



# Forced convection in a parallel-plate channel occupied by a nanofluid or a porous medium saturated by a nanofluid



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## ABSTRACT

An analytical study is made of fully-developed laminar forced convection in a parallel-plate channel occupied by a nanofluid or by a porous medium saturated by a nanofluid, subject to uniform-flux boundary conditions. A model incorporating the effects of Brownian motion and thermophoresis is adopted. (Previous analytical studies using this model have been concerned with natural convection.) It is found that the combined effect of these two agencies is to reduce the Nusselt number.

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## 1. Introduction

In recent years a large number of analytical studies of natural convection in nanofluids (suspensions of particles whose diameters are tens or hundreds nanometres) have been published, either in a fluid otherwise clear of solid material or in a porous medium, but the corresponding literature on forced convection is more sparse. In the case of a porous medium we are aware of just one published study, namely that by Maghrebi et al. [1], and that study was a numerical one. These authors investigated thermally developing forced convection in a parallel-plate channel.

For a fluid clear of solid material there have been several numerical as well as experimental investigations, and these have been reviewed by Dalkilic et al. [2]. A recent one is that by Rossi di Schio et al. [3]. However, as far as we know there has been just a single analytical published paper, namely that by Hung [4], who studied flow in microchannels. He considered just the variation of thermal conductivity, viscosity and heat capacity. On the other hand, Maghrebi et al. [1] employed the Buongiorno [5] model which incorporates the effects of Brownian motion and thermophoresis.

In the present paper we present an analytical study of fully-developed forced convection, using the Buongiorno model, in a parallel-plane channel with uniform heat flux on the boundaries. In the numerical work of Rossi di Schio et al. [3] the Buongiorno

model was used, but these authors assumed streamwise variable temperature boundary conditions rather than constant flux ones.

## 2. Analysis

In the analysis that follows we use asterisks to denote variables that have dimensions. We chose Cartesian coordinates such that the boundary plates are at  $z^* = \pm H$ , so that  $H$  is the channel half-width. Then for the case of a nanofluid otherwise clear of solid material (for brevity we will use the phrase “clear fluid”), the velocity  $\mathbf{v}^*$  has components  $(U, 0, 0)$ , where  $U$  is given by

$$U = \frac{3}{2} U_m (1 - z^2), \quad (1)$$

where  $U_m$  is the mean velocity. This is the well-known expression for plane-Poiseuille flow. For the case of a Darcy porous medium, we have slug flow, and

$$U = U_m. \quad (2)$$

We denote the temperature by  $T^*$  and the volumetric nanoparticle fraction by  $\phi^*$ . At the boundaries we suppose that the heat flux into the porous medium or clear nanofluid is  $q''$ , a constant quantity. Since the flow is thermally developed, the first law of thermodynamics requires that

$$\frac{\partial T^*}{\partial x^*} = \frac{q''}{(\rho c)_f H U_m}. \quad (3)$$

Here  $(\rho c)_f$  is the heat capacity of the nanofluid. The thermal energy and nanoparticle conservation equations are taken to be

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**Nomenclature**

$c$	nanofluid specific heat at constant pressure	$(x^*, y^*, z^*)$	Cartesian coordinates
$D_B$	Brownian diffusion coefficient	<b>Greek symbols</b>	
$D_T$	thermophoretic diffusion coefficient	$\alpha_N$	thermal diffusivity of the nanofluid, $\frac{k_N}{(\rho c_p)_f}$
$H$	half channel width	$\varepsilon$	porosity of the porous medium (set as unity for a clear fluid)
$k_m$	thermal conductivity of the porous medium	$\mu$	viscosity of the fluid
$k_N$	effective thermal conductivity of the nanofluid or porous medium	$\rho$	fluid density
$Le$	Lewis number, defined in Eq. (8)	$(\rho c)_f$	heat capacity of the nanofluid
$N_A$	modified diffusivity ratio, defined in Eq. (8)	$(\rho c)_N$	effective heat capacity of the nanofluid or porous medium
$N_B$	modified heat capacity ratio, defined in Eq. (8)	$(\rho c)_p$	heat capacity of the nanoparticles
$Pe$	Péclet number, defined in Eq. (12)	$\rho_p$	nanoparticle mass density
$q''$	wall heat flux	$\sigma$	heat capacity ratio, defined in Eq. (7)
$t^*$	time	$\phi^*$	nanoparticle volume fraction
$t$	dimensionless time, $t^* \alpha_m / \sigma H^2$	$\phi_0^*$	reference value for the nanoparticle volume fraction
$T^*$	nanofluid temperature	$\phi$	relative nanoparticle volume fraction, $\frac{\phi^* - \phi_0^*}{\phi_0^*}$
$T$	dimensionless temperature	$\Phi$	function giving z-dependence of $\phi$
$T_0$	representative temperature, chosen to be the wall temperature	$\theta$	dimensionless temperature
$(u, v, w)$	dimensionless Darcy velocity components, $(u^*, v^*, w^*)H / \alpha_m$	$\Theta$	function giving z-dependence of $\theta$
$U_m$	mean flow velocity	<b>Superscript</b>	
$\mathbf{v}$	fluid velocity	$*$	dimensional variable
$\mathbf{v}^*$	dimensional fluid velocity, $(u^*, v^*, w^*)$		
$(x, y, z)$	dimensionless Cartesian coordinates, defined in Eq. (6)		

