

# Modeling, simulation and performance analysis of parabolic trough solar collectors: A comprehensive review

İbrahim Halil Yılmaz<sup>a,\*</sup>, Aggrey Mwesigye<sup>b,\*</sup>

<sup>a</sup> Department of Automotive Engineering, Adana Science and Technology University, Adana, Turkey

<sup>b</sup> Department of Mechanical and Industrial Engineering, Ryerson University, Toronto, Canada



## HIGHLIGHTS

- Current and past studies on modeling of parabolic trough collector are presented.
- Optical and thermal models and the modeling approaches are discussed in detail.
- Analytic and ray-tracing approaches are investigated for optical modeling.
- Steady and transient heat transfer conditions are examined for thermal modeling.
- Novel, passive and nanofluid techniques are outlined in performance enhancement.

## ARTICLE INFO

### Keywords:

Parabolic trough collector  
Optical modeling  
Thermal modeling  
Performance enhancement  
Computational fluid dynamics

## ABSTRACT

Solar thermal systems are advantageous since it is easier to store heat than electricity on a large scale. As such, concentrated solar power is receiving considerable interest among researchers, developers and governments. Several concentrated solar power technologies have been developed including the solar tower, the parabolic trough technology, solar dish and linear Fresnel systems. Among them, the parabolic trough solar collector is a proven technology used dominantly for both industrial process heat and power generation. This technology has matured over the years, and its advancement has become the topic of numerous research studies which were the counter driving force of the field. Particularly in recent years, a significant amount of theoretical and numerical studies have been conducted to assess and improve the performance of parabolic trough solar collectors. This review methodologically holds colossal knowledge of current and past studies to assess the optical and thermal performances of parabolic trough solar collectors, modeling approaches and the potential improvements proposed on behalf of the parabolic trough solar collector design. The optical modeling approaches are identified to be analytical and ray-tracing. The review of thermal modeling approaches presents the steady and transient heat transfer analyses of single and two-phase (with direct steam generation) flows. Also, the computational fluid dynamics models used to analyze the physics of parabolic trough solar collectors with a better insight are reviewed and presented. Finally, the studies conducted on the performance improvement of parabolic trough solar collectors are separately examined and presented, these include novel designs, passive heat transfer enhancement, and nanoparticle laden flows.

## 1. Introduction

Solar energy is the world's most abundant source of energy, it has been shown to have significant potential to meet a considerable portion of the world's energy demand [1,2]. With  $1.7 \times 10^{14}$  kW of the sun's energy received by the earth surface, only 84 min of solar radiation was estimated to give 900 EJ which was equivalent to the world's energy demand for 2009 [1]. However, significant research and development

efforts are still needed to overcome challenges associated with harnessing this resource [3]. These include developing efficient technologies for harvesting, cost effective and efficient energy storage options, optimization of hybrid energy systems working with solar energy and another renewable energy resource.

Concentrating solar power (CSP) is an emerging technology and offers significant advantages such as built-in storage capability, high economic returns and reduced greenhouse gas emissions. The life-cycle

\* Corresponding authors.

E-mail addresses: [iyilmaz@adanabtu.edu.tr](mailto:iyilmaz@adanabtu.edu.tr) (İ.H. Yılmaz), [Aggrey.Mwesigye@ryerson.ca](mailto:Aggrey.Mwesigye@ryerson.ca) (A. Mwesigye).

<sup>1</sup> Both authors contributed equally.

