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Convective heat transfer of molten salt in the shell-and-tube heat exchanger with segmental baffles



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ABSTRACT

In this paper, a special flow layout with U-shaped tubes applied in the laboratory was designed for testing the heat transfer performances (HTPs) of molten salt in the shell side of a shell-and-tube heat exchanger (STHE). Based on this design, the transitional convective HTPs (6142 < Re < 9125) of molten salt with higher temperature (209.41-241.49 °C) in the STHE with segmental baffles (STHE-SBs) were experimentally studied, and the corresponding heat transfer correlations were fitted. Then the validation of the traditional correlation applied in the molten salt STHE-SBs was conducted. Finally, the comparison of heat transfer enhancement in the molten salt STHE-SBs was discussed. The results show that a good agreement with 2% deviations exists between the fitted correlations and the test data. Meanwhile, the traditional Kern correlation, with a 7.1% maximum deviation compared with the test data, is still appropriate to analyze the HTPs of molten salt STHE-SBs. The effects of segmental baffles on the molten salt heat transfer enhancement in lower flow rate region are better than those in higher flow rate region, and the maximum increment of Nusselt number is 26%.

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1. Introduction

The concentrating solar power (CSP) technology is one of the significant ways to solve the problem of environment pollution and energy shortage [1–3,30]. In the CSP system, the inorganic molten salt with high specific heat, low viscosity and operation pressure has commonly been adopted as the heat transfer fluid [4–8]. Besides, the molten salt always works in high temperature. For example, the operation temperatures of solar salt (60 wt% NaNO₃, 40 wt% KNO₃) and HITEC salt (53 wt% KNO₃, 40 wt% NaNO₂, 7 wt% NaNO₃) applied in the CSP system are 220–600 °C and 142–550 °C, respectively. Especially, the molten salt with high temperature could also be applied in the nuclear power system [9].

In the solar and nuclear plants, the molten salt shell-and-tube heat exchanger (STHE) is one of the critical units where the water medium could be heated sharply and then transferred into superheated steam with higher pressure and temperature. Thus, the thermal efficiency of plants is influenced significantly by the heat transfer performances (HTPs) of the molten salt STHE. Particularly, the molten salt STHE with segmental baffles (STHE-SBs) is one of the most common heat exchangers, which can improve the HTPs by enhancing the turbulence in the shell side of STHE-SBs. For the molten salt STHE-SBs, there are two special operation characteristics which need to be demonstrated. Firstly, the inlet molten salt could be heated to high temperature by the solar collectors [6,7,10–12] or nuclear reactors [9,13,14], as shown in Table 1. Secondly, the molten salt with low pressure (around the ambient pressure) always flows into the shell side of STHE-SBs.

Specially, the appropriate heat transfer correlations applied in molten salt with high temperature are needed when the STHE-SBs is designed. However, the traditional heat transfer correlations applied in the molten salt STHE-SBs [15] such as the Kern and Donohue equations were fitted based on the test data of water and oil etc. with low temperatures. For example, the Donohue equation was fitted by the experimental data of water and oil fluids around 65 °C [16]. Therefore, it is not clear that whether the traditional heat transfer correlations are appropriate or not in the optimization design of the STHE-SBs with molten salt above 142 °C, such as the solar salt and HITEC salt. Table 2 shows the traditional heat transfer correlations applied in the STHE-SBs.

Besides, although many studies on the HTPs of molten salt have been conducted, most of researches mainly focus on the analysis in the single tube, the annual tube and the STHE. For the single tube, the forced convective HTPs of LiNaK salt (46.5 wt% LiF, 11.5 wt% NaF, 42 wt% KF) have been experimentally carried out by Hoffman et al. [17], and the corresponding heat transfer correlations were fitted. Liu et al. [18] experimentally analyzed the turbulence con-

