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Research paper

Geothermal helical heat exchangers: Coupling with a reversible heat pump in western Europe



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HIGHLIGHTS

- Two exchangers model are compared: horizontal rings model and annular cylindrical conduit.
- The freezing of the subsoil is analyzed.
- The natural thermal load of the ground is studied.
- The thermal behavior in both long and short terms is investigated.
- Optimization rule is proposed to design the geothermal helical heat exchangers.

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ABSTRACT

This paper presents a computational analysis of geothermal heat pump used to assess the thermal needs of a 120 m² low-energy house. Helical heat exchangers are used to extract or inject heat into shallow subsoil. Two 2D-axisymmetric models of helical heat exchangers are compared. The frozen of the subsoil such as its thermal relaxation during the summer are analysed. The performance of the exchangers is being studied over a 10-year period. The impact of the heat-pump operating modes (short cycles vs. long cycle) on the system efficiency is assessed, and the influences of the subsoil properties and of the geometry of the geothermal exchangers are further studied. A sizing law relating the number of exchangers to be used and their height/diameter is proposed.

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1. Introduction

"Very low enthalpy" geothermal resources are part of the renewable energies that one can use for heating and cooling buildings [1,2,3]. These resources are available throughout the year and everywhere. They require a heat pump, whose market is growing in Europe, with more than a million units installed over

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the last few years. Applications include the heating and cooling of buildings, the production of domestic hot water, and even the thermal unload of solar installations in summer. Several types of geothermal exchangers exist: the most widespread are the "borehole heat exchangers" [4], but other kinds can be used instead, such as horizontal loops [5], helical heat exchangers [6,7], or energy piles [3]. Helical heat exchangers are generally cylindrical. Their reduced height and diameter (typically 2.5 m and 1 m respectively) allow the use of traditional excavator for their installation in the ground. The upper part is generally placed 1 m below the ground level in order to take advantage of the thermal potential of the subsoil and the ground heat recharging during summer [8]. Helical heat exchangers offer a compromise between borehole heat exchangers

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Nomenclature
                                                                                           annual pulse (rad s^{-1})
                                                                                Letters with special subscripts
                                                                                           convective and radiative heat transfer (W m<sup>-2</sup> K<sup>-1</sup>)
Latin letters
           thermal diffusivity (m<sup>2</sup> s<sup>-1</sup>)
                                                                                           helical length of the exchanger (m)
                                                                                L_{tot}
           specific heat (J kg<sup>-1</sup> K<sup>-1</sup>)
                                                                                           external radius of the pipe (m)
c_p
                                                                                r_e
           thermal effusivity (J K^{-1}m^{-2} s<sup>-1/2</sup>)
Е
                                                                                           internal radius of the pipe (m)
                                                                                r_i
           Fourier number (-)
                                                                                           yearly amplitude of the ground-level temperature (K)
Fo
                                                                                T_{amp}
Н
           height of the exchanger (m)
                                                                                           ambient air temperature (K)
                                                                                T_{ext}
           convective heat transfer (W m^{-2} K^{-1})
                                                                                T_{moy}
                                                                                           mean ground temperature (K)
h
           specific latent heat (J kg<sup>-1</sup>)
                                                                                           inlet temperature (K)
                                                                                T_{in}
Nu
           Nusselt number (-)
                                                                                           outlet temperature (K)
                                                                                T_{out}
P
           power (W)
                                                                                T_{\omega}
                                                                                           period of a heat pump cycle (s)
           volumetric flow rate (m<sup>3</sup> s<sup>-1</sup>)
                                                                                           coldest day (s)
q_{\nu}
                                                                                t_c
R
           radius of the exchanger (m)
                                                                                t_{ini}
                                                                                           initial time (s)
           thermal resistance (K W<sup>-1</sup>)
R_{th}
           radius (m)
                                                                                Subscripts
Т
           temperature (K)
                                                                                           heat transfer fluid
t
           date (s)
                                                                                flow
                                                                                           flow
                                                                                           solid water
           depth (m)
7
                                                                                ice
                                                                                lig.wat
                                                                                           liquid water
                                                                                            "dry matrix" of the underground
Greek letters
                                                                                mat
           auxiliary coefficient (-)
                                                                                mix
                                                                                           multi states
β
           ratio of water at liquid state (-)
                                                                                pipe
                                                                                           wall of the pipe
η
           volume fraction of water in the underground (-)
                                                                                           soil (next to the exchanger)
K
           thermal conductivity (W m<sup>-1</sup> K<sup>-1</sup>)
λ
                                                                                           infinite
                                                                                \alpha
           mass density (kg m<sup>-3</sup>)
ρ
           damping(-)
                                                                                Accentuation
υ
           heat flux density (W m<sup>-2</sup>)
                                                                                           undisturbed by the exchanger
φ
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and horizontal loops: they do not require such a large area as horizontal loops and are claimed to be cost effective. Furthermore, their operation has no impact on the building structure as it may sometime be the case with energy piles. Helical heat exchangers appeared on the market the last few years but the number of studies related to their operation is quite limited. This paper is a contribution to the study of such new compact exchangers.

The underground is usually described in a 2D axisymmetric way, considering the underground as a semi-infinite solid, spatially homogeneous, with thermal transfers occurring only by conduction. Its yearly temperature matches the heat equation, with an upper boundary condition corresponding to a sinusoidal variation of the upper temperature with a 1-year period. The problem of heat transfer can be solved with an equivalent electrical circuit of thermal resistances and capacitances [8]. This modeling usually doesn't consider the freezing of the subsoil that can appear during operation, as presented e.g. by Ref. [9]. However, the freezing of the subsoil locally impacts the physical properties of the subsoil and is accompanied by a release of energy, which mainly depends on the water content of the underground as shown in Ref. [10]. In this paper, the underground is considered as a homogeneous porous medium – the pores being full of water – with no coupling between moisture transfer and thermal conditions. Moreover, the phase change does not produce any volume change, and the thermal properties of the soil matrix are constant. Three regions are defined (freezing soil region, unfreezing soil region, and two-phase region), with moving interfaces where the phase changes occur. Actually, one can consider that the phase change occurs progressively in the two-phase region. This effect can be taken into account by defining an equivalent heat capacity of the underground in the heat equation, solved with a finite-element method. A more detailed description of the model is available in Ref. [6] and will be reminded in Section 2.

Different models are available to predict the performances of geothermal helical heat exchangers. The simplest model involves linear power source. This model does not take into account the soil inside the exchangers, and is not adapted to predict short duration heat solicitations due to the heat pump [11]. 3D models are too time consuming to enable simulations over long time period – typically annual performances of the exchangers over 1 year [8]. 2D axisymmetric such as horizontal rings model or annular cylindrical conduit model offer a good compromise between calculation duration and accuracy of simulations. They have been used to study heat storage in arid zones [12,13] or the heating and cooling of buildings taking into account the freezing of the subsoil [8]. Ref. [6] points out that the "annular cylindrical conduit" was easier to use and quite reliable. Moreover, thermal results taking into account the freezing of ground water appeared satisfactory. However, several questions remained. This paper investigates these points, using the models described in Ref. [6]. The thermal needs of a low-energy building over a year are used as boundary conditions, and the results from the "horizontal rings" model are compared in part 3.1.

2. 2D-axisymmetric models

2.1. Underground

The underground is considered as a semi-infinite solid, spatially homogeneous, with thermal transfers occurring by conduction. Its yearly temperature matches the heat equation, with an upper boundary condition corresponding to a sinusoidal variation of the upper temperature with a 1-year period:

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