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The effect of the short wave radiation and its reflected components on the mean radiant temperature: modelling and preliminary experimental results



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ABSTRACT

In outdoor as well as indoor environments, human thermal sensation strongly depends on the direct component of solar radiation incident on the body. Nevertheless, even though the direct component exerts the major contribute on this issue, especially in indoor environments and confined spaces, the diffuse and reflected components of the solar radiation also affects the thermal sensations of people. Despite this evidence, simple and reliable methods designed to take into account the effect of solar radiation on the indoor radiant field enveloping the human being in indoor environments are hardly available.

This article aims to provide a contribute on this topic, proposing a model for the computation of the mean radiant temperature (MRT) in indoor environments in presence of solar radiation. The most innovative facet of the proposed model regards the computation of the effects of the radiation components reflected by the internal surfaces.

Moreover, in order to try a preliminary validation of the model, an experimental campaign was also carried out and MRT values were measured in positions either directly irradiated by the sun or shielded from direct irradiation. The purpose of the measurements was to preliminarily analyze the extent of the accuracy with which the model might predict the rise of MRT due to solar direct irradiation.

1. Introduction

In both outdoor and indoor environments, human thermal comfort is influenced by various factors (climate conditions, physiological and subjective issues, etc.), which have combined and diversified effects on human energy balance. Comfort and energy demand in buildings are strongly interconnected and the achievement of high quality standards involves an energy cost, which must be taken into account [1-3]. The maintenance of the required environmental quality levels, as a matter of fact, implies that a series of environmental variables should be controlled within recommended values [4]; nevertheless, this approach could not suffice in many actual cases, when comfort indexes [5], parameters and variables should be accurately evaluated [6,7].

Among all these factors, radiative heat exchanges play a pivotal role: in confined environments they contributes as far as 30% of the whole thermal exchanges involving the subject [8], but in case of direct solar radiation they can become the most significant cause of heat gain and discomfort [9].

The mean radiant temperature, MRT, is one of the main factor used to quantify the effect of the radiant field on human thermal response [10]. This quantity plays a crucial role in both outdoor and indoor situations and several studies have stressed the influence of this variable on thermal comfort in urban settings [11], researching and comparing reliable and feasible methods for its assessment [12–14] or its measurement [15–18]. All these studies, moreover, consider solar radiation as a key factor in the assessment of the mean radiant temperature on the grounds that thermal comfort is highly dependent on both long wave and short wave radiation fluxes from the surround-ings.

In indoor environments, the influence of the mean radiant temperature on thermal comfort is also well documented [19-23]. Moreover, in this case, MRT also influences the energy consumption to a certain extent. The issue has been investigated, analyzing the energy saving potential in a PMV-controlled space [24]. The results suggested that energy consumption, in a thermal comfort-controlled space, is strongly affected by a change in the mean radiant temperature and that the thermal comfort control can be considered as a reasonable strategy for both thermal comfort and energy saving purposes which, in turn, are to become one of the main objective of the building industry all over the world [25–28].

On balance, the accurate assessment of radiant field and MRT is the key to achieving both comfort optimization and energy efficiency.

