



Thermal and economic analysis of charging and discharging characteristics of composite phase change materials for cold storage



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HIGHLIGHTS

- Solidification in open-cell metal foam is experimentally and numerically investigated.
- Porosity rather than pore density dominates solidification process.
- Natural convection affects greatly solidification front shape and temperature field.
- Using composite PCM is profitable with a short payback period.

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ABSTRACT

This study conducted both experimental and numerical investigations on the solidification behavior in a metal foam composite phase change material (PCM) for cold storage. Volume-average-method was adopted with the help of Forchheimer-Darcy equation to model the fluid flow through porous media. Experimental measurements were performed to validate the analytical model and the numerical method, with good agreement achieved. Local thermal equilibrium and non-equilibrium states were justified numerically and experimentally. Effect of pore morphological parameters (porosity and pore density) upon the solidification features of composite PCM were investigated. For the appliance of composite PCM to cold storage, techno-economic characteristics was also assessed. Results demonstrated that the full solidification time for metal foams with a porosity of 0.93 and 0.97 can be saved 87.5% and 76.7% respectively compared with pure water. It indicated that porosity of metal foam played a dominant role in heat transfer enhancement; while pore density seemed to have little influence on phase change behavior according to the results. Local natural convection in the unsolidified phase caused a remarkable promotion of the interface evolution, and the full solidification time with natural convection considered can be saved by 14.3% compared with pure conduction for the case with the same porosity of 0.97. The economic analyses indicated that using composite PCM was profitable with a short payback period less than 2 years.

1. Introduction

The energy consumption of fossil fuels all around the world accounts for as high as 87.9% in 2006, while the proportion of China is up to 93.8%. Recent studies have reported that the demand for oil would increase about 30% from 2007 to 2035 and coal and natural gas consumption would increase 50% [1]. According to research reports of 70 countries, the demand for energy has increased dramatically in recent years as many developing countries, including China, Brazil, India and

South Africa have accelerated the urbanization process and improved the living standards. This makes the energy consumption increased significantly and directly intensified the scarcity of energy [2–4]. Compared with 1970, the energy consumption of China is predicted to be more than 15 times by 2050, while Brazil and India will reach to 11 and 12 times, in respective [1]. The rapid consumption of energy has caused many adverse effects on the environment, such as global warming, energy shortages, environmental pollution, etc. [5–7]. Amongst the various demands for Heating, Ventilation and Air

