

Original Research Article

Fatigue hazards in welded plate crane runway girders – Locations, causes and calculations



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ARTICLE INFO

Article history: Received 14 October 2016 Accepted 12 May 2017 Available online

Keywords: Crane runway beams Fatigue mechanism Fatigue cracks Fatigue strength

ABSTRACT

Steel crane runway beams compared with other building structures are exposed to extremely complex load-stress conditions. It turns out, that significant from the point of view of the resistance of the crane runway beams is a cyclic nature of fluctuating loads, which leads to formation of numerous cracks and damages. This effect is especially characteristic for webs in plate I - cross sections of crane runway beams. The complex state of stresses is generated by overall bending that causes normal and shear stresses – σ_{x_i} , τ_{xz_i} , and by crane wheel eccentric load that produces respectively stresses – $\sigma_{z,x}$, $\sigma_{o,x}$, $\tau_{o,xz}$. Stress components produced by overall bending are determined as I kind stress, whereas the stress components from the crane wheel load are introduced as II kind stress. Such a combination of stresses lowers the fatigue strength of the web, which is ignored by many rules specified in standards. Limited fatigue strength is observable, among others, in crane rails splices. The results of numerical analyses obtained as II kind stresses in the web located directly beneath the crane rails splices that occur as: orthogonal contact, bevel contact and stepped bevel contact as well, confirmed the complexity of the issue. Following that, other factors, not being defined yet, but affecting the stress state of the both crane rail and crane runway beam are scheduled to be studied, as for instance, the eccentric load induced by crane trolley in mentioned above elements.

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1. Factors that influence the lowering of the fatigue strength of welded steel structures

When compared to other types of structures, crane runway beams operate under very complex load-stress conditions. One of the parameters describing the conditions of operation that affects fatigue loads is load spectrum. Fig. 1 shows, according to [1,2], schematic scatter bands of the relative stress variation ranges of stresses $\Delta \sigma_i / \Delta \sigma_{max}$ in crane runway beams, and railway bridges. The latter are widely regarded in the construction industry as structures of heavy fatigue exposure. The presented charts show (Fig. 1) that the stress intensities in crane runway beams, over the variation range spectrum, are much larger than the intensities in the spectrum of railway bridges. Detailed spectrum histograms

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http://dx.doi.org/10.1016/j.acme.2017.05.003

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xτ

distance from main maximum shear stresses

 $\tau_{1,2}$ to axis of the beam's support а spacing of long transverse ribs ılı relief factor design weld thickness aw partial factor for equivalent constant amplitude γff axial spacing of transverse ribs a₁ stress range $\Delta \sigma_{\rm E}$, $\Delta \tau_{\rm E}$ width of crane rail foot b. partial factor for fatigue strength $\Delta \sigma_{\rm C}$, $\Delta \tau_{\rm C}$ Υмf width and thickness of a single rib $b_{\rm s}, t_{\rm s}$ partial factor of load-bearing capacity of the γмо leg of right-angled triangular notch for a weld Cs cross-section eccentricity of vertical wheel load ey normal stress of the I kind in the longitudinal $\sigma_{\mathbf{v}}$ distance from the top of rail surface to the webez direction – x top flange joint equivalent normal stress for 2 million cycles $\sigma_{\rm E,2}$ nominal steel yield strength fy normal stress of the I kind due to bending $\sigma_{M,x}$ vertical wheel load F_z moment M_{ν} in the beam cross-section h_r depth of the crane rail local normal stress of the II kind in the web, $\sigma_{o,x}$ H_T transverse crane wheel load directly beneath the concentrated force F_z along depth and thickness of the beam's web h_w, t_w x-axis second moment of area of an upper crane beam Jf normal stress due to torque $\sigma_{\rm T}$ flange normal stress due to torque along x-x axis $\sigma_{T.x}$ second moment of area of the rail about its Jr normal stress due to torque along z-z axis $\sigma_{\rm T,z}$ horizontal centroid axis normal stress of the I-kind beneath the force F_z $\sigma_{z,0}$ second moment of area about y-y axis Jy vertical normal stress of the II-kind induced by $\sigma_{z,x}$ K drive force force F_z at x distance from applying point theoretical span length of a beam L amplitude normal stresses range under cyclic $\Delta \sigma, \Delta \sigma_i$ length of the uniform distribution of stresses σ_{z_i} l_{eff} load and in i-cycle, respectively $\Delta\sigma_{\max}$ maximum amplitude normal stresses range uneffective length of the weld l_{w,eff} der cyclic load during the life time M_T torque $\Delta \sigma_{x,E,2}, \Delta \sigma_{z,0,E,2}$ M_{v} bending moment about y-y axis Ν design life time of a beam expressed as a numequivalent constant amplitude stress range reber of cvcles lated to 2 million cycles along x-x and z-z axis, number of cycles of a permanent fatigue No respectively strength $\Delta\sigma_{\rm x,C}$ reference value of the fatigue strength along x-x i-cycle of fatigue load ni axis number of wheels in one crane runway beam n_k $\Delta \sigma_{z,0,E,2}$ equivalent constant amplitude stress range parameter $p_{i(i=a,k)}$ along z-z axis related to 2 million cycles relative number of investigated crane runway r equivalent constant amplitude stress range due $\Delta \sigma_{T,E,2}$ beams to torque related to 2 million cycles vield stress R, reference value of the fatigue strength along z-z $\Delta \sigma_{z,C}$ ultimate static tensile stress Rm axis load due the crane self-weight Qc $\Delta \sigma_L$, $\Delta \tau_L$ constant fatigue strength at N_L cycles equivalent fatigue load Qe $\Delta \sigma_{\rm C}, \Delta \tau_{\rm C}$ reference value fatigue strength at N_c = 2 million load-bearing capacity of the long, transverse rib Q_{gr} cycles Hoist load Qh $\Delta \sigma_{\rm E}, \Delta \tau_{\rm E}$ equivalent constant amplitude stress range rerange of crane load variability at i-cycle ΔQ_i lated to n_{max} Q_{\max} characteristic value of the maximum wheel main shear stress $\tau_{1,2}$ load shear stress due to the torque at plane xz $\tau_{T,xz}$ S static moment of the cross-section portion over shear stress of the I kind under transverse force $\tau_{V,xz}$ the z–z coordinate V_z thickness of the flange in a cross-section t_f local shear stress of the II kind at xz plane under $\tau_{o,xz}$ shear force Vz the concentrated force F_{z} distance of a butt weld from external surface of Z_1 local shear stress of the I kind at xz plane τ_{xz} the top flange shear stress in the weld τ_{O} effective concentration factor β_k $\Delta \tau$ amplitude shear stress range under cycle load initial deflection (flexure) of the web Δ_{o} equivalent amplitude shear stress range at xz $\Delta \tau_{\rm xz.E.2}$ dynamic factor due to self-weight of crane and φ_1, φ_2 plane related to 2 million cycles hoist load, respectively amplitude main shear stress range $\Delta \tau_{1,2}$ dynamic fatigue factor φ_{fat} amplitude shear stress range in the weld under $\Delta \tau_{O}$ $\varphi_{fat,1}, \varphi_{fat,2}$ dynamic fatigue factor for a self-weight of the

crane and to hoist load, respectively

cyclic load

Nomenclature

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