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Nomenclature

| A _c A _s B C _p Ct | maximum flow area of shell (m ²) heat transfer area of shell (m ²) pitch of segmental baffles specific heat (J·kg ⁻¹ ·K ⁻¹) factor | Abbrevia SPRT STHE STHE-SI | ations standard platinum resistance thermocouple shell-and-tube heat exchanger Bs shell-and-tube heat exchanger with segmental baffles |
|--|--|---|---|
| C _t d d _e D f h k K L N _b Nt Nu Pr q _v Q Re s T ΔT ΔT _m u | ractor diameter of tubes (m) equivalent diameter of tubes (m) diameter of shell (m) equivalent diameter of shell (m) friction coefficient heat transfer coefficient (W·m ⁻² ·K ⁻¹) conductivity (W·m ⁻¹ ·K ⁻¹) overall heat transfer coefficient (W·m ⁻² ·K ⁻¹) length of tubes (m) number of segmental baffles number of segmental baffles number of tubes Nusselt number Prandtl number Prandtl number volume flow rate (m ³ ·s ⁻¹) heat transfer capacity (W) Reynolds number pitch of tubes (m) temperature (K) temperature difference (K) log-mean temperature difference (K) flow rate (m·s ⁻¹) | Greek sy μ Φ σ Superscr ave corr f i in o out max min s t w | ymbols viscosity (Pa·s) density (kg·m ⁻³) heat balance deviation error ripts average correction fluid inner inlet outer outlet maximum minimum shell side tube side wall |
| | | | |

Table 1

Operation temperatures of molten salt in the heat exchangers of solar and nuclear power plants.

| Applications | Items | Projects | Countries | Years | Temperatures of molten salt ($^{\circ}$ C) |
|----------------|---------------------|----------------|-----------|-------|---|
| Solar plants | Tower | Delhi | China | 2016 | 568 |
| | | Crescent Dunes | US | 2015 | 550 |
| | | Gemasolar | Spain | 2011 | 565 |
| | | Solar Two | US | 1996 | 565 |
| | | MSEE | US | 1984 | 566 |
| | | THEMIS | France | 1984 | 450 |
| | Parabolic Trough | Aksay | China | 2016 | 550 |
| | | Archimedes | Italy | 2010 | 550 |
| | | Andasol | Spain | 2009 | 390 |
| | Linear Fresnel | Areva | US | 2014 | 550 |
| Nuclear plants | Molten salt reactor | MOSART | Russia | 2014 | 715 |
| | | TMSR | China | 2013 | 650 |
| | | FHR | US | 2012 | 1000 |
| | | MSFR | France | 1997 | 650 |
| | | MSR | US | 1976 | 705 |
| | | MSRE | US | 1963 | 705 |
| | | ARE | US | 1954 | 815 |

Table 2

Traditional heat transfer correlations applied in STHE-SBs.

| Items | Equations | Range | Years |
|--|---|---------------------------------|--------------|
| Kern method [15] | $\begin{aligned} Nu &= 0.5 Re^{0.507} Pr^{1/3} (\mu/\mu_w)^{0.14} \\ Nu &= 0.36 Re^{0.55} Pr^{1/3} (\mu/\mu_w)^{0.14} \end{aligned}$ | 10 < Re < 2000 Re > 2000 | 1950 |
| Donohue method [16] Bell-Delaware method [15] | $Nu = 0.23Re^{0.6}Pr^{1/3}(\mu/\mu_w)^{0.14}$ $h = j_{\rm H}.G_{\rm s}.c_{\rm p}(\mu/\mu_w)^{0.14}Pr^{-2/3}.J_{\rm c}.J_{\rm e}.J_{\rm b}.J_{\rm s}.J_{\rm r}$ | 20,000 < <i>Re</i> < 30,000 | 1949 1947 |

vective HTPs of $LiNO_3$ salt in a single tube, and the experimental correlation was obtained. Shen et al. [19] also analyzed the heat transfer characteristics of HITEC salt in a single tube with non-uniform heat flux on the tube walls, and the corresponding

correlation with heat flux was fitted. The heat transfer and friction characteristics of LiNaK salt were numerically investigated in the smooth tube [20]. Meanwhile, the heat transfer enhancement of spiral ridges and transversally corrugated structures applied in

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