$$(\rho c)_N \frac{\partial T^*}{\partial t^*} + (\rho c)_f \mathbf{v}^* \cdot \nabla^* T^* = k_m \nabla^{*2} T^* + \varepsilon (\rho c)_p \left[ D_B \nabla^* \phi^* \cdot \nabla^* T^* + \left( \frac{D_T}{T_0} \right) \nabla^* T^* \cdot \nabla^* T^* \right], \quad (4)$$

$$\frac{\partial \phi^*}{\partial t^*} + \frac{1}{\varepsilon} \mathbf{v}^* \cdot \nabla^* T^* = D_B \nabla^{*2} \phi^* + \left( \frac{D_T}{T_0} \right) \nabla^{*2} T^*. \quad (5)$$

Here  $t^*$  is the time,  $(\rho c)_N$  is the heat capacity of the porous medium or clear nanofluid,  $k_m$  is the thermal conductivity of the porous medium,  $\varepsilon$  is the porosity of the porous medium (set as unity for a clear fluid),  $(\rho c)_p$  is the heat capacity of the nanoparticles,  $D_B$  and  $D_T$  are the Brownian motion and thermophoresis coefficients, respectively, and  $T_0$  is a representative temperature which we can choose, in the absence of any better alternative, to be the wall temperature, even though in the case of uniform flux boundary conditions this varies with the axial coordinate. We are following the classical treatment of forced convection given in textbooks such as Bejan [6].

We introduce dimensionless variables as follows. We define

$$(x, y, z) = (x^*, y^*, z^*) / H, \quad t = t^* \alpha_m / \sigma H^2, \quad (u, v, w) = (u^*, v^*, w^*) H / \alpha_m, \quad \phi = (\phi^* - \phi_0^*) / \phi_0^*, \quad T = (T^* - T_0) k_m / H q'', \quad (6)$$

where  $\phi_0^*$  is a reference scale for the nanoparticle fraction and

$$\alpha_N = \frac{k_N}{(\rho c_p)_f}, \quad \sigma = \frac{(\rho c_p)_N}{(\rho c_p)_f}. \quad (7)$$

We also introduce the dimensionless parameters  $Le$ ,  $N_A$  and  $N_B$  defined by

$$Le = \frac{\alpha_N}{D_B}, \quad N_A = \frac{D_T}{D_B T_0}, \quad N_B = \frac{\varepsilon (\rho c_p)_p \phi_0^*}{(\rho c)_f}. \quad (8)$$

Then Eqs. (4) and (5) become

$$\frac{\partial T}{\partial t} + \mathbf{v} \cdot \nabla T = \nabla^2 T + \frac{N_B}{Le} \nabla \phi \cdot \nabla T + \frac{N_A N_B}{Le} \nabla T \cdot \nabla T, \quad (9)$$

$$\frac{1}{\sigma} \frac{\partial \phi}{\partial t} + \frac{1}{\varepsilon} \mathbf{v} \cdot \nabla \phi = \frac{1}{Le} \nabla^2 \phi + \frac{N_A}{Le} \nabla^2 T. \quad (10)$$

The parameter  $Le$  is a Lewis number,  $N_A$  is a modified nanofluid diffusivity ratio and  $N_B$  is a modified nanofluid heat capacity ratio.

Eq. (3) becomes

$$\frac{\partial T}{\partial x} = \frac{1}{Pe}, \quad (11)$$

where  $Pe$  is a Péclet number defined by

$$Pe = \frac{H U_m}{\alpha_N}. \quad (12)$$

In this notation,

$$u = Pe f(z), \quad (13)$$

where

$$f(z) = \begin{cases} 1 & \text{for a porous medium,} \\ \frac{3}{2}(1 - z^2) & \text{for a clear fluid.} \end{cases} \quad (14)$$

Now Eqs. (9) and (10) become, for a steady-state situation,

$$\nabla^2 T + \frac{N_B}{Le} \nabla \phi \cdot \nabla T + \frac{N_A N_B}{Le} \nabla T \cdot \nabla T = f(z), \quad (15)$$

$$\nabla^2 \phi + \frac{N_A}{Le} \nabla^2 T - Le Pe f(z) \frac{\partial \phi}{\partial x} = 0. \quad (16)$$

We need to solve Eqs. (15) and (16) subject to the symmetry conditions

$$\frac{\partial T}{\partial z} = 0, \quad \frac{\partial \phi}{\partial z} = 0 \text{ at } z = 0, \quad (17a, b)$$

and the boundary conditions

$$T = \frac{x}{Pe}, \quad \frac{\partial \phi}{\partial z} + N_A \frac{\partial T}{\partial z} = 0 \text{ at } z = 1. \quad (18a, b)$$

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