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Numerical modelling of oxy fuel combustion, the effect of radiative and convective heat transfer and burnout



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HIGHLIGHTS

- CFD model is developed to investigate the oxy-fuel combustion in furnace.
- Radiative and convective heat flux is significantly manipulated by the RR.
- A working range for air to oxy-firing is determined showing unique balance.
- Air equivalent radiative heat flux can be obtained at a RR of \approx 71%.
- Unburned carbon in ash (CIA) is improved in oxy-fuel conditions.

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ABSTRACT

In this study, a numerical investigation on the radiative and convective performance of the combustion of pulverised Russian coal having higher Calorific value in a 0.5 MWth combustion test facility (CTF) has been conducted for air and CO₂-rich oxy-fired environment considering an IFRF Aerodynamically Air Staged Burner (AASB) configured with the furnace. This study is carried out considering a finite volume method (FVM) tool using AVL Fire version 2009.2 coupled with the user defined subroutines especially for the coal devolatilisation and char combustion modelling. This code is validated comparing the experimental radiative heat flux with the numerically predicted data and results show that reasonable agreement has been found with the measured radiative heat flux on the furnace wall. Different combustion environments were investigated including an air-fired as reference case and three recycled flue gas (RFG) fired combustion environments. The different cases are composed of varying recycled ratio (RR) between 65% and 75%. It was found that the flame temperature distribution for the reference case (air fired) and RR72% case were found to be similar. The flame temperature increased with O₂ concentration and decreasing RR. With the decrease of RR, the length of the flame is also shortened. The ignition condition improved with enriched O₂ concentration in the RR65% (O₂ 30.9%, CO₂ 69.2% by mass) case. The results show that radiative and convective heat flux is significantly manipulated by the RR. The position of peak radiative flux moves downstream with increasing RR due to increased mass flow rate and reduced O₂ at higher RR. With the increase of normalised total flow (NTF), mean flame temperature and exit temperature decreased whereas with the increase of normalised O₂ flow (NOF), mean flame temperature and exit temperature increase. The presented working range for the Russian coal, suggests that the air equivalent radiative heat flux can be obtained at a RR of \approx 71% while air equivalent flame temperature were observed at RR of 72%. Reasonable agreement has been obtained for unburned carbon in ash (CIA). An improved burnout was observed in RR conditions.

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1. Introduction

Combustion of fossil fuel is a proven thermochemical conversion technology for heat and power production which is connected with formation of pollutant, also a source of deterioration of the global environment. Coal is a major source of fossil fuel, responsible for generating electrical energy in the world [1]. Also, combustion of coal results in the emission of greenhouse gases, the accumulation of which in the atmosphere since the start of the industrial revolution has been contributing to climate change [2]. In this concern, international efforts, like Kyoto protocol bound







