Contents lists available at ScienceDirect



International Journal of Heat and Mass Transfer

journal homepage: www.elsevier.com/locate/ijhmt

## Novel methods of oil fouling inhibition on surface of plate heat exchanger in simulated oilfield geothermal water



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#### ARTICLE INFO

Article history: Received 24 October 2016 Received in revised form 23 May 2017 Accepted 26 May 2017

Keywords: Crude oil soiling Plate heat exchanger Surface hydrophilization Flow velocity Chemical additive

### ABSTRACT

Crude oil fouling on heat transfer surface (HTS) is a very troublesome problem for the utilization of oilfield geothermal water in which slight crude oil exists. The deposition of crude oil on HTS will decrease heat transfer efficiency, increase the pressure drop of fluid flow, and even block flow channel. In this study, surface hydrophilization via anodization, flow velocity enhancement and chemical addition three measures were adopted to solve the problem of crude oil soiling via weakening the adhesion force between crude oil and HTS as well as intensifying the interaction between water and crude oil. Contact angle measurement and microscopic morphology study were conducted on the highlyhydrophilic surface fabricated via anodization. Oil-soiling experiments accompanied with three mentioned techniques were carried out in plate heat exchanger (PHE) apparatus, and obvious oil-fouling inhibition effects were presented. The effect was reflected by the decrease of fouling thermal resistance. For 110 °C simulated oilfield geothermal water, which contained approximate 1 vol.% crude oil soiling, with 0.24 m/s flow velocity, the fouling induction period extended to 500 min from 0 min when surface hydrophilization was applied through anodization. In anodized PHE with titanium plates, the value of fouling thermal resistance decreased by about 77.8% when the flow velocity of 110 °C hot fluid without chemical additive increased from 0.24 m/s to 0.72 m/s, and the value could be reduced by around 62.5% after the addition of 0.29 g/L chemical additive LIUXU-10 mainly composed of sodium dodecyl sulfonate (SDS), when the flow velocity of 110 °C hot fluid was 0.12 m/s. Compared with PHE with titanium plates (Ti-PHE) at 0.24 m/s hot fluid flow velocity, the fouling thermal resistance decreased by approximate 93.3% when the Ti-PHE was anodized to be highly hydrophilic, the hot fluid flow velocity increased to 0.6 m/s and 0.87 g/L chemical additive was added.

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### 1. Introduction

Due to the excessive exploitation of fossil energy such as petroleum and natural gas, people start to seek newer substituted energy source. With the advantages of wide distribution, abundant reserves, low cost and low contamination, geothermal energy gradually becomes a kind of potential new energy [1]. While scaling or fouling in heat exchanger is a ubiquitous problem existing in the utilization of geothermal source. Fouling problems, which cause severe decrease of heat transfer coefficient in heat exchanger, exist in about 90% heat exchangers [2] and bring economic loss which takes up to 0.25% of GNP (Gross National Product) at developed country [3]. Therefore, it is very necessary to understand the process of fouling deposition and exploit new methods to solve the heat exchanger fouling problems.

Plenty of researches have been focused on the fouling behaviors through experiments or modelling [4–8]. Besides, many antifouling methods have also been attempted to mitigate the fouling attachment on HTS. These methods include electromagnetism antifouling technology [9–11], ultrasonic antifouling technology [12,13], optimizing operation parameters or HTS textures [5,14,15], physical or chemical addition [16–19], and surface coating [20–34]. For electromagnetism antifouling technology, the scaling will become fragmented and loose under electric field, which contributes to the fouling mitigation on HTS [11]. For ultrasonic antifouling

