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# Position based simulation of solids with accurate contact handling

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## ABSTRACT

Simulating multi-body dynamics with both rigid and flexible parts and with frictional contacts is a hard problem. We solve this by expressing the couplings between the bodies as position level constraints. The implicit treatment of the constraint directions gives us improved stability over velocity based methods. Then by employing regularization of nonlinear constraints and a convex minimization formulation, we bridge constraint-based methods to traditional force-based methods. In fact, the former are just a dual variables formulation of the latter. We solve this dual problem using position based dynamics (PBD). We show how PBD is a completely valid modeling technique and we extend it with an accurate contact and Coulomb friction model. We further show for the first time how the same solver can be used to simulate both rigid and deformable solids with two way coupling. For the soft bodies we introduce a novel form of linear finite elements expressed as constraints, that is more accurate than PBD mass-spring systems. More of our results include the energy conserving Newmark integrator and the accelerated Jacobi solver suitable for parallel architectures. Note that this paper is an extended and revised version of the conference paper published in [1].

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#### 1 1. Introduction

For the last decade position based dynamics (PBD) has been 2 successfully applied to the simulation of particle systems and de-3 4 formable bodies [2]. This was possible given the inherent stability of the method due to its full implicit formulation: not only the 5 magnitude of the constraint forces are considered implicit, but also 6 7 their directions [3]. This is especially true for materials with fast 8 changing constraint gradients and transverse oscillations, e.g. cloth 9 or threads [4].

10 At its heart, PBD is a nonlinear constraint projection scheme, similar to the ones used in molecular dynamics [5]. The main 11 drawback of PBD is that it has no rigorous mathematical model for 12 contact and friction and thus it is almost never used for rigid body 13 14 simulations (with the exception of [6]). In our literature research we have not found any clear proof for the convergence of a PBD-15 like method with unilateral constraints and friction. Because of 16 this, some authors choose to treat contacts as bilateral constraints 17 [3] or approximate friction at the end of the step [7,8] without giv-18 19 ing a sound recipe for mixing friction with the position corrections. This paper reiterates our existing work in [1] and brings some ex-20 tensions to it. 21

We offer not only an accurate treatment of contact and fric-22 23 tion, but we also simulate deformable bodies in a physically cor-

https://doi.org/10.1016/j.cag.2017.09.004 0097-8493/© 2017 Elsevier Ltd. All rights reserved. rect manner using the theory of continua and the finite element 24 method (FEM). By employing the constraint regularization tech-25 nique [9] at position level, we are able to show that our constraint 26 solving problem is just the dual variables formulation of an equiv-27 alent elasticity problem. In the end, we are able to unify the simulation of rigid and deformable solids under the umbrella of PBD, using constraints as building blocks.

## 1.1. Related work

There has been a wealth of work published on the subject of 32 rigid body simulation with contact and friction - for a survey see 33 [10]. We note the advances made in the 90s by Baraff, Stewart 34 and Anitescu. Given the drawbacks of penalty forces, Baraff in-35 troduced the acceleration based linear complementarity problem 36 (LCP) method. This method had its problems too (related to im-37 pacts and the Painlevé paradox) that were later solved by a veloc-38 ity based approach that allows discontinuities in the velocities, i.e. 39 impulses. The new velocity time stepping (VTS) schemes [11,12] be-40 came very popular in computer graphics, games and real time sim-41 ulators. We take a similar approach in this paper, but based on 42 more recent work geared towards convex optimization [13-15], 43 although expressing the problem as such an optimization is not 44 mandatory. 45

Traditionally in computer graphics deformable bodies have 46 been simulated using implicit integrators due to their uncondi-47 tional stability properties. These have been applied not only to 48

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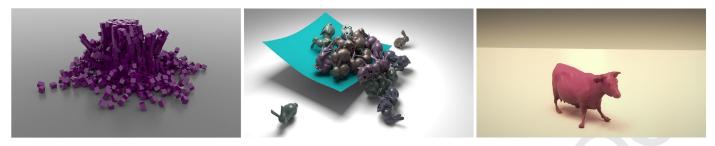


Fig. 1. From left to right: rigid boxes falling on ground, bunnies falling on a piece of cloth, a flexible cow falling on ground.

mass-spring systems, but also to simulations using finite differences or the finite element method (FEM) [16]. Recently, the popular Backward Euler integrator has been recast as an optimization
problem [17] helping us to gain new insights.

53 While initially constraint based methods were not considered for simulating deformable bodies, this changed with the advent of 54 55 PBD [7] and constraint regularization [9]. PBD was originally introduced by Jakobsen for games based on molecular dynamics meth-56 ods and a nonlinear version of the Stewart-Trinkle solver for rigid 57 bodies [8]. Goldenthal later showed how PBD stems from the fully 58 59 implicit integration of a constrained system [3]. Even though in theory constraints do not allow deformation for all the degrees of 60 freedom (often resulting in locking [18]), in practice, they proved 61 quite successful for simulating a wide range of objects (e.g. cloth, 62 63 hair, soft bodies - see 2 for a survey). This is due to the fact that 64 iterative solvers are often not run to convergence and this makes the constraints soft. 65

The idea of a unified solver is not new and our simulator bears maybe most similarity to Autodesk Maya's Nucleus. Our results are also along the line of more recent PBD work [6,19–21] and Projective Dynamics [17].

