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Research Paper

Development and experimental study of a small-scale compressed air radial inflow turbine for distributed power generation



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HIGHLIGHTS

- Proposing methodology for developing small-scale RIT as expander of CAES and ORC.
- 1D modelling enables estimation of turbine performance and obtaining key geometry.
- Turbine efficiency improved from 81.3% obtained by 1D model to 84.5% obtained by CFD.
- CFD predicts turbine efficiency and power with accuracy of ±16% and ±13% compared to tests.

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ABSTRACT

With ever increasing demand on energy, disturbed power generation utilizing efficient technologies such as compressed air energy storage (CAES) and organic Rankine cycle (ORC) are receiving growing attention. Expander for such systems is a key component and its performance has substantial effects on overall system efficiency. This study addresses such component by proposing an effective and comprehensive methodology for developing a small-scale radial inflow turbine (RIT). The methodology consists of 1-D modelling, 3-D aerodynamic investigation and structural analysis, manufacturing with pioneering technique and experimental testing for validation. The proposed 1-D modelling was very effective in determining the primary geometry and performance of turbine based on parametric studies of turbine input design variables. However with CFD analysis, it was shown that more efficient turbine geometry can be achieved that not only provides more realistic turbine performance by capturing the 3-D fluid flow behaviour but also improves turbine efficiency with the aid of parametric studies of turbine geometry parameters. Turbine efficiency was improved from 81.3% obtained from 1-D modelling to 84.5% obtained by CFD. Accuracy of the CFD model was assessed by conducting experiments on the RIT manufactured with stereolithography technique. The CFD model can predict turbine efficiency and power with accuracy of ±16% and ±13% respectively for a wide range of tested operating conditions. Such results highlights the effectiveness of the proposed methodology and the CFD model can be used as benchmarking model for analyses of small-scale RITs. Besides, it was shown that for such applications, the novel manufacturing technique and employed material are very effective for producing prototypes that assist design decisions and validation of CFD model with reasonable accuracy at reasonable cost and in timely manner.

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1. Introduction

Nowadays energy is a key factor in the global economy and the effectiveness of energy generation and consumption processes has remarkable impacts on our society and environment. Following the international energy agency (IEA) report [1], extending the current

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trend of energy consumption and energy efficiency to 2050 yields a growth of 70% and 60% in the global energy demands and emissions respectively compared to 2011. The associated emissions result in a long-term global average temperature rise of 6 °C by 2050 which can result in potentially devastating consequences such as climate change and energy security. IEA (2014) suggested an effective scenario called "2DS" which offers a vision for a sustainable energy system that reduces $\rm CO_2$ emissions to maintain the global temperature rise within 2 °C by 2050 and to limit increases in energy demand by 25% and cut emissions by 50%. This

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ymbols	. 25	ω	rotational velocity (RPM)
l	area (m²)		
)	blade height (m)	Subscrip	ots
3K	blockage factor (-)	1-5	stations across turbine
C	absolute flow velocity (m/s)	abs	absolute
C_{nozzle}	nozzle chord length (m)	b	back plate
C_p	specific heat capacity at constant pressure (J/kg K)	cfg	centrifugal
C_s	spouting velocity (J/kg) ⁻¹	Cor	coriolis
d	diameter (m)	f	friction
d_{hyd}	hydraulic diameter (m)	hub	rotor hub
EŘ	expansion ratio (-)	i	counter
f	friction coefficient (–)	m	meridional direction
f _{curve}	turbine friction factor (–)	r	radial
h	enthalpy (J/kg)	ref	reference
Δh_{actual}	actual specific enthalpy drop (J/kg)		
⊿h _{ideal}	ideal specific enthalpy drop (J/kg)	rel	relative
∆h _{losses}	enthalpy drop due to losses (J/kg)	rms	root mean square
I Hosses	rothalpy (J/kg)	rotor	rotor
k	coefficient (–)	s .	isentropic
l		sonic	sonic velocity (speed of sound)
	length (m)	X	axial direction
l_{hyd}	hydraulic length (m)	stage	turbine inlet to turbine outlet
m	mass flow rate (kg/s)	stat	static
Ма	Mach number (–)	t	total, stagnation
N_s	specific speed (–)	tip	rotor tip
	nozzle throat width (m)	ts	total to static
P	pressure (Pa)	θ	tangential direction
R	gas constant (J/kg K)		· ·
R_0	universal gas constant ((J/(k mol))	Supersc	rints
RR	relative roughness (m)	T	matrix transpose
Re	Reynolds number (–)	-	
r	radius (m)		average
r	location vector (m)		
r_c	mean radius of curvature (m)		natical operators
S	entropy (J/kg K)	∇	vector operator $\left \frac{\partial}{\partial x}, \frac{\partial}{\partial y}, \frac{\partial}{\partial z} \right $
\mathbf{S}_{E}	energy source (kg/(m s ³))	\otimes	dyadic operator (tensor product)
S_M	momentum source $((kg/(m^2 s^2))$		
	nozzle blade pitch (m)	Acronyn	ns
S _{nozzle}		CAES	compressed air energy storage
T	temperature (K)	CAD	computer aided design
t	time (s)	CFD	computational fluid dynamics
U	rotor blade velocity (m/s)	CHP	combined heat and power
U	vector velocity $\mathbf{U}_{x,y,z}$ (m/s)	CPG	centralized power generation
W	relative flow velocity (m/s)		
W	molecular weight (kg/mol)	DPG	distributed power generation
Z	Blade number (-)	DUI	datum universal interface
		EES	engineering equation solver
Greek let	tters	FEA	finite element analysis
α	absolute flow angle with respect to radial (degree)	GGI	general grid interface,
β	relative flow angle with respect to radial (degree)	IEA	international energy agency
γ	specific heat capacity ratio (-)	LE	leading edge
δ	identity matrix (–)	MOGA	multi-objective genetic algorithm
		ORC	organic Rankine cycle
E	clearance (m)	PS	pressure surface
ζ	shear stress tensor (kg/m s ²)	RANS	Reynolds-averaged Navier-Stokes
η	efficiency (–)	RIT	radial inflow turbine
λ	thermal conductivity (W/m K)	RPM	revolutions per minute
μ	dynamic viscosity (kg/(m s))	SS	suction surface
υ	isentropic velocity ratio (-)		
ho	density (kg/m³)	SST	shear stress transport
σ	solidity (–)	TE	trailing edge
φ	flow coefficient (–)		
ψ	loading coefficient (-)		

strategy creates a framework for a future sustainable energy systems which are expected to be smarter, renewable oriented, integrated, well-regulated and more distributed.

Improvements in energy efficiency have significant contribution to the "2DS" scenario. For example, power in the traditional electrical grid (or centralized power generation) followed one

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