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Nomenclature

Nomeno	clature		
$A_0$	effective heat transfer area $(m^2)$	$\Delta P$	pressure drop between hot fluid outlet and hot fluid in-
A'	projected heat transfer area (m <sup>2</sup> )		let (kPa)
$C_p$	fluid specific heat capacity at constant pressure (J/	$\Delta P_f$	friction pressure drop of hot fluid (Pa)
Р	(kg·K))	$\Delta P_h$	flow channel pressure drop of hot fluid in PHE without
$C_{p,h}$	hot fluid specific heat capacity at constant pressure (J/	<i>n</i>	fouling deposition (Pa)
⊂p,n	$(kg\cdot K))$	$\Delta P'_h$	flow channel pressure drop of hot fluid in PHE with foul-
C	cold fluid specific heat capacity at constant pressure (J/	$\Delta h$	ing deposition (Pa)
$C_{p,c}$		-	
с и –	(kg·K))	$\tau_f$	friction shear stress (N)
С, К <sub>р</sub> , <i>z</i>	constants	$\Delta G_{SWO}^T$	interfacial free energy between solid surface and oil sur-
$D_p$	port diameter (mm)		face in aqueous medium (mJ/m <sup>2</sup> )
$d_e$	equivalent diameter of fluid channel in PHE (m)	$\Delta G_{SO}^T$	interfacial free energy between solid surface and oil sur-
$f_p$	mean cross-sectional flow area per channel (m <sup>2</sup> )	<b>T</b>	face (mJ/m <sup>2</sup> )
f	friction coefficient	$\Delta G_{OO}^T$	interfacial free energy between oil surface and oil sur-
Н	corrugation amplitude or mean channel spacing (mm)		face (mJ/m <sup>2</sup> )
h	mean flow channel spacing (m)	$\theta_1$	water contact angle on substrate (°)
$L_{v}$	vertical distance between centers of ports (mm)	$\theta_2$	glycerol contact angle on substrate (°)
L <sub>h</sub>	horizontal distance between centers of ports (mm)	$\theta_3$	diiodomethane contact angle on substrate (°)
L <sub>w</sub>	plate width inside gasket (mm)	$\theta_{OW}$	intersection angle between solid-oil interfacial tension
L <sub>eff</sub>	effective flow length in PHE (m)	011	and oil-water interfacial tension (°)
-e,,,	length of fluid flow (m)	$\theta_{PTFE}$	contact of water with the addition of LIUXU-10 on PTFE
М	number of passes in PHE	OPIFE	(°)
P	corrugation pitch (mm)	0	contact of water with the addition of LIUXU-10 on
	fluid Prandtl number	$\theta_{PMMA}$	PMMA (°)
Pr <sub>f</sub>	hot fluid Prandtl number	$\gamma_{S}^{LW}$	
$Pr_h$		γs	Lifshitz-van der Waals component of surface energy for
$Pr_c$	cold fluid Prandtl number	114/	substrate (mJ/m <sup>2</sup> )
$Q_c$	heat transfer load of cold fluid (W)	$\gamma_W^{LW}$	Lifshitz-van der Waals component of surface tension for
$R_f$	fouling thermal resistance ((m <sup>2</sup> ·K)/W)		water (mJ/m <sup>2</sup> )
Re <sub>f</sub>	fluid Reynolds number	$\gamma_{O}^{LW}$	Lifshitz-van der Waals component of surface tension for
Re <sub>h</sub>	hot fluid Reynolds number		oil soiling (mJ/m <sup>2</sup> )
Re <sub>c</sub>	cold fluid Reynolds number	$\gamma_s^{+}$	electron-acceptor component of surface energy for sub-
$R_a$	arithmetical mean deviations of profile ( $\mu$ m)		strate (mJ/m <sup>2</sup> )
$T_{h,i}$	hot fluid inlet temperature (°C)	$\gamma_{W}^{+}$	electron-acceptor component of surface tension for
$T_{h,o}$	hot fluid outlet temperature (°C)		water $(mI/m^2)$
$T_{c,i}$	cold fluid inlet temperature (°C)	$\gamma \dot{o}$	electron-acceptor component of surface tension for oil
$T_{c,o}$	cold fluid outlet temperature (°C)	10	soiling $(mJ/m^2)$
U	overall heat transfer coefficient ( $W/(m^2 \cdot K)$ )	$\gamma \overline{s}$	electron-donor component of surface energy for sub-
$U_0$	initial overall heat transfer coefficient $(W/(m^2 \cdot K))$	73	strate (m]/m <sup>2</sup> )
V	volumetric flow rate of fluid $(m^3/s)$	γw	electron-donor component of surface tension for water
$V_h$	volumetric flow rate of hot fluid $(m^3/s)$	<i>YW</i>	$(mJ/m^2)$
	flow velocity of hot fluid (m/s)	<u></u>	electron-donor component of surface tension for oil
$v_h$		γō	
$V_c$	volumetric flow rate of cold fluid (m <sup>3</sup> /s)	Ysw	soiling (mJ/m <sup>2</sup> )
			solid-water interfacial tension (mN/m)
Greek sy		γso	solid-oil interfacial tension (mN/m)
$\delta$	plate sheet thickness (mm)	γow	oil-water interfacial tension (mN/m)
$\Phi$	enlargement factor	$\gamma_{water}^{LW}$	Lifshitz-van der Waals component of surface tension for
ρ	fluid density (kg/m <sup>3</sup> )		water with the addition of LIUXU-10 (mJ/m <sup>2</sup> )
$\rho_h$	density of hot fluid (kg/m <sup>3</sup> )	$\gamma_{water}^{+}$	electron-acceptor component of surface tension for
$\rho_c$	density of cold fluid $(kg/m^3)$		water with the addition of LIUXU-10 (mJ/m <sup>2</sup> )
η	experimental relative error of overall heat transfer coef-	$\gamma_{water}^{-}$	electron-donor component of surface tension for water
'1	ficient (%)	, mater	with the addition of LIUXU-10 (mJ/m <sup>2</sup> )
~	convective heat-transfer coefficient (W/(m <sup>2</sup> ·K))	Ywater	surface tension for water with the addition of LIUXU-10
α		7 water	(mN/m)
$\alpha_h$	convective heat-transfer coefficient of hot fluid (W/	$\gamma_{PTFE}^{LW}$	Lifshitz-van der Waals component of surface energy for
	(m <sup>2</sup> K))	<i>YPIFE</i>	PTFE $(mJ/m^2)$
$\alpha_c$	convective heat-transfer coefficient of cold fluid (W/	LW	
	$(m^2 \cdot K))$	γ <sup>LW</sup> γPMMA	Lifshitz-van der Waals component of surface energy for
υ	fluid flow velocity in PHE channel (m/s)	_	PMMA (mJ/m <sup>2</sup> )
$v_h$	hot fluid flow velocity in PHE channel (m/s)	<i>γ</i> ₽́мма	electron-donor component of surface energy for PMMA
$\mu_f$	fluid viscosity in flow channel of PHE (Pa·s)		$(mJ/m^2)$
$\mu_w$	fluid viscosity at wall temperature in PHE (Pa·s)		
$\mu_h$	hot fluid viscosity in flow channel of PHE (Pa·s)	Subscrip	ts
$\mu_c$	cold fluid viscosity in flow channel of PHE (Pa·s)	C	cold fluid
$\lambda_f$	fluid thermal conductivity (W/(m·K))	p	constant pressure
$\lambda_h$	hot fluid thermal conductivity $(W/(m \cdot K))$	р e	equivalent
$\lambda_h$ $\lambda_c$	cold fluid thermal conductivity (W/(m·K))	v	vertical
	thermal conductivity of titanium (W/(m·K))	h	horizontal/hot fluid
λ	thermal conductivity of titalliulli (W/(III·K))	п	nonzontal/not nutu

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