| Nomenclature         |   | Subscripts           |                                    |
|----------------------|---|----------------------|------------------------------------|
| $A$                  | area, $m^2$   | $a$                  | ambient air; absorber              |
| $c, c_p$             | specific heat, $J/kg\ ^\circ C$                         | $ae$                 | absorber envelope                  |
| $C$                  | concentration ratio                                     | $conv$               | convection                         |
| $D$                  | diameter, m; absorber's outer diameter, m               | $e$                  | envelope                           |
| $E$                  | total energy per unit mass, J                           | $f$                  | fluid                              |
| $f$                  | focal length, m   | $Gauss$              | Gaussian distribution              |
| $h$                  | convection heat transfer coefficient, $W/m^2\ ^\circ C$ | $htf$                | heat transfer fluid                |
| $I_b$                | beam radiation, $W/m^2$                                 | $ia$                 | inside of absorber                 |
| $k$                  | thermal conductivity, $W/m\ ^\circ C$                   | $ie$                 | inside of envelope                 |
| $k_{eff}$            | effective thermal conductivity, $W/m\ ^\circ C$         | $nf$                 | nanofluid                          |
| Kn                   | Knudsen number  | $oa$                 | outside of absorber                |
| $l$                  | collector length, m                                     | $oe$                 | outside of envelope                |
| Nu                   | Nusselt number  | $p$                  | particle                           |
| Pr                   | Prandtl number  | $rad$                | radiation                          |
| $Q$                  | heat transfer, W  | $s$                  | support                            |
| $q''$                | heat flux, $W/m^2$                                      | $sky$                | sky                                |
| Ra                   | Rayleigh number   | $tot$                | total                              |
| Re                   | Reynolds number   |                      |                                    |
| $r$                  | local mirror radius, m                                  |                      |                                    |
| $r_r$                | rim radius, m   |                      |                                    |
| $t$                  | time, s   |                      |                                    |
| $T$                  | temperature, $^\circ C$                                 |                      |                                    |
| $U_L$                | thermal loss coefficient, $W/m^2\ ^\circ C$             |                      |                                    |
| $w_a$                | aperture width, m                                       |                      |                                    |
| $x, y$               | cartesian coordinates                                   |                      |                                    |
| <b>Greek symbols</b> |   | <b>Abbreviations</b> |                                    |
| $\alpha$             | absorptivity  | 1-D                  | one-dimensional                    |
| $\Gamma$             | end-loss factor   | 2-D                  | two-dimensional                    |
| $\gamma$             | intercept factor; surface azimuth angle, $^\circ$       | 3-D                  | three-dimensional                  |
| $\delta$             | declination angle, $^\circ$                             | CFD                  | computational fluid dynamics       |
| $\varepsilon$        | emissivity  | CSP                  | concentrating solar power          |
| $\eta_o$             | optical efficiency                                      | DARS                 | direct absorption receiver system  |
| $\theta$             | incidence angle, $^\circ$                               | DSG                  | direct steam generation            |
| $\mu$                | dynamic viscosity, $kg/m\ s$                            | FDM                  | finite difference method           |
| $\rho$               | reflectivity; density, $kg/m^3$                         | FEM                  | finite element method              |
| $\sigma$             | optical error, mrad                                     | FVM                  | finite volume method               |
| $\tau$               | transmissivity  | HCE                  | heat collector element             |
| $\vec{v}$            | velocity vector   | HTF                  | heat transfer fluid                |
| $\varphi$            | local rim angle, $^\circ$                               | IPH                  | industrial process heat            |
| $\phi_r$             | rim angle, $^\circ$                                     | LCR                  | local concentration ratio          |
| $\varphi$            | particle concentration, %                               | MCRT                 | Monte Carlo ray tracing            |
| $\omega$             | hour angle, $^\circ$                                    | NCPTSC               | nanofluid-based concentrating PTSC |
|                      |   | PEC                  | performance evaluation criterion   |
|                      |   | PSA                  | Plataforma Solar de Almería        |
|                      |   | PTSC                 | parabolic trough solar collector   |
|                      |   | PV                   | photovoltaic                       |
|                      |   | SEGS                 | solar energy generating systems    |
|                      |   | SNL                  | Sandia National Laboratories       |

CO<sub>2</sub> emissions of solar-only CSP plants are estimated to be 17 g/kWh while they are on the level of 776 g/kWh and 396 g/kWh for coal and natural gas combined plants, respectively [4]. Although the investment costs of CSP plants are relatively higher compared to the conventional technologies, new plants are guaranteeing commercial maturity, increased plant efficiency, and reduced levelized costs. With increasing installed CSP capacity, investment and energy costs are estimated to fall. The levelized costs of electricity will come to the level of US \$97–130/MWh by 2015–2020 [4] from US\$194/MWh [5].

The parabolic trough solar collector (PTSC) is a dominant technology available today in both commercial and industrial scale among the medium-temperature solar collectors. In comparison to other systems, Fig. 1 shows the temperature ranges of commonly used solar thermal systems [1]. Numerous manufacturing companies have focused on this technology, Fresnel and parabolic dish technologies have become largely overshadowed. Feed-in tariffs and grant programs have driven the successful deployment of the technology, as well. Significant

research and development efforts have been put into action to improve the technology and make it competitive with counterpart energy systems [6,7]. The PTSC technology has just maintained a substantial progress in mirror and receiver development, use of alternative heat

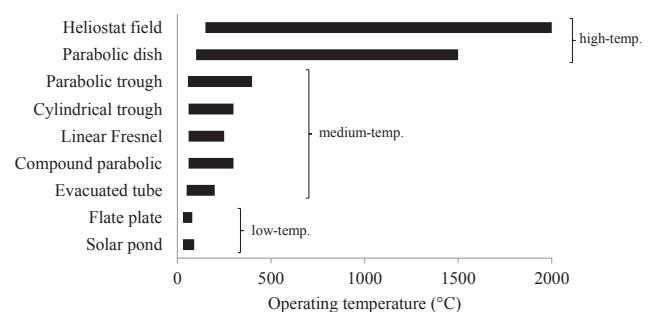


Fig. 1. Temperature ranges attainable with different solar technologies.

Download English Version:

<https://daneshyari.com/en/article/6679899>

Download Persian Version:

<https://daneshyari.com/article/6679899>

[Daneshyari.com](https://daneshyari.com)