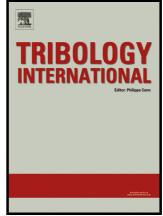
### Author's Accepted Manuscript

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## ACCEPTED MANUSCRIPT Effects of Angular Misalignment on the Performance of Rotor-Bearing Systems Supported by Externally Pressurized Air Bearing

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#### Abstract

In this study, the performance of a rotor system supported by externally pressurized air bearing was investigated. Angular misalignment was introduced to the bearing along two perpendicular direction separately, and then both at the same time (combined). The effect of angular misalignment to the pressure distribution in the air bearing was obtained. A clearance function was introduced for misalignment conditions of the bearing, and then the air flow between the rotor and the bearing was modeled using Reynold's equation. Next this flow model was solved using Differential Transform and Finite Difference hybrid numerical solution method to obtain the pressure distribution in the bearing. The effects of the angular misalignment conditions on the load carrying capacity were also analyzed as well.

Keywords: Misalignment, Air bearings, Hybrid numerical solution method

#### Nomenclature

Nomenclature	e
с	radial clearance discharge coefficient feeding hole diameter bearing diameter eccentricity center of mass air film thickness, $H = h/c$
$c_d$	discharge coefficient
$d_o$	feeding hole diameter
D	bearing diameter
е	eccentricity
G	center of mass
h, H	air film thickness, $H = h/c$
$h_{am}$	angular misalignment part of the air film thickness
$h_{rm}$	rocking motion part of the air film thickness
I(k)	convolution of <i>H</i> and <i>H</i>
J(k)	convolution of $I(l)$ and $H$
k, l, m	transform parameters
L	length of the bearing
m	mass flow rate
M	dimensionless mass flow rate
М, N	number of axial and circumferential grid points
<i>O</i> , <i>O</i> <sub>1</sub> , <i>O</i> ′	rotor center, bearing and journal center
$p_a$	atmospheric pressure
р, Р	air film pressure, $P = p/P_a$
$p_s, P_s$	supply pressure, $P_s = p_s / P_a$
$P_d$	dimensionless downward pressure at feeding holes
R	bearing radius
$R_0$	Gas constant
$T_0$	absolute temperature
t	time
U	velocity of the rotor surface
W	load carrying capacity
<i>x</i> , <i>y</i> , <i>z</i>	Cartesian coordinates axes on the rotor
$x_1, y_1, z_1$	Cartesian coordinates axes on the bearing
σ	squeezing number, $\sigma = (12\mu/Pa)(R/c)^2$
Λ	bearing number, $\Lambda = (6\mu\omega/Pa)(R/c)^2$
μ	dynamic viscosity of air
α	angular misalignment
κ	heat capacity ratio
ε	eccentricity ratio, $\mathcal{E}=e/c$
$\theta$	dimensionless coordinate axis in circumferential
	direction, $\theta = x/R$
$ heta_a$	attitude angle

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