



# Boundary control of flexible satellite vibration in planar motion



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## ABSTRACT

In this paper, the planar maneuver of a flexible satellite with regard to its flexible appendages vibration has been studied. The flexible satellite translates and rotates in a plane; in addition, the flexible appendages can also vibrate in that plane. The system governing equations, which are coupled partial and ordinary differential equations, are obtained based on Hamilton's principle. Then the original system converts to three equivalent subsystems, two of which contains one partial differential equation and one ordinary differential equation along with four boundary conditions, by using change of variables. Employing control forces and one control torque which are applied to the central hub, control objectives, tracking the desired angle and suppressing the satellite and its flexible appendages vibrations, are fulfilled. Furthermore, to eliminate spillover instability phenomenon and to exclude in-domain measurement and actuator usage problem, these control torque and forces are designed based on boundary control method. Lyapunov's direct method is employed to prove the asymptotic stability in absence of any damping effect in modeling the vibrations of flexible appendages. Eventually, in order to demonstrate the effectiveness of the designed boundary control, numerical simulations are presented.

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## 1. Introduction

Communicate with earth, remote sensing, and satellite imagery are the most important reasons of using satellites in past decades [1]. Flexible satellites, a large group of satellite, comprise interconnection between a rigid structure and flexible appendages such as large solar arrays or antennae. Consequently, the system governing equations are coupled partial and ordinary differential equations. Due to the coupling between appendages vibration and the rigid body dynamics of the satellite, appendages vibration affects on satellite maneuvers and causes the satellite translation and rotation; in addition, the satellite rotation or translation leads to the appendages vibration. Therefore, tracking the desired angle along with suppressing the satellite and its flexible appendages vibration must be accomplished simultaneously.

Dealing with the control design of systems whose governing equations consist of PDEs and ODEs, such as a flexible satellite system, has always been elaborated. Usually, such a system is discretized into a set of ODEs; afterward the obtained ODEs are

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### Nomenclature

$A$	cross-sectional area of the panels ( $\text{m}^2$ )
$c$	velocity of wave propagation in cantilever beams
$E$	Young modulus of the flexible panels ( $\text{kg/m}^2$ )
$f$	control force in $x_b$ -direction
$J$	inertia of central hub ( $\text{kg m}^2$ )
$I$	area moment of inertia of the flexible appendages cross-section ( $\text{m}^4$ )
$l$	length of each flexible appendages (m)
$L$	Lagrangian function of the system ( $\text{N m}^2$ )
$m$	hub mass (m)
$r$	each appendage distance from the hub center (m)
$t$	time (s)
$T$	Kinetic energy
$u$	control force in $y_b$ -direction
$U$	Potential energy
$V$	Lyapunov function
$W$	work of control torque on the satellite
$x$	space independent variable (m)
$\dot{x}_{hub}, \dot{y}(t), \dot{X}(t), \dot{Y}(t)$	satellite velocity in $i$ -, $j$ -, $I$ - and $J$ -axis, respectively
$\tau$	boundary control torque (N m)
$\rho$	density of the panels ( $\text{kg/m}^3$ )
$\theta$	pitch angle (radian)
$\theta_d$	desired pitch angle (radian)
$e_\theta$	pitch angle tracking error (radian)
$e_y$	displacement tracking error in the special case (m)
$w_R, w_L$	elastic deformation of right and left appendage element (m)
$E_a$	a summation of the potential and Kinetic energy of the flexible appendages
$E_{as}, E_s$	a summation of the potential and kinetic energy of the flexible appendages in special cases
$V_s, V_{as}, \Delta_{as}$	some terms in Lyapunov function
$\alpha_0, \alpha_1, \beta_1, \beta_2, k_1, k_2, c_x$	constant coefficients in Lyapunov function and the controller
$q_{s-i}, q_{as-i}$	generalized coordinates
$\varphi_i$	the normal modes for clamped-free Euler-Bernoulli beam
$S_i$	invariant set
$M_g$	couple in $z$ -direction due to angular velocity in $x_b$ - and $y_b$ -direction
$I_{x_b x_b}, I_{y_b y_b}$	inertia of satellite in $x_b$ - and $y_b$ -direction ( $\text{kg m}^2$ )
$\omega_{x_b}, \omega_{y_b}$	angular velocity in $x_b$ - and $y_b$ -direction ( $\text{kg m}^2$ )

controlled by various control methods of ordinary differential equations [2,3]. In the field of the flexible satellite dynamic modeling and control based on discretizing model, the satellite hub along with the flexible appendages are considered as a rigid body and Euler-Bernoulli beam, respectively [4,5]; moreover, in some other investigations, Euler-Bernoulli beams with tip masses are considered as the flexible satellite appendages [6,7].

In Ref. [8], flexible orbiting space systems are modeled and a formulation suitable for synthesizing optimal control strategies during the deploying maneuvers of robotic arms or solar arrays is presented. In Refs. [9,10], the satellite rotational motion and anti-symmetric vibration in a plane are controlled. In Refs. [11,12], the three-axis maneuver control and vibration suppression of a flexible satellite in a circular orbit are studied. In Ref. [13], the three-axis maneuver control of the flexible satellite in the face of faulty actuators, system uncertainties, actuator saturation and external disturbances are studied. In Ref. [14], disturbance effects like gravity gradient torque are investigated. In Ref. [15], system uncertainties in the flexible satellite control are studied. In Refs. [16,17], in addition to the control torque used for the flexible satellite maneuver control, the surface piezoelectric layer is used for active vibration suppression. Furthermore, different control methods are addressed in previous investigation, such as sliding mode [6], robust control [18], variable structure control [19], adaptive control [20], and non-linear PD control [21].

In some discretized model, the proposed controllers are difficult to implement, for expensive distributed actuator and sensor instruments or a large set of properly located sensors and actuators are required. Moreover, in some cases, some sensors are needed to feedback information in the domain and an observer must be used to estimate the required information. Furthermore, the high order modes are ignored in the discretized model for the distributed system and as a result, these neglected modes will cause spillover instability that should be avoided in the control design. All aforementioned

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