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Modelling and Simulation of Drag Forces of Non-spherical Particles Moving Towards a Surface in Polymer Melt Flows



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ABSTRACT

This work deals with the drag forces acting on non-spherical particles translating normal to a surface in a polymer melt in which the particle–surface distance is small compared with particle dimension in the flow direction. Hence, drag forces will be modified as a function of distance. A finite element model was used in order to calculate the drag forces for the motion of particles. Several non-spherical particles geometries were simulated towards a plane surface wall.

The fitting of the numerical results with empirical expressions presented two very well defined tendencies. Up to a given value of particle-wall distance the influence of the wall was strong, hence it was necessary to apply a correction to the analytical calculations. In addition, the relevance of the shape of the channel between the particle and the sink was demonstrated. The computational results provided a clear benefit to validate empirical approaches using analytical approximations.

1. Introduction

The study of particle motion and the hydrodynamic interactions between individual particles is a logical first step oncoming to understand the hydrodynamic behavior of suspensions and especially determining process design calculations in several industrial processing applications. The hydrodynamics control the distribution of particles hence determine the dynamical behavior of the particles. The drag force effect is of particular applicability in the study of colloidal dispersions and colloid–polymer mixtures [1]. As an example, in sedimentation the major problem comes from the lack of spatial location control of the particles within the bed of particles or near fixed walls. Sedimentation velocity depends on the drag coefficient (i.e. the drag force) which is modified by the proximity of the particles to the wall. Hence, affecting sedimentation velocity. Other typical examples include filtration or phase-change material processing when part of the viscous liquid material fluid flows around a particle.

Most of the studies of drag forces in the Stokes regime (Re < 1) mainly address empirical correlations for spheres and usually are formulated for free falling velocity. For example, Zhong et al. presented the state of art in numerical modelling and theoretical development for particle-fluid flows, particularly focusing on non-spherical particles [2]. In that review, models and techniques describing the forces between

particles and fluid in the case of spherical and non-spherical particles were highlighted, but in none of these studies the movement of the particle towards a wall was taken into account. In addition, there are other similar articles where the drag coefficients for non-spherical particles were determined, but in all of the studies the drag coefficients were calculated for free fall conditions [5–8].

In the present work, on the other hand, the drag force on a particle immersed into a polymeric matrix flowing near a wall or surface taken as a sink was investigated. Experimental results focusing on this topic were previously reported by Ambari et al., but only for spherical particles [9]. These results were compared here with our results obtained from simulation. However, our model not only simulates spherical particles but also addresses non-spherical particles.

The dynamics of non-spherical particles is considerably more complex than that of spherical particles, particularly, the particle-fluid interaction force. The movement of the particle is not parallel to the wall, hence, the drag force changes as a function of the surface distance in the flow direction [10-11].

The drag force was calculated here considering the influence of particle shape, the distance h to the surface and the fluid rheological properties.

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Fig. 1. Isometric perspective views of the particles used in the model. In all cases, the interface approaches the particle from the left in the direction of its axis of symmetry.

2. Methods: Model Description

A Computational Fluid Dynamics (CFD) model was implemented to calculate the drag force onto a particle immersed into a polymeric matrix near a wall taken as a sink. The results were compared with those obtained in the literature from analytical expressions and also with experimental results for the case of spherical particles [9,12–13].

The physical system used in the calculations was a solid particle with different shapes and a flat surface at different distances from the particle. In order to evaluate the influence of the wall on the drag force at least 6 distances h were modeled.

Spherical, cylindrical, conical, prolate spheroid and hemispherical particles were assayed. These geometries were chosen as the model was made in an axisymmetric domain which limits the types of non-spherical geometries that can be used, and in order to take into account typical fillers in polymeric materials such as short fibers, different carbon structures (carbon black, carbon nanotubes, graphite), and magnetic particles (barium hexaferrite) [11,14].

Isometric perspective views of the particles are shown in Fig. 1. For particles geometries containing flat faces, one of these faces was oriented parallel to the surface wall in order to consider extreme situations.

