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journal homepage: www.elsevier.com/locate/issn/15375110

### **Research** Paper

# Day-to-night heat storage in greenhouses: 1 Optimisation for periodic weather



Ido Seginer <sup>a,\*</sup>, Gerrit van Straten <sup>b</sup>, Peter J.M. van Beveren <sup>b,1</sup>

<sup>a</sup> Technion, Haifa, Israel <sup>b</sup> Wageningen UR, Wageningen, The Netherlands

#### ARTICLE INFO

Article history: Received 24 October 2016 Received in revised form 14 June 2017 Accepted 22 June 2017 Published online 24 July 2017

Keywords: Greenhouse Heat buffer Periodic weather Optimal control Constant co-state CO<sub>2</sub> enrichment

Day-to-night heat storage using water tanks (buffers) is common practice in cold-climate greenhouses, where gas is burned during the day for carbon dioxide enrichment. In this study an optimal control approach is outlined for such a system, based on the idea that the virtual value (shadow price) of the stored heat, its 'co-state', could be used to guide the instantaneous control decisions. If this value is high, the system has an incentive to fill the heat storage (buffer), and vice versa if the co-state is low. The optimal co-state trajectory maximises the net income (performance criterion). To illustrate the method, a system description and a parameter-set roughly representative of tomato greenhouses in The Netherlands is used. The results, for daily-periodic weather, show: (1) The optimal co-state is constant (same value night and day), in contrast to the varying set-points and control fluxes. (2) The optimal solution is associated with minimum time on the storage bounds (minimum time of full or empty buffer). (3) The optimal virtual value (co-state) of stored heat is about the same as the actual cost of boiler heat during winter and about zero in summer. (4) The gain from installing a buffer is highest during spring and minimal in winter. (5) The intensive utilisation of the heat buffer in summer and its low utilisation in winter indicate that the justification of the heat storage practice, under the assumed conditions, is more the need for CO<sub>2</sub> enrichment in summer than the need for heating in winter.

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#### 1. Introduction

In cold-climates, where natural gas is burned during the day to enrich greenhouses with carbon dioxide (CO<sub>2</sub>), water tanks (heat buffers) are often used to store extra daytime heat for heating at night (De Zwart, 1996; Salazar, Miranda, Schmidt, Rojano, & Lopez, 2014). Attempts to utilise this technique in milder climates have also been reported (Bailey et al., 2012), although our calculations (not shown) do not seem to justify its use in mild climates. The inverse approach, of night-to-day storage of  $CO_2$  in activated carbon, has also been tried (Sánchez-Molina, Reinoso, Acién, Rodríguez, & López, 2014).

There are several possible, and actual, configurations of such facilities; however, the focus of the present study is not on a particular configuration, but rather on an optimal

\* Corresponding author.

E-mail address: segineri@tx.technion.ac.il (I. Seginer).

<sup>&</sup>lt;sup>1</sup> Now at B-Mex, Wageningen, The Netherlands.

