



Research Paper

One-dimensional simulation for attemperator based on commissioning data of coal-fired steam power plant



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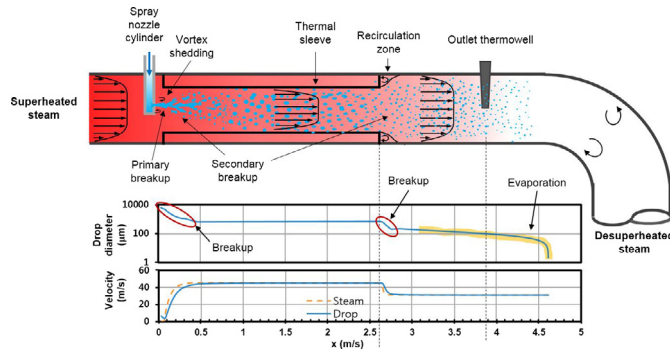
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HIGHLIGHTS

- An attemperator is a device to spray water into the superheated steam.
- The evaporation was analyzed using the enthalpy balance from the commissioning data.
- The spray atomization and its concurrent evaporation in an attemperator were physically modeled.
- A simple one-dimensional simulation was conducted to verify the commissioning results.

GRAPHICAL ABSTRACT



ARTICLE INFO

Article history:

Received 14 July 2016

Revised 30 October 2016

Accepted 8 November 2016

Available online 9 November 2016

Keywords:

Attemperator

Power plant

Spray

Atomization

Droplet evaporation

ABSTRACT

An attemperator is a device that is used to spray water into the superheated steam between the primary, platen, and final superheaters and the reheat lines. The goal of the attemperator is to control the temperature of the superheated steam in accordance with desired turbine-inlet temperature during both steady-state and transient operation. Because the thermowell installed at the attemperator outlet is tied back to the feedback control of the spray water, the spray water should evaporate ahead of the thermowell for accurate control of the steam temperature. In this work, the completion of the evaporation ahead of the thermowell was analyzed using the enthalpy balance from the start-up commissioning data of an 800-MW coal-fired steam power plant. In addition, the phenomena of the spray atomization and its concurrent evaporation in an attemperator were physically modeled, and a simple one-dimensional simulation was conducted to verify the analysis of the commissioning data.

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1. Introduction

The boiler system in a coal-fired power plant includes various devices to produce superheated steam. Briefly, the steam separated from a drum is superheated through primary, plate-shaped

(platen), and final superheaters in series and flows into a high-pressure turbine (Fig. 1). It is well known that the steam temperature should be increased to improve the thermal efficiency of power plants, but this temperature is limited to approximately the creep point of the superheater tubing material. On the contrary, if the temperature of superheated steam is below a setpoint, the thermal efficiency of the power plant decreases, and the mechanical and thermal damages due to the decrease in the steam quality can shorten the lifespan of the turbine blades. In addition, the

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Nomenclature

A	area, m^2	We	Weber number
B_H	heat transfer number	x	distance from the nozzle exit, m
BF	blowing factor	<i>Greek symbols</i>	
c_p	specific heat capacity, $kJ\ kg^{-1}\ K^{-1}$	β	evaporation coefficient
C_d	drag coefficient	μ	viscosity, $kg\ m^{-1}\ s^{-1}$
d	droplet diameter, m	ρ	density, $kg\ m^{-3}$
D	diameter, m	σ	surface tension, $N\ m^{-1}$
h	heat transfer coefficient, $W\ m^{-2}\ K^{-1}$	<i>Subscript</i>	
h_{eff}	effective heat transfer coefficient, $W\ m^{-2}\ K^{-1}$	0	initial
h	enthalpy, $kJ\ kg^{-1}$	br	breakup
h_{fg}	latent heat of vaporization, $kJ\ kg^{-1}$	d	droplet
l	turbulent intensity	D	pipe diameter
k	thermal conductivity, $W\ m^{-1}\ K^{-1}$	in	inlet
L	length, m	l	liquid
m	mass, kg	L	left
\dot{m}	mass flow rate, $kg\ s^{-1}$	N	nozzle
m''_{evap}	evaporative mass flux, $kg\ m^{-2}\ s^{-1}$	out	outlet
n	number of droplets	p	pipe
Nu	Nusselt number	R	right
Pr	Prandtl number	s	steam
P_G	generator load, W	sat	saturation
Re	Reynolds number	TS	thermal sleeve
Re'	turbulent Reynolds number	TW	thermowell
t	time, s	v	vapor
T	temperature, $^{\circ}C$	w	water
$T_{s,desuper}$	temperature of completely desuperheated steam, $^{\circ}C$		
U	relative velocity, $m\ s^{-1}$		
v'	root-mean square of the turbulent velocity fluctuation, $m\ s^{-1}$		
V	average velocity, $m\ s^{-1}$		

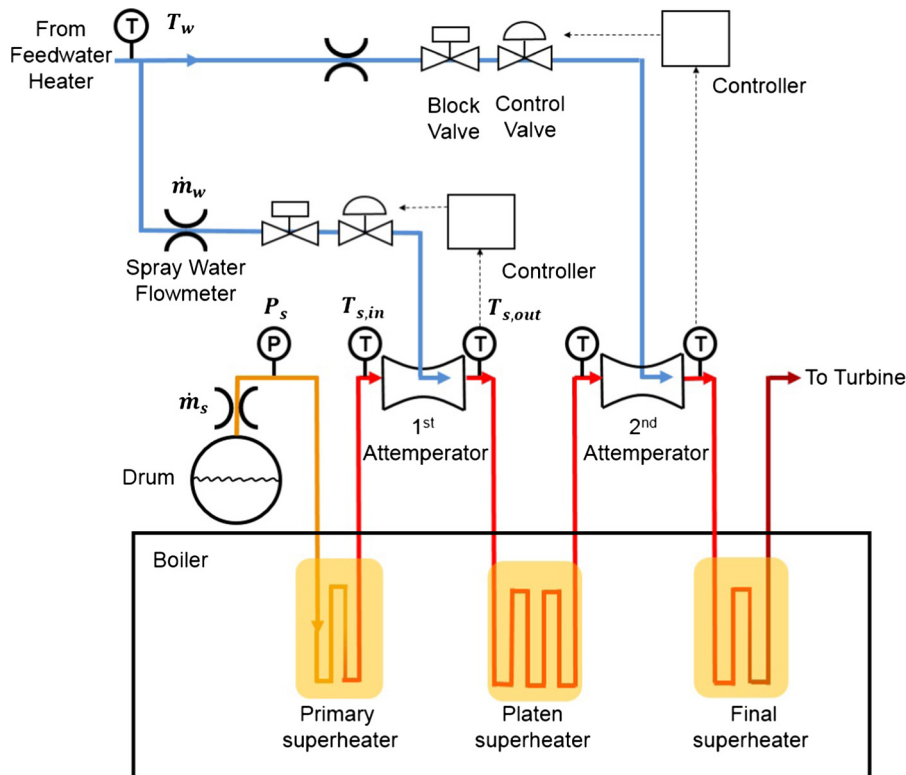


Fig. 1. Typical arrangement of superheaters and attemperators.

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