To compare simulation results for the "non-spherical particles" with those for the sphere, the aspect of the "non-spherical particles" was designed to be equivalent, in one dimension, to the sphere radius *R*. All the conditions used are summarized in Table 1. In the case of the cylinder, the values were adjusted to obtain an equivalent sphere volume, and in the case of the prolate spheroid particle, the projected area was considered to be equivalent to the sphere projected area.

The sphericity Φ of the particles was calculated as follows [15]:

$$\Phi = \frac{6V_p}{D_{eq}A_p} \tag{1}$$

where V_p is the volume of the particle, A_p is the surface of the particle and D_{eq} is the equivalent diameter of the sphere of the same volume of the particle.

At least fifteen growth velocities (v) in the range of 1×10^{-16} m/s

Table 1Particles dimensions and surface shape.

Particle shape	Thickness (T) [µm]	R _{ext} [µm]	Surface shape	Sphericity Φ
Circular disk Hemisphere Truncated cone	R/10 - R/10	- - R + (R/ 10)	Flat Flat Flat	0,323 0,840 0,513
Cylinder Prolate spheroid Sphere	<i>R</i> /0.75 200 (c = 100) -	-	Flat Flat Flat, concave and convex	0,857 0,930 1

to 10^{-10} m/s were considered, covering the range of velocities where isotactic polypropylene is able to produce impurity segregation in spherulitic crystallization for micrometric particles [16]. Therefore, corresponding *Re* values were between 1×10^{-21} and 1×10^{-13} .

The boundary conditions in the model were: i) constant fluid velocity in the surface taken as a sink, ii) no-slip condition on the particle surface and iii) other boundaries: free. Newtonian fluid in a laminar flow regime, ($Re \ll 1$).

The particle radii (*R*) simulated were 50 μ m, 10 μ m and 1 μ m; and the values of *h*/2*R* were: 0.0001, 0.01, 0.05, 0.1, 0.5 and 1.

The physical system was simulated on an axisymmetric model which has been previously validated [17]. Similar flow behavior was found and drag force results for three dimensional and axisymmetric models were very close, with a small difference of 0.13%.

The numerical solutions of the problem includes the equations of conservation of mass and momentum. The domain was discretized using 30,000 and 50,000 quadrilateral elements, with second order interpolation functions for the velocity and first order for the pressure. The resulting system of equations was solved employing the Picard method [18]. The fluid flow was assumed to be that for a viscous fluid polymer at zero shear viscosity. Polymers at very slow shear rates display complete Newtonian behavior, and Stokes' law is recovered [10]. In all the cases, the viscosity (μ) of the melt was assumed to be constant and uniform with a value of $1 \times 10^{+3}$ Pa.s (properties of isotactic polypropylene), and a density (δ) of 2700 kg/m³ and 900 kg/m³ for particle and melt, respectively [19–20]. The drag force onto the particle was calculated numerically from the velocity field in the fluid flow model by solving the full Navier Stokes equations by a finite element method (Eq. (2)).

$$F_{i} = \int_{S} \varphi \overline{\sigma_{i}} dS = \int_{S} \varphi \sigma_{ij} n_{j} dS$$
⁽²⁾

where, F_i is the component of the force in the i-th direction, n_j is the versor in the j-th direction, σ_{ij} is the stress tensor, φ is the column vector of interpolation functions, and dS is the differential surface. The integration was performed on the whole surface *S* of the particle.

The stress tensor was obtained from the velocity field using the following equation:

$$t_i = \sigma_{ij} n_j, \qquad \sigma_{ij} = -p \delta_{ij} + \mu (u_{i,j} + u_{j,i})$$
(3)

where *p* is the pressure term, μ is the viscosity and $u_{i,j}$ and $u_{j,i}$ are the velocity gradients of the *i* and *j* components in the *j* and *i* directions, respectively.

3. Results and Discussion

The results of the drag force as a function of the separation distance h were rearranged to obtain dimensionless values (Eq. (4)) in order to compare with empirical expressions and for a better visualization.

$$\frac{F_{dSIM}}{6\pi\mu\nu R} = f\left(\frac{R}{h}\right) \tag{4}$$

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