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Passive cooling potential in buildings under various climatic conditions in India



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ABSTRACT

Passive cooling is a versatile technology through which energy demands to achieve the thermal comfort in a building can be minimized in places like India, where different climatic zones exist. This paper presents the outcome of various research works carried out in the field of evaporative cooling, nocturnal radiative cooling, and phase change material (PCM) based free cooling of buildings under various climatic conditions. Further, the developed empirical equations/correlations for determining the critical design parameters pertaining to the above said passive cooling technologies are summarized in order to provide a panoramic view. The achievable monthly and yearly average cooling potential through the implementation of direct evaporative cooling, nocturnal radiative cooling and PCM based free cooling system are estimated and reported for five selected cities (Chennai, Bangalore, New Delhi, Jaipur, and Jammu and Kashmir) in India, based on their corresponding weather data. It is inferred that Jaipur with hot and dry climatic zone has the highest yearly average passive cooling potential of 251 W m⁻². In addition, concept of a novel hybrid passive system is presented along with the strategies required in the present scenario to promote passive building concepts. This comprehensive evaluation of cooling potential helps the researchers to select the appropriate passive cooling technologies based on the influencing parameters with respect to local climatic zones.

1. Introduction

India is the second most populous country in the world with 17.6% of world population and it is projected that by 2022 India will surpass China's population [1]. In India, energy consumption, economic growth, and increase in population rate have a strong interrelationship with each other [2]. India became the fourth biggest energy consumption rate has increased by 7.5% in 2014 – 2015, the highest growth rate among the developing countries [4]. India's dependency on imported fuels had increased to 38% in 2012, though it has significant domestic fossil fuel resources [5]. The energy intensive sectors in India is categorized into three main sectors namely industry, transportation, and others. Among the 'others' category, the agriculture and buildings are highly energy intensive sectors due to the agrarian nature and rapid urbanization, which leads to an unexpected hike in energy demand and con-

sumption pattern in residential and commercial buildings. The energy consumption in the buildings is due to various appliances used for human requirement and comfort. Particularly, 45% of the total energy is spent towards maintaining the thermal comfort conditions in the residential building sector as shown in Fig. 1 [6]. Not surprisingly, the greenhouse gas (GHG) emissions associated with the building sector has also increased exponentially. The Global Buildings Performance Network (GBPN) reported that if the existing Business-As-Usual (BAU) policy for buildings is followed, India could easily experience an increase in building energy consumption and GHG emissions of around 700% by 2050 compared to 2005 levels [7]. In order to reduce the rising energy demand in buildings and its associated emissions, researchers are showing more interest towards the net zero energy/emission buildings in the recent years. The increase in number of publications in recent years regarding the energy conservation in buildings through passive cooling technologies illustrate its large energy

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Abbreviations: A, Area; AFR, Air flow rate; CDD, Cooling Degree Days; CDM, Cooling Degree Months; CFC, Chlorofluorocarbons; CO_{2e}, Carbon dioxide equivalence; COP, Coefficient of performance; EC, Evaporative cooling; GHG, Greenhouse gases; GW, Gigawatt; HCFC, Hydro chlorofluorocarbons; HVAC, Heating Ventilation and Air Conditioning; IEHX, Indirect evaporative heat exchanger; kWh, Kilowatt hour; MtCO_{2e}, Million tonnes of Carbon-di-oxide Equivalence; NRC, Nocturnal radiative cooling; NRP, Nocturnal radiative panel; PCM, Phase change material; PF, Performance factor; RH, Relative humidity; TWh/yr, Terawatt hour per year; VCRS, Vapor compression refrigeration system