http://dx.doi.org/10.1016/j.biosystemseng.2017.06.024

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Notatior	n	Х	$CO_2$ flux, mol [C] m <sup>-2</sup> [ground] s <sup>-1</sup>
*	units may differ among vector elements	х	vector of state variables, *
[ground]	greenhouse ground surface	Y	growth rate of salable fruit (yield), mol [fruit-C] $m^{-2}$ [ground] $s^{-1}$
Symbols		β	temperature exponent of respiration, K <sup>-1</sup>
a	light extinction coefficient, m <sup>2</sup> [ground] mol <sup>-1</sup> [C]	Г	gain from installing a buffer ( $\equiv J\{S_c\} - J\{0\}$ ), \$ m <sup>-1</sup>
В	Bowen (sensible to latent heat) ratio, –		[ground]
С	$CO_2$ concentration, mol [C] m <sup>-3</sup> [air]	ε	efficiency of heat storage, –
с	specific heat of air, J [heat] $kg^{-1}$ [air] $K^{-1}$	$\eta_{FH}$	heating coefficient of global (solar) radiation, J
e	vector of exogenous (weather) variables, *		[heat] J <sup>-1</sup> [global]
F	global (solar) radiation flux, J [global] $m^{-2}$ [ground] $s^{-1}$	$\eta_{FL}$	conversion factor solar energy to photosynthetic light, mol [PAR] J <sup>-1</sup> [global]
f	sunlit leaf area index, m $^2$ [sunlit-leaf] m $^{-2}$	$\eta_{ m HX}$	conversion factor heat to $CO_2$ , mol [C] $J^{-1}$ [heat]
5	[ground]	$\eta_{LX}$	conversion factor light to CO <sub>2</sub> (photosynthetic
f	vector function describing state rate-of-change, *		'efficiency'), mol [C] mol <sup>-1</sup> [PAR]
g	running (control) costs, $m^{-2}$ [ground] s <sup>-1</sup>	ζ	fraction growth of saleable fruit out of total
H	heat flux, J [heat] $m^{-2}$ [ground] $s^{-1}$		growth, –
${\cal H}$	Hamiltonian, $m^{-2}$ [ground] $s^{-1}$	κ	temperature correction coefficient, $K^{-2}$
h	termination value, \$ m <sup>-2</sup> [ground]	Λ	co-state of S, $J^{-1}$ [heat]
J	performance criterion (objective function), $\$ m <sup>-2</sup>	ρ	air density, kg [air] m <sup>-3</sup> [air]
	[ground]	σ	leaf conductance to $CO_2$ , m <sup>3</sup> [air] m <sup>-2</sup> [sunlit-leaf
L	photosynthetic light flux, mol [PAR] m <sup>-2</sup>		s <sup>-1</sup>
	[sunlit-leaf] $s^{-1} = mol [PAR] m^{-2} [ground] s^{-1}$	au	transmissivity of greenhouse-cover to light, —
М	carbon content of crop, mol [C] $m^{-2}$ [ground]	Subscripts	
Ν	growth rate of non-fruit organic matter, mol [C]		-
	$m^{-2}$ [ground] $s^{-1}$	A	dissipated to atmosphere
Р	gross photosynthesis rate, mol [C] $m^{-2}$	В	supplied from boiler
	[sunlit-leaf] s <sup>-1</sup>	C D	installed capacity
р	gross photosynthesis rate at optimal temperature,	D	day dia kanala
	mol [C] $m^{-2}$ [sunlit-leaf] $s^{-1}$	d F	discharging
р	vector of co-state variables, *	F	due to global (solar) radiation
q	temperature response of photosynthesis, –	f C	final
R	respiration rate, mol [C] m <sup>-2</sup> [sunlit-leaf] s <sup>-1</sup>	G i	to greenhouse
S	stored heat, J [heat] m <sup>-2</sup> [ground]	•	indoor
S <sub>D</sub>	daytime storage requirement, J [heat] $m^{-2}$	max	maximum value
	[ground]	min	minimum value
S <sub>N</sub>	nighttime storage requirement, J [heat] ${ m m}^{-2}$	N	night
	[ground]	0	outdoor
S	slope of stored heat trajectory, J [heat] $m^{-2}$	р	optimal for photosynthesis
	[ground] s <sup>-1</sup>	r	at reference temperature
Т	air temperature, K, °C	S	heat storage (in buffer)
t	time, s	Т	total loss from greenhouse
U	overall heat transfer coefficient across greenhouse	t	on upper storage bound
	cover, J [heat]m <sup>-2</sup> [ground] K <sup>-1</sup> s <sup>-1</sup>	u V	charging
u <sub>B</sub>	unit price of boiler heat, $J^{-1}$ [heat]	V	by ventilation
u <sub>Q</sub>	unit price of ventilation, $m^{-3}$ [air]	Acronyms	
u <sub>Y</sub>	unit market price of produce (fruit) dry matter, \$	D	Dimension
	mol <sup>-1</sup> [fruit-C]	PAR	Photosynthetically Active Radiation
u	vector of control variables, *		

strategy to control the operation of such systems. Attempts to solve the  $CO_2$  enrichment problem in conjunction with heat buffers have been made before (Aikman, Lynn, Chalabi, & Bailey, 1997; Chalabi, Biro, Bailey, Aikman, & Cockshull, 2002). These, however, considered the heating needs (setpoints) separately from  $CO_2$  enrichment and did not treat the

heat flux in and out of the buffer as a control variable. Here the method of optimal control is followed (Pontryagin, Boltyansky, Gamkrelidze, & Mischenko, 1962), the basic idea being that the co-state of the stored heat, namely its current virtual (marginal, shadow) value, could be used to guide the instantaneous control decisions. If this value is high, the Download English Version:

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