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| Nomenclature | |
|----------------------|---|
| <i>Abbreviations</i> | |
| HVAC | heating, ventilation and air conditioning |
| ES | energy storage |
| TES | thermal energy storage |
| PPI | pore per inch |
| PU | polyurethane |
| MF | metal foam |
| VAM | volume-averaged method |
| FVM | finite volume method |
| TDMA | Tri Diagonal Matrix Algorithm |
| FCAC | fan coil air conditioning |
| PCM | phase change material |
| RPP | replication of polymeric path |
| COP | coefficient of performance |
| <i>Symbols</i> | |
| A | area of side and top wall of container (m^2) |
| A_m | numerical coefficient for damping velocity |
| C | capital investment (\$) |
| C_0 | profit earned by pure water as the energy storage medium (\$) |
| C_E | inertia coefficient (m^{-1}) |
| C_m | manufacturing costs (\$) |
| C_w | welding costs (\$) |
| c_d | drag coefficient |
| c_p | specific heat ($J \cdot kg^{-1} \cdot K^{-1}$) |
| $c_{p,f}$ | specific heat of fluid phase ($J \cdot kg^{-1} \cdot K^{-1}$) |
| c_e | electricity price for commercial building cooling ($kWh \cdot \$^{-1}$) |
| C_{ut} | specific material cost for metal foam ($\$ \cdot m^{-3}$) |
| d | average ligament diameter (m) |
| d_p | pore diameter (m) |
| e | thickness ratio of node to solid ligament |
| f_s | solidification fraction |
| G | shape function for metallic ligaments |
| g | gravity acceleration ($m \cdot s^{-2}$) |
| Gr | Grashof number |
| h_{sf} | interstitial heat transfer coefficient ($W \cdot m^{-2} \cdot K^{-1}$) |
| K | permeability |
| k | thermal conductivity ($W \cdot m^{-1} \cdot K^{-1}$) |
| $k_{e,f}$ | effective thermal conductivity of metal foam saturated by fluid phase ($W \cdot m^{-1} \cdot K^{-1}$) |
| $k_{e,s}$ | effective thermal conductivity of metal foam saturated by solid phase ($W \cdot m^{-1} \cdot K^{-1}$) |
| k_f | thermal conductivity in fluid phase ($W \cdot m^{-1} \cdot K^{-1}$) |
| k_p | thermal conductivity of the container ($W \cdot m^{-1} \cdot K^{-1}$) |
| L | latent heat of PCM ($J \cdot kg^{-1}$) |
| N | payback time (year) |
| n | operating hours (hr) |
| Nu | Nusselt number |
| \overline{Nu} | integral mean Nusselt number |
| P | pressure (Pa) |
| p_f | pressure in fluid phase (Pa) |
| Pr | Prandtl number |
| Q_t | total stored energy (kJ) |
| Q_{loss} | heat loss through side walls (W) |
| q_w | boundary wall heat flux ($W \cdot m^{-2}$) |
| Re | Reynolds number |
| RR | response time ($K \cdot s^{-1}$) |
| $S(t)$ | solidified layer thickness (m) |
| T | temperature (K) |
| t | time (s) |
| \vec{U} | velocity vector ($m \cdot s^{-1}$) |
| u | component velocity in x axis ($m \cdot s^{-1}$) |
| V | void volume for a ES unit (m^3) |
| v | component velocity in y axis ($m \cdot s^{-1}$) |
| V_c | fully charged unit volume for a ES unit (m^3) |
| w | component velocity in z axis ($m \cdot s^{-1}$) |
| $\overline{\phi}$ | integral mean quantity |
| \overline{RR} | integral-mean temperature response rate |
| $\langle \rangle$ | extrinsic average of a quantity over a control volume |
| $ $ | magnitude of a vector |
| <i>Greek symbols</i> | |
| α | cross-sectional area ratio of node to solid ligament |
| α_{sf} | specific area (m^{-1}) |
| β | thermal expansion coefficient (K^{-1}) |
| δ | numerical constant |
| ε | porosity |
| ρ | density ($kg \cdot m^{-3}$) |
| ρ_f | density in fluid phase ($kg \cdot m^{-3}$) |
| μ | dynamic viscosity ($N \cdot s \cdot m^{-2}$) |
| μ_f | dynamic viscosity ($N \cdot s \cdot m^{-2}$) |
| Π_r | 30-year profit (\$) |
| σ | liquid fraction in the porous medium |
| χ | flow tortuosity |
| ω | pore density |
| <i>Subscripts</i> | |
| adh | adhesive |
| amb | ambient environment |
| c | copper foam |
| con | contact state |
| e | effective parameter |
| f | phase change material |
| full | full solidification rate |
| i | initial state |
| lig | metallic ligament |
| m | solidification point |
| s | solid phase |
| td | thermal dispersion |
| w | wall |

Conditioning (HVAC) system is now playing an increasingly significant role in the total energy consumption in these days, due to the increasing demand for thermal comfort. The energy consumption in buildings accounts for 50% of the total consumption in America [5]. On one hand, the energy demand is significantly increasing; on the other hand, there exists severe mismatch of the energy supply and demand during the daytime and nighttime.

Energy (thermal or cold) storage technology provides an effective

way to balance mismatch of energy supply and demand during day and night. The air conditioning system incorporated with ice storage technology can make full use of the cheap electricity at night to reduce operating costs, favoring playing a vital role in balancing the load of the power grid during the daytime [8]. A commercialized air-conditioning system with cold storage technology (water/glycol mixtures is utilized as the cold storage medium) is successfully applied in Xi'an Xianyang International Airport, which is demonstrated to improve the overall

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