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Nomenclature		$Q_{S,abs}$	absorbed share of the thermal flow that reaches the subject [W]
A_i area of the <i>i</i> th surface of t	the envelope [m ²]	T_a, t_a	air temperature [K, °C]
A_E whole area of the surface	of the envelope $[m^2]$	T_{cl}	mean temperature of clothed surface [K]
A_n projected area of the subj	ject onto a plain normal to the	T_a, t_a	globe-thermometer temperature [K, °C]
direction of the solar bear	$n [m^2]$	Tad. tad	globe temperature, in not directly irradiated position [K.
$A_{\rm s}$ effective area of the hum	an body exposed to radiation	gu / gu	°C]
[m ²]		Tab. tab	globe temperature, in irradiated position [K, °C]
D globe-thermometer diame	ter [m]	T_i	temperature of the <i>i</i> th surface of the envelope [K]
$\Lambda \overline{T}_{rb}$ contribution to the mean	radiant temperature due to the	$\frac{\overline{T}}{\overline{T}}$, \overline{t}	mean radiant temperature [K. °C]
direct component of the s	olar radiation [K]	$\overline{T}_{rh} C, \overline{t}_{rh} C$	calculated mean radiant temperature, in the presence of
f projected area factor	[]	-10,0, 10,0	the direct component of the solar radiation [K. °C]
$F_{i \rightarrow i}$ angle factor between the	<i>i</i> th glazed surface and the <i>i</i> th	Trh M. Trh M	measured mean radiant temperature, considering the
surface of the envelope		10,117 -10,11	effect of the direct component of the solar radiation [K.
$F_{S \rightarrow i}$ angle factor between the si	ubiect and the <i>i</i> th glazed surface		°C]
of the envelope	j	$\overline{T}_{r,C}, \overline{t}_{r,C}$	calculated mean radiant temperature [K, °C]
h_{ca} convective heat transfer co	oefficient [W m ⁻² K ⁻¹]	\overline{T}_{rd}	mean radiant temperature, without considering the direct
I_{h} direct solar radiation that	strikes the subject [W m ⁻²]		component of the solar radiation [K]
I_{bh} direct solar radiation on t	he horizontal plane [W m ⁻²]	$\overline{T}_{rd,M}$	measured mean radiant temperature, without considering
I_{di} diffuse sky radiation ente	ering the room through the <i>j</i> th		the effect of direct component of the solar radiation [K]
glazed surface [W m ⁻²]	0 0 0	$\overline{T_{r,M}}$	measured mean radiant temperature [K]
I_h global solar radiation on t	he horizontal plane [W m ⁻²]	va	air velocity [m s ⁻¹]
Q_{0S} emitted flow [W]	•		
$Q_{A \to S}$ radiative flux emitted by t	he surfaces of the environment	Greek sy	mbols
[W]		-	
$Q_{B \to S}$ short wave solar radiant flu	ux on the body surface, entering	α	solar altitude [°]
the room through the glaze	ed surface and due to the direct	α_{LW}	long wave absorbance of the human body
radiation [W]		α_{SW}	short wave absorbance of the human body
$Q_{D \to S}$ short wave solar radiant flu	ux on the body surface, entering	ϵ_{g}	emissivity of the globe-thermometer
the room through the glaze	ed surface and due to the diffuse	ϵ_s	emissivity of the human body
sky radiation [W]		ϵ_E	emissivity of the surfaces of the environment
$Q_{E\leftrightarrow S}$ flux exchanged for radiation	on among the human body and	$ ho_i$	reflectance of the <i>i</i> th surface of the envelope
the surfaces of the enviror	nment [W]	$ ho_{floor}$	reflectance of the pavement
Q_S net flux leaving the human	n body [W]	σ	Stefan–Boltzmann constant (5,67×10 ⁻⁸ W m ⁻² K ⁻⁴)

However, also in indoor environments, both radiant field and MRT are strongly affected by short wave and long wave radiation fluxes and several experimental analysis have demonstrated that solar radiation is a significant cause of discomfort to people [29].

Despite this evidence, there are few models [8,30,31] which allow analytical assessment of MRT taking into account solar radiation.

This article aims to provide a contribute on this topic, proposing a model for the computation of mean radiant temperature values, in thermal moderate indoor environments, in the presence of solar radiation. The most innovative facet of the proposed model regards the computation of the effects of the radiation components reflected by the indoor surfaces. Indeed, considering the various surfaces which, by and large, the building envelope is composed of, these components might have a considerable influence on MRT.

In order to further investigate on this subject, an experimental campaign was also carried out using the globe-thermometer method and MRT values were measured in positions directly irradiated by the sun or shielded from direct irradiation.

2. The calculation of the mean radiant temperature of a subject exposed to solar radiation: the proposed model

The proposed model is aimed at the calculation of the mean radiant temperature (\overline{T}_r) inside an indoor environment when the human subject is exposed to direct, diffuse and reflected solar radiation.

It considers the human body totally surrounded by an enclosed environment, so that, in this condition, if only radiative thermal exchange are taken into account, the net flux leaving the human body (Q_S) is equal to the flux exchanged for radiation among the human body and the surfaces of the environment $(Q_{E\leftrightarrow S})$ which the subject is in. That is [8,32]:

	(1)	۱
$\Sigma h \leftrightarrow h = \Sigma h$	(<u> </u>	

According to the MRT definition (which is the uniform temperature of an enclosure where the radiative flux on the subject, is the same as in the actual environment [33]), it is now assumed that the environment is an enclosure with an uniform temperature \overline{T}_r ; it is also assumed that the temperature of the human body is equal to the mean temperature of its clothed surface, T_{cl} . In this case, the thermal radiative flow exchanged between a subject and the surrounding surfaces of the enclosed environment is given by [8,32]:



Fig. 1. Radiative exchanges between the environment and the human body.

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