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Nomenclature

$A_c A_{\nu}$	pre-exponential factor (s^{-1}) pre-exponential factor (s^{-1})	$rac{x_i}{ar{y}_{pr}}$	spatial distance in the <i>i</i> th direction mass fraction of product (kg/kg)
A_p	cross sectional area of particle (m^2)	\bar{y}_{ox}	mass fraction of oxidizer (kg/kg)
C_D	drag coefficient	\bar{y}_{fu}	mass fraction of fuel (kg/kg)
C_{fu}	combustion model constant, (= 3.0)		
C_{pr}	combustion model constant, (=0.5)	Greek sy	vmbols
Dp	diameter of the particle (μ m)	μ, Ĵ	turbulent viscosity (m ² /s)
E _c	activation energy (kJ mol ^{-1})	e	particle emissivity
\underline{E}_{ν}	activation energy (kJ mol ⁻¹)	τ_R	turbulent time scale (s^{-1})
<u>F</u> idr	drag force (N)	i _b	blackbody emissivity
<u>F</u> ig	gravitational force (N)	Φ	variables
F react	reaction force (N)	λ	thermal conductivity (W/m K)
G	production due to the buoyancy force	í	radiation intensity (W/m ²)
Κ	turbulent kinetic energy (m²/s²)	i'_{n+1}	total radiation intensity (W/m^2)
K_{ν}	rate constant of devolatilisation	a_i	absorption coefficient
m_{vp}	mass of particles at devolatilisation (kg)	ρ	gas density (kg/m ³)
$m_{\rm RFG}$	amount of recycled flue gas	, 3	turbulent dissipation rate (m ² /s ³)
$m_{\rm PFG}$	amount of product flue gas	ρ_{σ}	density at gas phase (kg/m^3)
m_p	mass of the particle (kg)	ρ_n	density at particle phase (kg/m ³)
min	minimum value of the operator	Γ	diffusion coefficient of variable Φ
Р	pressure (N/m ²)	σ	Stephan–Boltzmann constant $(W/m^2 K^4)$
P_A	atmospheric Pressure (N/m ²)		
Pr	Prandtl number	List of a	bhreviations
р	sum of partial pressure (N/m ²)	IFRF	International Flame Research Foundation
P_g	partial pressure of oxygen at furnace (N/m²)	AASB	Aerodynamically Air Staged Burner
P_s	oxygen partial pressure at surface (N/m ²)	DTRM	discrete transfer radiation method
Rep	particle Reynolds number	EBU	Eddy Breakup combustion model
R_p	radius of the particle (µm)	RTE	radiation transfer equation
\dot{r}_{fu}	fuel consumption rate (kg/m³ s)	CTF	combustion test facility
S	path length	FVM	finite volume method
S_{Φ}	variable source, Φ	GCV	gross calorific value
$S_{p\Phi}$	additional source term	NTF	normalised total flow
S	stoichiometric air/fuel ratio	NOF	normalised O ₂ flow
T	temperature (K)	RFG	recycled flue gas
T_p	temperature of the particle (K)	PFG	product flue gas
T_g	gas temperature (K)	GHG	greenhouse gas
U_i	velocity in the <i>i</i> th direction (m/s)	VM	volatile matter
V_p	velocity of the particle (m/s)	RR	recycled ratio
$ u_{rel} $	relative velocity (m/s)	HC	hydro carbon
V	product of volatiles at time, t	FC	fixed carbon
V_f	ultimate product of volatiles		

an obligation for the reduction of emissions specially CO₂ from industrialized countries. For the reduction of CO₂ emissions from coal fired power plants, a number of CO₂ capture technologies, can be implemented for continuing the use of fossil fuel [2–5] such as post-combustion capture, pre-combustion capture, and oxycombustion. Apart from these processes, different separation techniques are also suggested as gas phase separation, absorption, adsorption and hybrid processes. Recent developments in the CO₂ capture technologies include some innovative concepts (i.e., calcium looping, chemical looping, and amines scrubbing systems) suggested in literature. Calcium looping (CaL), a post combustion capture technology, is suggested as a competitive concept for CO₂ reduction technique [6] for power plants. CaL is achieved by oxy-firing for the generation of CO₂ rich flue gas which is absorbed by CaO and CaCO₃ in carbonator. But this technique is relatively more energy consuming. The ECO-Scrub combustion concept [7], a combination of partial oxy-fuel combustion and post combustion capture, provide comparatively higher efficiency than only post combustion process. Another important concept is chemical looping combustion (CLC) [8–12]. In CLC, high purity and concentrated

 CO_2 stream is produced as there is no contact between the fuel and the air as metal oxide works as transportation media for O_2 . Hence, lower energy is required compared to other CO_2 separation processes.

Among all the carbon capture technologies, oxy-fuel combustion is a greenhouse gases (GHG) abatement technology in which coal is burned using a mixture of O₂ and recycled flue gas (RFG), to obtain a rich stream of CO₂ ready for sequestration [4]. This carbon capture technology has been considered as one of the most effective technologies to reduce gaseous emissions. In general, the conventional boiler use air as an oxidizer in the combustion process, hence nitrogen is the main component in the flue gas as its concentration in the air is approximately 79% by volume. But, in the oxy fuel combustion, air is replaced with oxygen and recycled flue gas mixture. The flue gas produced in oxy fuel combustion has very high concentration of CO₂ comparing with the flue gas achieved in the air combustion and is ready for sequestration. The basic schematic of oxy fuel combustion with the RFG is presented in Fig. 1. Recycled ratio (RR) is the most significant parameter in an oxy-fuel system when operated with RFG. It is a function Download English Version:

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