Construction and Building Materials 168 (2018) 532-546

Contents lists available at ScienceDirect

Construction and Building Materials

journal homepage: www.elsevier.com/locate/conbuildmat

Research on extending the fatigue life of railway steel bridges by using intelligent control

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HIGHLIGHTS

• Optimum multi-scale structural analysis combining shell and solid FE models of a bridge.

- Calculating fatigue life of a steel bridge considering both dynamic and thermal actions.
- Smart vibration control towards optimising fatigue performance of steel bridges.
- Developing a case study application for the Poyang Lake Bridge in China.

ARTICLE INFO

Article history: Received 31 October 2017 Received in revised form 18 February 2018 Accepted 19 February 2018

Keywords: Sub-model/multi-scale method Railway steel bridges Fatigue Structural control

ABSTRACT

This paper investigates the potential of a vibration control-inspired method towards extending the fatigue life of railway steel bridges. Based on coupled thermal-mechanical and vehicle-track analysis, both the residual stresses from welding and these from traffic on the bridge are obtained. Subsequently, a multi-scale approach with a shell Finite Element (FE) model of the whole bridge and a solid FE model of its critical joints is put forward. The equation of motion is established for the controlled bridge, equipped with a Magnetorheological-Tuned Mass Damper (MR-TMD) system, while the combination of excitation, welding and control effects is practiced through own-developed packages and commercial software sub-model routines. The framework is showcased for the study of the Poyang Lake Railway Bridge in China. After obtaining the controlled stress states at the critical welded joint, the fatigue crack initial life is evaluated by using the critical plane method and the linear cumulative damage theory. Simulation results indicate that the multi-scale modelling approach followed, meets the accuracy needs for capturing the cracking process of the welded joint stress amplitudes, can improve substantially the overall bridge fatigue resistance over the uncontrolled structure.

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1. Introduction

Railway steel bridges are very common long span bridges, constituting a large sector of the traffic communication network. Owing to their long time and frequent train traffic, cumulative damage may occur in their welded regions. In particular, stress concentrations can easily be generated under the combined action of multi-axial dynamic-excitation stress and welding-owed residual stress. Such could further lead to fatigue damage that sets off as crack initiation and, in a worst case scenario, could progress to failure of parts of the bridge, putting a serious concern in terms

* Corresponding authors. E-mail address: n.nikitas@leeds.ac.uk (N. Nikitas). of life cycle design. Collapse and damage of steel bridges caused by cumulative fatigue action have been frequently reported across the world up to this date [1–3]. Therefore, it is important to propose and adopt effective means to prolong the fatigue crack initial life of steel bridge parts; such could subsequently extend the service life of railway steel bridges, and reduce the risk, economic losses and casualties caused by potential damage.

In the past decades, many scholars have carried out considerable theoretical and experimental research in the field of bridge fatigue life assessment. Namely, within the achievements produced in the field one can quote the establishment of fatigue damage models, the study of initial crack mechanisms and the evaluation of a structure's overall fatigue life [4,5]. Li and Chan, for instance, applied continuous damage mechanics to their





