven parameters.
- Influence rank of seven parameters were obtained by orthogonal test.
- The channel gap and NTU of IEC were optimized.

#### ARTICLE INFO

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# ABSTRACT

The indirect evaporative cooler (IEC) is a low-carbon device which cools the air with water evaporation. Its performances in dry regions have been investigated intensively. However, its application in hot and humid regions, where the IEC is used as a pre-cooling device in an air-conditioning system, is still at developing stage. The exhausted air from air-conditioned space is humidified and used as secondary air to pre-cool the fresh air. As the dew point temperature of the fresh air is high, condensation may occur in the dry channels. However, the parameter sensitivity analysis of IEC with condensation is lacking. Besides, the optimized IEC configuration may be different from that of dry regions because of distinguished air handling process. So the sensitivity analysis among seven parameters by orthogonal test was conducted based on the experimental-validated IEC model emphasizing on condensation condition. The optimization was then conducted to the most influential and engineering controllable parameters. The results indicate the channel gap and cooler height are the key influential factors on IEC thermal performance. The optimized channel gap is 2–3 mm and 3–4 mm under condensation and non-condensation state, respectively. The optimized NTU<sub>n</sub> is 4–7 and 3–5, respectively.

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# 1. Introduction

Indirect evaporative cooler (IEC) is an energy efficient and environmental friendly cooling device which uses water evaporation to produce cooling air [1,2]. The IEC has been regarded as a promising energy-saving technology in the near future for its high efficient, low energy consumption, pollution-free and easy maintenance features [3–5]. The most commonly used plate-type IEC consists of alternative wet and dry channels which are separated by thin plates. In the wet channels, the spraying water drops form a thin water film on the plate surface and consistently evaporates into the main stream of secondary air. The primary air in the adjacent dry channels is sensibly cooled by the low temperature wall without change in moisture content [6]. The schematic diagram of a typical plate type counter flow IEC is shown in Fig. 1. The channel gap is an important geometric parameter which refers to the gap between adjacent plates in IEC.

The air with lower humidity provides larger evaporation driving force, so larger cooling capacity can be achieved when IEC applied in hot and dry regions. The cooled primary air is supplied to the interior directly in these regions [7–9]. But in hot and humid regions, its application is restricted because the supplied primary air temperature is limited to the high wet-bulb temperature of ambient air. However, in recent years, a new hybrid air-conditioning system consisted of IEC and mechanical cooling is proposed for IEC application in hot and humid regions [10–15]. The IEC, installed before an AHU or cooling coil, is used to precool the fresh air for energy conservation. The exhausted air with







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# Nomenclature

m²

- H cooler height, m
- *L* cooler length, m *P* pressure, pa
- Pr Prandtl number
- NTU number of heat transfer unit
- $c_{pa}$  specific heat of air, J/kg °C
- $c_{pw}$  specific heat of water, J/kg °C
- $d_e$  hydraulic diameter of channel, m
- h heat transfer coefficient, W/m<sup>2</sup> °C
- $h_m$  mass transfer coefficient, kg/m<sup>2</sup> s
- $h_{fg}$  latent heat of vaporization of water, J/kg
- $i^{JS}$  enthalpy of air, J/kg °C
- m mass flow rate, kg/s
- *n* number of channels
- s channel gap, m
- *t* Celsius temperature, °C
- *u* air velocity, m/s

## Greek symbols

- $\omega$  moisture content of air, kg/kg
- $\sigma$  wettability
- η efficiency
- ε enlargement coefficient

- $\mu$  dynamic viscosity, Pa s
- v kinematic viscosity, m<sup>2</sup>/s
- $\lambda$  thermal conductivity, W/m °C

#### Subscripts

- c condensation
- *e* evaporation
- p primary/fresh air
- s secondary/exhaust air
- w wall
- *cw* condensate water
- ew evaporation water
- in air inlet
- out air outlet
- wb wet-bulb
- *lat* latent heat transfer

#### Abbreviation

- RH relative humidity
- IEC indirect evaporative cooler
- RIEC regenerative indirect evaporative cooler
- COP coefficient of performance



Fig. 1. Schematic diagram of plate type counter flow IEC.

lower temperature and humidity from air-conditioned space is used as secondary air for enhancing the evaporation process. Unlike the normal cases in which the IEC operates for only sensible cooling in dry regions, condensation from high humidity primary air may occurs when IEC applied in humid regions, resulting in not only sensible cooling but also dehumidification effect. However, it is found that relevant study on IEC with condensation is still at developing stage [16]. For high efficient cooling system, the parameter study, sensitivity analysis and configuration optimization are crucial because they could provide valuable guidance for favorable operating conditions and system design. A lot of such studies have been conducted to IEC under its non-condensation state. The models of IEC including traditional IEC, RIEC, M-cycle IEC (a dew-point IEC proposed by Maisotsenko et al. [17]) and other dew-point IEC [18] were established and solved numerically or analytically to Download English Version:

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