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# Revealing the effects of damping on the flow-induced vibration of flexible cylinders



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#### ABSTRACT

This study reveals how damping shapes the global vortex-induced vibration (VIV) response of flexible cylinders. Global behavior may vary from full-length standing waves to traveling waves on infinite cylinders. Structural damping rules the standing wave case whereas radiation damping regulates VIV response on very long cylinders. A single scalar equation expresses the balance of power flowing through the structure. In that equation, A<sub>rms</sub>, which is the root-mean-square response in the VIV excitation region, is shown to be an excellent indicator of global response because of its relation to power flow. Under steady-state conditions, the net power flow must be zero, which directly leads to three independent dimensionless damping parameters, namely  $\alpha$ ,  $\beta_R$ , and  $c^*$ .  $\beta_R$  indicates when radiation damping is important,  $\alpha$  reveals the relative importance of structural versus radiation damping, and c\* locates the global VIV behavior on the spectrum of lightly to strongly damped systems. Structural, hydrodynamic, and wave radiation damping are all taken into account. Plots of  $A_{rms}^*$  versus  $c^*$  show the global effects of damping on response. Uncontrolled factors often reveal themselves as graphical anomalies, leading to new insights on VIV. Data from experiments and numerical simulations are presented to support the conclusions.

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#### 1. Introduction

Introducing flow-induced vibration to a newcomer is most often accomplished with reference to a few key dimensionless parameters. Strouhal number, Reynolds number, mass ratio, and reduced velocity immediately come to mind when explaining how the vortex-induced vibration (VIV) world works. Remarkably, there has been limited success in finding useful dimensionless parameters involving damping. In the 1950s, Scruton introduced what is today called the mass-damping parameter,  $m^*\zeta_s$ , which revealed a remarkably high correlation with peak response amplitude of smoke stack models in wind tunnels [1]. Unfortunately,  $m^*\zeta_s$  is based on structural damping only and is of no use for long flexible cylinders, because it does not account for hydrodynamic or radiation damping. Currently, there are no dimensionless damping parameters that are able to place global structural response on a spectrum of lightly to heavily damped systems. The primary goal of this study is to define a set of dimensionless parameters, which show the role of damping in the regulation of the VIV of flexible cylinders.

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Nomenclature	
<i>A</i> *	Non-dimensional local peak amplitude
$A_{1,rms}$	Root-mean-square (RMS) amplitude for outbound wave
$A_{2,rms}$	RMS amplitude for returning wave
Arms	Spatial and temporal RMS of $y(x, t)$ in excitation region
$A^*_{rms}$	Non-dimensionless value of $A_{rms}$ in excitation region
c(x)	Damping coefficient, damping/length
ca	Added mass coefficient
Ce	Equivalent damping coefficient
C <sub>h</sub>	Hydrodynamic damping coefficient
Cout	Damping coefficient in power-out region
Cs	Structural damping coefficient
С*	Dimensionless damping parameter for spring-mounted cylinder
$c_e^*$	Dimensionless equivalent damping parameter
$c_u^*$	Dimensionless damping parameter for dominant structural damping cases
$c_Z^*$	Dimensionless damping parameter for dominant radiation damping cases
$C_L(x,t)$	Lift coefficient
$C_{L,rms}$	RMS value of $C_L(x, t)$ in power-in region
$C_y(x,t)$	Force coefficient
$C_{y0}$	Peak value of $C_y(x,t)$
$C_{y,rms}$	RMS value of $C_y(x, t)$ in power-in region
D	Cylinder diameter
EI	Bending stiffness
$f_{ u}$	VIV response frequency in Hz
$f(\mathbf{x}, \mathbf{t})$	Force per unit length
k	Wave number
L	Pipe length
L <sub>in</sub>	Length of power-in region
$L_{out}$	Length of one power-out region Mass/longth with added mass
$m_{x}$	Mass/length (kg, $m^{-1}$ ) fairing only in air
mfairing	Mass/length (kg $m^{-1}$ ) in air without fairing
m*Y	Mass/length (kg·m) in an without failing
$n \zeta_s$	Number of half waves in one power-out region
$P(\mathbf{x})$	Tension
$r_1$	Amplitude ratio $A_{\rm rms}/A_{\rm 1}$ mass
S.	Stroubal number
T	Oscillation period of pipe
Ū	Flow speed
$U_{rmc}^2$	Average mean square flow velocity in excitation region
Va	Energy velocity/group velocity of wave
Vr	Reduced velocity
$V_{\phi}$	Phase velocity of wave
x	Axial coordinate along pipe
y(x,t)	Time series displacement
<b>y</b> rms	RMS of $y(x,t)$
$Z_R$	Impedance of a tensioned cable/beam
ζ <sub>out</sub>	Damping ratio in power-out region
ζs	Structural damping ratio
α	Structural damping parameter
$\beta_R$	Wave attenuation parameter
λ	Wave length
ρ	Density of fluid
ω	Response frequency in rad/s
$\omega_n$	Natural frequency for mode <i>n</i> in rad/s

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