A great job of emphasizing the role of nonlinearity for achiev-70 ing stability was done in [22] - or rather the importance of using 71 a full implicit integration of nonlinear forces. Keeping the implicit 72 formulation intact is also the idea in [4] and the fact that dissi-73 pation (even if artificial) is key to the stability of the system is 74 75 stressed in [23]. We pursue a similar approach, but rely mostly on 76 updating the constraint directions at every iteration, without altering the mass matrix. 77

## 78 1.2. Contributions

We aim in this paper to show that PBD is a physically sound 79 method. This is done in Section 2 where we introduce a fixed point 80 81 iteration and prove it converges in Appendix A. Also, PBD can be used for both rigid and deformable bodies with constraints, contact 82 and friction in a single unified solver. The advantages of this for-83 mulation include better constraint satisfaction, improved stability 84 85 and out of the box two way coupling of rigid and elastic materials. 86 In addition to [1], we explain more in depth why the method is so robust and how it can be made more conservative while maintain-87 ing its stability properties. Another new sub-section explains how 88 our method is not bound to a minimization formulation and can 89 90 also be recast as a fixed point iteration of a box LCP solver.

Our new viscoelastic model permits us to incorporate soft 91 constraints, damping and FEM into PBD - for applications see 92 Section 3. Another goal we had in mind was to keep the computa-93 tional overhead to a minimum compared to existing methods. This 94 is why we chose our mathematical formulation to be expressible 95 as a matrix-free solver. We present a novel projected gradient de-96 scent algorithm for nonlinear optimization in Section 4. The algo-97 rithm is based on both the Jacobi and the Nesterov methods so 98 it can be parallelized. In Section 5 we continue to give some more 99

details on how to implement this solver (or a Gauss–Seidel one) for specific examples like the frictional contact constraint or the FEM tetrahedron constraint. In the end we give some code implementation notes and take a closer look at our results. As an extension to [1], we added some extra figures, plots and comments proving the nice stability and energy conservation properties of our solver.

## 2. Mathematical model 106

## 2.1. Equations of motion

In this section we present the continuous equations of motion 108 and a way to discretize them that will be the basis of our further 109 developments. We start with the equations of motion for a gen-110 eral system of bodies and, at first, we also introduce bilateral con-111 straints between the bodies: general nonlinear functions equated 112 to zero, describing for example a bead on a wire or joints articulat-113 ing rigid bodies. The resulting equations can also be derived from 114 Hamilton's principle and the principle of virtual work by using a 115 Lagrangian augmented by a special constraint potential:  $-\gamma^T \Psi(\mathbf{q})$ 116 [11]. They form a special type of *differential algebraic equations* 117 (DAE) [24] 118

 $\mathbf{M}\dot{\mathbf{v}} = \mathbf{f}_{tot} + \nabla \Psi(\mathbf{q})\boldsymbol{\gamma},\tag{1}$ 

$$\dot{\mathbf{q}} = \zeta \left( \mathbf{q} \right) \mathbf{v}, \tag{2}$$

$$\Psi(\mathbf{q}) = \mathbf{0},\tag{3}$$

where  $\mathbf{v} \in \mathbb{R}^n$  is the generalized velocity vector, *n* is the number of 121 degrees of freedom of the system,  $\mathbf{q} \in \mathbb{R}^{n'}$  is the generalized posi-122 tion vector ( $n' \ge n$  is the optimal number of parameters describing 123 position and orientation),  $\zeta$  is a linear kinematic mapping between 124 velocities and position derivatives, **M** is the mass matrix [10],  $\Psi(\mathbf{q})$ 125 is a vector-valued bilateral constraint function,  $\nabla \Psi(\mathbf{q})$  is its gradi-126 ent (i.e. the constraint directions),  $\boldsymbol{\gamma} \in \mathbb{R}^m$  is a Lagrange multipliers 127 vector enforcing the bilateral constraints in (3) (*m* is the number 128 of constraints), and  $\mathbf{f}_{tot}$  is the total generalized force acting on the 129 system (external and Coriolis). 130

In order to discretize the equations of motion we use the Implicit Euler (IE) integrator 131

$$\mathbf{M}(\mathbf{v}^{l+1} - \mathbf{v}^{l}) = h\nabla\Psi(\mathbf{q}^{l+1})\boldsymbol{\gamma}^{l+1} + h\mathbf{f}_{tot}^{l},$$
(4)

$$\mathbf{q}^{l+1} = \mathbf{q}^l + h\mathbf{L}\mathbf{v}^{l+1},\tag{5}$$

$$\Psi(\mathbf{q}^{l+1}) = \mathbf{0},\tag{6}$$

where *l* is the current simulation frame, *h* is the time step (considered constant), and  $\mathbf{L}(\mathbf{q}^l)$  is a linear kinematic mapping [10] with 136  $\mathbf{L}^T \mathbf{L} = \mathbf{1}$  (the identity matrix). The IE discretized equations can be brought to a minimization form 138

$$\mathbf{v}^{l+1} = \underset{\Psi(\mathbf{q}^{l}+h\mathbf{L}\mathbf{v})=\mathbf{0}}{\arg\min} \frac{\frac{1}{2}\mathbf{v}^{T}\mathbf{M}\mathbf{v} - \hat{\mathbf{f}}^{T}\mathbf{v},\tag{7}$$

where  $\hat{\mathbf{f}} = \mathbf{M}\mathbf{v}^l + h\mathbf{f}_{tot}^l$  and the new positions come from (5).

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