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Nomenclature

- the y directional thermal conductivity factors k_v
- the z directional thermal conductivity factors k_z efficiency of the heat source 2
- U_w welding voltage
- welding current
- I_w volume of the welding unit V_H
- the parent material density Ø
- Ċ specific heat capacity
- Т the joint temperature generated by the welding
- the independent time variable t
- N_x the perpendicular to the boundary of x direction cosines
- N_{v} the perpendicular to the boundary of y direction cosines
- the perpendicular to the boundary of z direction cosines Nz
- the heat transfer coefficient of convection h_c
- the heat transfer coefficient of radiation h_r
- boundary heat flux q_s
- \hat{T}_r temperature of radiation
- T_{∞} surrounding temperature
- the heat source distribution of the double ellipsoid ff model for front heat source
- fr the heat source distribution of the double ellipsoid model for rear heat source
- the heat source of the double ellipsoid model for front q_f heat source
- the heat source of the double ellipsoid model for front q_r heat source
- v welding speed
- the x coordinate of the welding initial position Xn
- arc welding parameter a_1
- arc welding parameter a2
- arc welding parameter h
- arc welding parameter С
- vector of stress $\{d\sigma\}$
- $\{d\varepsilon\}$ vector of strain
- dΤ temperature increment
- [D]elastic or elastic-plastic constitutive law matrix
- {C} temperature dependence vector
- M_c weight of the train car inertia of the train car
- Jc M_t weight of bogies
- inertia of bogies I+
- M_{wi} the wheel's weight
- vertical displacement of each wheel v_{wi}
- vertical displacement of the bogies v_{ti}
- φ_{ti} rotation angle of the bogies
- vertical displacement of the repeating train car v_c
- rotation of the repeating train car φ_c
- u_1, u_2 the rail vertical displacements
- u_3, u_4 the railway bridge sleeper vertical displacements
- the ballast vertical displacement u_5, u_6
- the elastic coefficients of the fastener k_{v1} the elastic coefficients of the ballast
- k_{y2} the elastic coefficients of the railway bridge sleeper k_{v3}
- the damping coefficients of again the fastener
- C_{y1} the damping coefficients of again ballast
- C_{V2} the damping coefficients of again the railway bridge C_{V3}
- sleeper
- M_v the mass matrices of the train the damping matrices of the train Cv
- stiffness matrices of the train Kv
- the acceleration vector äv
- the velocity vector ά_ν
- the displacement vector av $\mathbf{Q}_{\mathbf{V}}$ the force vector

the equivalent spring stiffness kн the interpolating function of u_1 and u_2 of the nodal dis u_i placement M_C the mass matrices of the coupled system C_{C} the damping matrices of the coupled system Kc the stiffness matrices of the coupled system Q_C the load vector C_{VI}, C_{IV} the coupling damping spatial angular frequency variable 0 Ω_c the truncated spatial angular frequency of the vertical profile irregularity Ω_r the truncated spatial angular frequency of the alignment irregularity A_{v} the high interference roughness coefficient Z_r, Z the vertical displacements of the bridge deck and TMD mass respectively ÿ the acceleration vector of the bridge discrete model the velocity vector of the bridge discrete model Ż х the displacement vector of the bridge discrete model Μ the mass matrices the stiffness matrices C Κ the damping matrices н locator matrix of installed MR-TMD system F(t)the external force input the control force vector provided by the MR-TMD sys-U_{MR-TMD} tem the stiffness properties of the i^{th} TMD in the MR-TMD K_{Ti} the damping properties of the i^{th} TMD in the MR-TMD C_{Ti} the damping force of the i^{th} MR damper F_{MRi} the viscous damping coefficient C_d the coefficient of controllable Coulomb damping force F_d K_d the equivalent axial stiffness of the damper the output force deviation caused by the damper accu f_{0i} mulator the Bingham sliding displacement eь the coefficient of viscous damping of damper on the C_{ds}

the mass, damping vector of the track

the stiffness matrices vector of the track

the element identifier of the 2D track beam

the damping vector of the track

the load vector of the track

- condition of zero electric field strength
- the coefficient of controllable Coulomb damping force of F_{ds} damper on the condition of zero electric field strength K_{ds} the equivalent axial stiffness of damper on the condition
- of zero electric field strength the voltage sensitivity of viscous damping of the dam- C_{dd} per
- F_{dd} the voltage sensitivity of controllable Coulomb damping force of the damper
- the voltage sensitivity of equivalent axial stiffness of the K_{dd} damper
- the time coefficient of the damper's magnetic hysteresis η response
- I the applied current intensity
- the internal variable reflecting the relationship between и model parameters and current intensity. the maximum coefficient $F_{d \max}$
- the minimum coefficient $F_{d\min}$
- the adjustment factor of the Coulomb damping force Ĕ
- the *x*-directional normal strain \mathcal{E}_{X}
- the y-directional normal strain ε_y
- the z -directional normal strain \mathcal{E}_Z
- the *xz* -directional shear strain Exy
- the yz -directional shear strain Eyz

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