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# Energy retrofitting of residential buildings—How to couple Combined Heat and Power (CHP) and Heat Pump (HP) for thermal management and off-design operation



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#### ABSTRACT

Cogeneration (CHP) and Heat Pump (HP) are playing a key role in energy systems due to their high efficiency, especially, in energy refurbishment of buildings and industrial processes. This paper explored the opportunity to couple those two well-established technologies for heating purposes. Basically, the coupling entails limitations and constraints consisting of the mismatch between the power sizes of merchandised machines as well as the deriving technical issues. The main aim of this paper is to analyse the coupling procedure to help HVAC specialists in their job. This paper deals with a simple model formulation, based on the First Law of thermodynamics, for CHP/HP systems rating and off-design operations. Indeed, an analytical method was designed along with the definition of a size factor. Then, a graphical method for coupling CHP and HP was presented so as to be an expeditious tool in designing phase. Finally, two numerical applications were illustrated highlighting the energy performance gains and the high level of operation flexibility.

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## 1. Introduction

Nowadays, the need for efficient conversion technologies promotes the deployment of combined heat and power (CHP) production and heat pump (HP), accentuating their role in the energy systems. In terms of the hydraulics, these units are installed just like any conventional boiler to provide heat to the energy system, while, in the CHP case, there is the opportunity to generate electrical energy too. Such technologies can foster the diffusion of distributed generation systems, taking into account the conformation of the power grids [1–3].

As regards the CHPs, they have problems to meet efficiently the dynamic thermal energy demand depending on their technology. So, they are usually provided by a buffer system, e.g. a water tank [4]. In that behaviour, the CHP works at rated load so that to have a higher efficiency. At the same time, the electricity surplus should be sold to the grid, whether a net metering option is allowed, otherwise it should be stored locally. Selling to the grid could be

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uneconomical when the electricity price on the spot market is lower than the CHP generation cost, while it could be economically feasible if associated to an electric storage [5]. Many technologies, so-called Power-To-Power solutions, are already tested in this combination. Among those ones, Compressed Air Energy Storage (CAES) shows promising performance even hybridized with Phase Change Material [43] or coupling renewable generation with fossil-fuel one by means of synthetic fuel technology [44].

Furthermore, the CHP designed to meet the thermal energy demand implies to account for the matching between the CHP and end-user power to heat ratio as much as possible in order to increase the plant profitability.

Referring to HPs, they are a promising option to reduce the energy-related greenhouse gases emissions in the building and industrial sectors thanks to the use of freely available heat such as ambient air, water, ground and thermal cascade as well. In the building sector, the electrically driven air—water compression heat pumps (AWHP) are the most common technology for retrofits owing to their relatively low investment cost, easy installation and little required space [6]. Its drawback consists of a reduced efficiency as well as lower thermal output caused by low source medium temperatures and larger required temperature difference during the coldest period of the year. In the industrial sector, the choice of refrigerant plays a key role. Indeed, thermodynamic

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### Nomenclature $CO_2$ Carbon dioxide $C_{E,f}$

CHP thermal generation cost [€/kWh]

 $C_{svs}$ System capacity factor

System capacity factor minimum value  $C_{sys,min}$  $C_{TOT}(E_{in})$  Cost related to the overall inlet energy [ $\in$ ]

CHP capacity factor  $C_{f,CHP}$ Capacity factor  $C_{f,HP}$ 

Water specific heat [k]/kg °C]  $c_{\text{w}}$ E134 Bis (difluoromethyl) ether Electrical energy from CHP [kWh]  $E_{el.CHP}$ 

Total thermal energy from combined system [kWh]  $E_{H.out}$ 

Size factor  $f_s$ 

 $\mathbf{f}_{s}$ Size factor in off-design operation

Power to heat ratio  $I_{el}$  $K_1$ Correction factor no.1

 $K_{1,p}$ Correction factor accounting for primary energy

 $K_2$ Correction factor no.2 Correction factor no.3  $K_3$ 

Correction factor accounting for electricity over- $K_{el}$ 

production

 $K_f$ Efficiency correction factor Water mass flow rate [kg/s]  $\dot{m}_w$  $\dot{m}_{pollutant}$  Pollutant mass flow rate [kg/s] Pel,aux. Auxiliary electrical power [kW] CHP electrical power [kW]  $P_{el,CHP}$ Electrical power excess [kW] P<sub>el,exc</sub>. HP electrical power [kW]  $P_{el,HP}$  $P_{\text{fuel}}$ CHP fuel consumption [kW<sub>t</sub>] CHP thermal power [kW<sub>t</sub>] P<sub>heat.CHP</sub> HP thermal power [kW<sub>t</sub>]

