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Research Paper

Influence of a circular strainer on unsteady flow behavior in steam turbine control valves

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HIGHLIGHTS

• Influence of strainer on unsteady flow dynamics in control valves was studied.

• A porous-medium model of strainer was established and experimentally validated.

• Oscillation of the annual flow around main valve's seat was attenuated by strainer.

• Continuous flow impingement onto the throttle valve's cavity wall was found.

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ABSTRACT

The influence of a circular strainer on unsteady flow behavior in steam turbine control valves, which are commonly placed between an intermediate-pressure turbine and a boiler in thermal power plants, was numerically studied. A porous-medium model, which established the dependencies of the pressure drop through the strainer on the magnitude and direction of the fluid flow's velocity, was validated by experimental measurements in a water flow test rig. As the benchmark configuration, a valve without a strainer was used for comparison. The turbulent steam flow in the complex serpentine channel was simulated with the implementation of the proposed porous model for the strainer. The numerical results demonstrated that placing the strainer in the main valve resulted in dramatic changes of the flow patterns in the main valve's chamber and its diffuser, and even in the downstream throttle valve. The complex steam flow in the main valve was efficiently managed by the circular strainer, significantly reducing the crosssectional force on the main valve's spindle; this is attributed to attenuated oscillation of the annular flow around the main valve's seat. As for the downstream throttle valve, the pressure drop and the fluctuating lateral force on the spindle were intensified, which was shown to be closely related to the continuous impingement of the flow onto the throttle valve's cavity wall. In comparison with the configuration without a strainer, the placement of the strainer in the main valve gave rise to a pair of intensified secondary vortices in the diffuser section behind the throttle valve.

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1. Introduction

As the major flow control unit in thermal power plants, steam turbine control valves are commonly integrated by the sequential placement of a main valve and a throttle valve; the main valve alerts to severe malfunction of the steam turbine by rapidly cutting off the steam supply, and the throttle valve operates in response to the required power output. In practice, the closely spaced

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configuration of these two valves creates a complicated serpentine channel, giving rise to a highly three-dimensional flow pattern. Previous studies have identified various unsteady flow behaviors inside control valves, for example, the impingement of shear flow [16], rotating pressure fluctuations [8], wall-attached jet flow and wall-detached jet flow [4]; these behaviors can have many cause-and-effect consequences, such as intensified noise [7], structural vibration [9] or even operational risk [10]. However, placing a circular strainer around the main valve which is commonly performed to prevent impurities coming from the boiler from reaching the downstream turbine stages, was found to considerably change the flow pattern [1,2]. Consequently, unsteady flow behaviors inside steam turbine control valves tend to show distinctly





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Nomenclature

		Greek Symbols	
Main symbols		θ	incident angle
$ ho_{in}^{*}$	stagnation pressure of valve inlet	ξ	pressure drop coefficient
V_{in}	velocity of valve inlet		
y^+	non-dimensional wall distance	Abbreviations	
ΔP	pressure drop	CFD	computational fluid dynamics
V	velocity in strainer	LES	large-eddy simulation
μ	viscosity	DES	detached-eddy simulation
D	thickness of strainer	SAS-F	scale-adaptive simulation with an embedded forcing
K _{perm}	permeability of strainer		zone
Kloss	quadratic loss coefficient of strainer	RANS	Reynolds-averaged Navier–Stokes
\tilde{F}_L	normalized lateral force on valve's spindle	SST	shear stress transport
\tilde{F}_A	normalized axial force on valve's spindle	DEH	digital electric hydraulic control system
F_L	actual lateral force on valve's spindle	LIC	line-integral convolution
F_A	actual axial force on valve's spindle	RMS	root mean square
		FFT	fast Fourier transform
		PSD	power-spectral density

different dynamics when a circular strainer is placed inside the main valve's chamber.

A literature survey shows that considerable efforts have been made to numerically determine the unsteady flow behaviors of various industrial valves [11,15]. A large-eddy simulation (LES) of the flow in a control valve by Morita et al. [8] revealed rotating pressure fluctuations caused by the asymmetric flow around the valve spindle, which was determined to be the excitation source of valve vibration. Detached-eddy simulations (DES) of unsteady flows in a steam turbine throttle valve by Zeng et al. [14] demonstrated that the alternating flow pattern induced a low-frequency intermittent vibration of the valve's spindle. Domnick et al. [4] numerically simulated the time-variant flow field of steam turbine control valves using a scale-adaptive simulation with an embedded forcing zone (SAS-F), identifying three different flow topologies in the valve's diffuser section: full diffuser flow, wall-attached jet flow and wall-detached jet flow. Unfortunately, numerical simulations using SAS-F, DES and LES are extremely computationally expensive for unsteady internal flows in complex valves at high Reynolds numbers; hence, the cost-effective prediction of strongly unsteady flows using unsteady Reynolds-averaged Navier-Stokes (RANS) simulations at an allowable discrepancy is highly desirable. Using the shear stress transport (SST) model, Tecza et al. [12] numerically analyzed the flow-induced vibration of a steam turbine control valve; the flow instability within the pressurebalanced valve plug was successfully predicted to be the cause of valve vibration, as the self-induced pressure oscillations locked onto the valve chamber's natural frequency. Clari et al. [3] also performed SST simulations of the flow in a control valve. These results, along with the measurement of the dominant frequency, elucidated the behaviors of three-dimensional flow separation in the valve's diffuser section. However, to the best of the authors' knowledge, no studies have been conducted to assess the influence of a strainer on unsteady flow behaviors in control valves.

In the present study, the main focus is the influence of a circular strainer, placed in the main valve, on unsteady flow behaviors in steam turbine control valves, as shown in Fig. 1. To this end, a porous model of the strainer was established, incorporating the directivity of the pressure loss and allowing variation of the approaching flow directions and flow speeds. For comparison, the same valves without the inclusion of the strainer were used as the benchmark configuration. SST simulations of the unsteady steam flow field in the valves were carried out. The influence of the strainer on the unsteady flow behaviors was discussed in terms of the fluctuating aerodynamic forces on the valve's spindle, the





(b) Geometry of circular strainer (mm)

Fig. 1. Schematic illustration of steam turbine control valves and strainer.

pressure fluctuations inside the valves, the annular flow oscillations around the main valve's seat and the impingement flow onto the throttle valve's cavity.

2. Numerical approach

2.1. Numerical setup

The configuration of the steam turbine control valves is shown in Fig. 1a, in which the different components, i.e., the valve's Download English Version:

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