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Nomen	clature	b^*	Therma
,	1.1 C 1 1 1	c, inf	Convec
b	Width of the bond		panel
C _b	Sond conductance between the pipes and absorber plate	c, sup	Convec
$C_{p,v}$	Diameter		Convor
d d	Mass flow rate of evenerated water		Diroct
d _{mw}	Latent heat transfer assed by the eveneration of water	DEC dn	Direct
uQ _l	Eatent field transfer caused by the evaporation of water	up	Dew pc
E _{consumed}	Monthly average avenerative cooling potential	DFIEC	Dew pc
£	Monthly average evaporative cooling potential	e EC	Externa
1 f	Even proting appling appendix in flow	EC	Evapor Eluid +
l _{cc}	Convertive best transfer coefficient	ji ;	Incide l
		l :*	Inside I
П Ъ	Enthalpy Mass transfer coefficient		Interna
n _m	Mass transfer coefficient	IEC in*	Enthol
н _v 1-	The armole conductivity	IN." :**	Enthalp Inlat II
K	Maga flow rote	<i>III</i>	Innet H
m	Mass now rate	0	Outside
n 	Tetal answer aland amount (0 for alaan alm and 1 for	• *	ger
n _c	Total opaque cloud amount (0 for clear sky and 1 for	0	Outside
л	Overcast sky)	×	ger
P D.	Pressure Deve til som har	pcm*	Mass of
Pr	Prandti number	pw	Enthall
Q	Cooling capacity	r *	Radiati
ке с	Reynolds number	r* 	Radiate
о Т +*	Absorbed solar energy	r, uy	Radiati
1, l"	Character direction	r, sup	Kadiati
t _c	Charging duration	S*	Humid
V	Volumetric air nowrate	S*	Surface
0	Velocity of air	s, in	Coturnet
Va*	Air specific volume	sai	Saturat
v _m	Monthly average wind speed	Crook lo	ttone
V _W	Will velocity	Greek le	llers
VV 147 [*]	Distance between the pipes	٨D	Air pro
w,x	Distance from the collector inlet	Δr	Difform
У	Distance from the conector finet	$\Delta \omega$	oir
Subcomin	to	a	Convoa
Subscrip	15	u _c	Rond th
a moan	monthly mean ambient temperature	Y S	Fin Thi
1 u, meun	Entoring air dry bulb temperature	0 8 [*]	Time ()
1 1 db	Dry bulb temperature of intake air	0 A+	Time (I
1, uv 1 dn	Dry build temperature of intake air		Emiceix
1, up 1, ub	Wet hulb temperature of intake air	с °	Dorogit
1, WU 2	Leaving air dry bulb temperature	ъ 	Viscosi
$\frac{2}{2}$ db	Dry hulb temperature of supply air	μ n	Efficien
2, 00	Entoring air wet hulb temperature	۱ <u>۱</u> ۵'	Monthl
5 4	Entering wat hulb temperature of secondary air	o m	Monthl
7 5	Entering dew hulb temperature of product air	υ _m λ	Latont
J 0	Ambient temperature	Λ,	Dongite
u a in cu	Autorent temperature	μ _a σ	Stefer
u, m, 00	has temperature	0	Stelan-
υ	base temperature	ω	пиша

b^*	Thermal conductivity of the bond		
c. inf	Convective heat transfer coefficient, lower side of the		
.,.,,	panel		
c, sup	Convective heat transfer coefficient, upper side of the		
., I	panel		
conv	Convective heat transfer coefficient		
DEC	Direct evaporative cooler		
dp	Dew point temperature		
DPIEC	Dew point indirect evaporative cooler		
е	External diameter of the pipe		
EC	Evaporative cooler		
fi	Fluid temperature at radiator inlet		
i	Inside heat transfer coefficient of the PCM heat exchanger		
i*	Internal diameter of the pipe		
IEC	Indirect evaporative cooler		
in*	Enthalpy of inlet air		
in**	Inlet Humidity ratio		
0	Outside heat transfer coefficient of the PCM heat exchan-		
	ger		
0*	Outside heat transfer coefficient of the PCM heat exchan-		
-	ger		
ncm*	Mass of the PCM solidified in the spherical container		
nw	Enthalpy of water vapor		
r	Radiative panel		
r^*	Radiator emissivity		
r. inf	Radiative heat transfer coefficient, lower side of the panel		
r, sup	Radiative heat transfer coefficient, upper side of the panel		
s	Humidity ratio of the wet-bulb temperature at saturation		
<i>s</i> *	Surface area of the spherical capsule		
s, in	Inlet wet bulb temperature of the air stream		
sat	Saturation pressure		
	-		
Greek letters			
ΔP	Air pressure drop		
$\Delta \omega$	Difference between moisture content of supply & outdoor		
	air		
α_{c}	Convective heat transfer coefficient		
γ	Bond thickness		
δ	Fin Thickness		
δ	Time (hours)		
Δt	Time required to reach the 'i'th location		
3	Emissivity		
<u></u> 3	Porosity of the PCM storage tank		
μ	Viscosity of air		
η	Efficiency		
θ'm	Monthly average air wet bulb temperature		
θm	Monthly average air dry bulb temperature		
λ,	Latent heat of the PCM		
ρ_{a}	Density of air		
σ	Stefan-Boltzmann constant		
ω	Humidity ratio of air		

saving potential in the near future. It is forecasted that it is possible to reduce the GHG emissions from buildings through the implementation of effective energy conservation measures such as adoption of passive cooling technologies/green building concepts, maximizing the renewable energy harnessing and equipping the energy efficient equipment. It is also predicted that, by implementing stringent policies and adopting the various energy conservation measures, the increase in household electricity consumption (650 kWh in 2012 to 2750 kWh in 2050) could be curtailed to 1170 kWh/household by 2050 [7].

2. Climatic zones of India - An overview

India, being a tropical country comprises of versatile climatic zones which are categorized as hot-dry, warm-humid, composite, temperate/ moderate, and cold zones [8]. Continental climate with extreme summer and winter conditions prevails in the northern part of India but southern and coastal regions of the country experience the warm

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