Heat sink thermal power  $[kW_t]$  $P_{Sink}$ HP thermal power [kWt] P<sub>heat.HP</sub> Auxiliary electrical power in off-design operation

P'<sub>el,aux</sub> (when the HP results oversized relative to CHP) [kW] Electrical power excess in off-design operation [kW]

 $P'_{el,exc}$  $P'_{el,HP}$ HP electrical power in off-design operation [kW]

P'<sub>HP</sub> HP thermal power at partial load [kW<sub>t</sub>] P'heat,CHP CHP thermal power at partial load [kWt]

Fuel flow rate [Nm<sup>3</sup>/s]  $q_{\text{fuel}}$ Chlorodifluoromethane R22 R114 1,2-dichlorotetrafluoroethane R141b 1,1-dichloro-1-fluoroethane 1,1,2-trifluoroethane R143

R236ea 1,1,1,2,3,3-hexafluoropropane R236fa 1,1,1,3,3,3-hexafluoropropane

R744 Carbon dioxide

 $T_{amb}$ Outdoor temperature [°C] Exhaust gas inlet temperature [°C]  $T_{exh,in}$  $T_{exh,out}$ Exhaust gas outlet temperature [°C]

Heat exchanger water inlet temperature [°C]  $T_{in.HX}$ Water outlet temperature for end-user supplying T<sub>out.HX</sub>

[°C]

Cold sink temperature for HP [°C]  $T_{sink}$ 

Water temperature difference due to the CHP con- $\Delta T_{SH}$ 

tribution for the end-user [°C]

 $\Delta T_{w,HP}$ Water temperature difference due to the HP [°C]

#### **Abbreviations**

Authority for electricity and natural gas (in italian) **AEEG** 

**AWHP** Air-Water compression heat pump

CHP Combined heat and power **CHPR** Crossed heat to power ratio

CHPR' Crossed heat to power ratio in off-design operation

COP Coefficient of performance

EU European union

**GEHP** Gas engine-driven heat pump **GSHP** Ground source heat pump

HP Heat pump

**HPGHP** Hybrid-Power gas engine-driven heat pump HVAC Heating, ventilating and air conditioning

**ICE** Internal combustion engine LHV Lower heating value [M]/Nm<sup>3</sup>]

NG Natural gas

ORC Organic rankine cycle PV/T Photovoltaics/Thermal RES Renewable energy sources TOE Tons of oil equivalent  $TPF_{HP}$ HP thermal power fraction TPF<sub>CHP</sub> CHP thermal power fraction

#### Greek symbols

Liquid-to-gas heat exchanger effectiveness

CHP first law efficiency  $\eta_{I,CHP}$ 

First law efficiency related to the whole combined  $\eta_{I,svs}$ 

CHP/HP plant

Corrected first law efficiency related to the whole  $\eta_{I,sys(act)}$ 

combined CHP/HP plant CHP electrical efficiency  $\eta_{el}$ 

Equivalent electrical system efficiency  $\eta_{el.svs}$ 

Power grid efficiency  $\eta_{Grid}$ 

Reference heat generator conversion efficiency  $\eta_{g,ref}$ Combined CHP/HP system efficiency for heating  $\eta_{H,svs}$ Average combined CHP/HP system efficiency for  $\eta_{H,sys(av)}$ 

heating

 $\eta_{H,sys(act)}$  Actual combined CHP/HP system efficiency for

heating

CHP heat recovery efficiency  $\eta_{hr}$ 

Relative efficiency between distributed generation  $\eta_{REL}$ 

system and power grid

 $\eta'_{I,sys(act)}$  Actual first law efficiency related to the whole com-

bined CHP/HP plant in off-design operation

 $\eta \dot{\,\,}_{H,sys(act)}$  Actual combined CHP/HP system efficiency for heating in off-design operation

properties, safety and transport issues promoted the use of refrigerants such as R744 (Carbon Dioxide), which could be used as coolant or in thermal cascade system [7,8]. This latter provides the opportunity to integrate large scale Heat Pump in recovering waste heat from existing thermal cycles.

Having said, the HPs represent a more efficient solution owing to their COP but they work at lower input-output temperature. As matter of fact, the recent deployment of Heat Pumps in replacing conventional systems such as boilers has raised many issues linked to the new adjustments required to the local electricity distributors. Indeed, the HP is an adjunct electrical demand which entails the upgrade of the end-users' electricity meter along with further transmission costs. For instance, in Italy the Authority for Electricity and Natural Gas (AEEG) imposes the purchase of a dedicated electricity meter for the HP installation [9]. So, considering the advantages of both CHP and HP technologies, a foreseeable solution could be their coupling to increase the overall energy efficiency for heating purposes as well as to reduce the associated  $CO_2$  emissions.

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