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³ Design and implementation of an out-of-plane electrostatic vibration ⁴ energy harvester with dual-charged electret plates

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ABSTRACT

This paper presents the design, simulation and implementation of an out-of-plane electrostatic power 29 generator with dual-charged electret plates for low-level ambient kinetic energy harvesting. The rota- 30 tional symmetrical resonator includes one movable disk-shaped circular mass and a series of spiral 31 springs for suspension. The whole device is fabricated by CMOS compatible silicon micromachining tech- 32 nology with an overall volume of about 0.12 cm^3 . The two-plate device has both positive and negative 33 charged electret plates. Experimental analysis shows that the present prototype is able to achieve an out- 34 put power of 0.34 µW at a low resonance of 66 Hz at 0.5g, which corresponds to a normalized power den- 35 sity of 11.67 µW cm⁻³ g^{-2} . With an acceleration changing from 0.1 to 0.5g, it is observed the operating 36 half-power bandwidth increases by 2.6 times from 2.5 to 6.5 Hz. This may be attributed to the spring soft-
37 ening nonlinearity induced by the strong electrostatic force. The results could potentially provide an 38 intriguing design methodology for developing nonlinear MEMS devices for broadband random energy 39 harvesting. 40

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45 1. Introduction

 Energy harvesters from ambient natural sources, such as envi- ronmental vibrations and ambient heat, have generated great interest amongst both scientific and industrial communities in past decade. Such harvesters may pave the primary step forward to actualizing of self-autonomous devices and performing of intelli- gent monitoring activities [\[1–3\].](#page--1-0) Typical vibration-based energy harvesters normally make use of either electromagnetic [\[4\],](#page--1-0) piezo- electric [\[5,6\]](#page--1-0), or electrostatic transduction mechanism [\[7\]](#page--1-0). Among these, electrostatic energy harvesters have the advantages in micro-scale applications as they are more compatible with the sili- con CMOS processes offering greater flexibilities in the design of 57 mechanical and electrical features separately $[8,9]$. Specifically, an electrostatic energy harvester mainly operates on the capaci- tance change of biased variable capacitors to initiate the conver- sion. The bias source can be either provided by pre-charged electret material or an external voltage. Electret-based electrostat- ic vibrational energy harvester (EVEH) is one where the sustainable voltage source is based on a special electret material. An electret is

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<http://dx.doi.org/10.1016/j.mee.2015.02.036> 0167-9317/© 2015 Published by Elsevier B.V. a dielectric with quasi-permanent electric charge or dipole 64 moment, which can maintain an electric field for tens of years. 65

A conventional EVEH usually has an in-plane movable proof 66 mass patterned with parallel-interdigital electrodes and the 67 corresponding stripe-shaped electrets. The electrical current is 68 generated when the overlapping area varies in response to an 69 external horizontal vibration source. In order to enhance the per-

70 formance, the values of electret pattern width are usually designed 71 in ten to hundred micrometer level to be compatible with the tiny $\frac{72}{ }$ displacement of the proof mass $[10]$. This, however, gives rise to 73 problems in terms of low charging efficiency and rapid charge 74 decay [\[11,12\].](#page--1-0) Furthermore, the optimal working condition for 75 such harvester is when two electrodes have precisely alignment 76 to each other. Even small scale of misalignment may affect or lower 77 its output performance. In order to overcome these problems, Chiu 78 et al. [\[13\]](#page--1-0) proposed an out-of-plane EVEH with a non-patterned 79 negative-charged plate that operates on the variation of the air 80 gap, where neither precise alignment nor micro-patterning of the 81 electret film is necessary. Boisseau et al. [\[14\]](#page--1-0) also introduced an 82 out-of-plane EVEH with cantilever beam–mass system with a posi- 83 tive charged electret plate as permanent voltage source. However, 84 the fabrication of these devices is based on manual cutting process 85 that is neither practical for mass-production nor compatible to 86 integration with other electronic devices. These out-of-plane 87 EVEHs are only based on single-charged electret plate. 88

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Fig. 1. Schematic of the conventional single-charged out-of-plane gap closing EVEH (left) compared with the new designed concept with double-charged plates (right).

 On the other hand, typical energy harvesting devices tend to operate within narrow bandwidths where the vibration energy can only be scavenged when the resonance matches with the fre- quency of the external vibrations. Various nonlinear techniques have been exploited to broaden the operation bandwidth, such as 94 utilizing mechanical prestress [\[15\]](#page--1-0), special spring design [\[16,17\],](#page--1-0) 95 stopper impacts, electromagnetic force [\[18\]](#page--1-0) or a combination of 96 prestress and stopper impacts [\[19\]](#page--1-0). However, none of these works have attempted to use the electrostatic force created by the elec- tret field to harvest energy over a wide operating bandwidth, which is practically more advantageous and compatible with MEMS/CMOS micromachining technology process. This paper pre- sents an out-of-plane EVEH structure that combines both positive and negative charged electrets into single resonant system. Schematic of the conventional single-charged out-of-plane gap closing EVEH compared with the new designed concept with dou-105 ble-charged plates is demonstrated in $Fig. 1$. By adding positive charged electrets into the new concept device, as shown on right 107 side of Fig. 1, the enhanced electric field within two plates will give rise to an increased electrostatic force, which would induce a stronger soft spring effect of spring–mass system as well as a high- er output voltage. These aspects will be introduced in the following sections.

112 2. Design and modeling

113 2.1. Device configuration

 Fig. 2 exhibits 3D EVEH structure that enables to scavenge vibrational energy using an out-of-plane scheme. The device is mainly constructed with two parallel Si plates with gold elec- trodes. The spring–mass resonant structure has a circular mass with 6 mm in diameter at the center and three parallel suspension 119 spiral beams around with 50 μ m in width and 300 μ m in height. 120 The spacing between each adjacent beam is $250 \mu m$. Electret thin

Fig. 2. 3D schematic view of the energy harvester.

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films, acting as permanent charge voltage source are mounted on 121 the both sides of electrodes. When the device is excited by out- 122 of-plane vibration, the relative movement of the two electrode 123 plates would generate an alternating current through the external 124 load due to the capacitance change between top and bottom 125 electrodes. 126

2.2. Mechanical model 127

Finite element method is employed to analyze the stress distri-
128 bution and dynamic behavior of the spring–mass system of the 129 generator. Static structure and modal analyses performed by 130 ANSYS simulation are used to determine the stress and the vibra- 131 tion mode, respectively. Fig. $3(a)$ shows the shape and resonant fre- 132 quency of the primary vibration mode. It can be found that the 133 spring–mass resonant structure vibrates at a low resonant fre- 134 quency of about 95 Hz in an out-of-plane direction at its primary 135 mode. Since the natural frequency of resonator is dependent on 136 the beam geometry and proof mass, resonance of the system can 137 be easily tuned from tens to hundreds of Hz. From our previous 138 investigation $[8]$, the proposed spring–mass resonant structure is 139 not only capable of vibrating in an out-of-plane direction at mode 140 I but also able to oscillate with in-plane directions at mode II and 141 mode III. It is observed that the robustness of the system is mainly 142 determined by the out-of-plane vibration of mode I, since collapse 143 is more likely to take place in out-of-plane direction due to the 144 occurrence of 'pull-in' effect. Although the in-plane oscillation 145 modes could possibly be excited near their resonant frequencies, 146 their maximum oscillation amplitudes are constrained by limited 147 frame space; therefore, the severe damage of device could be 148 avoided. Thus, the robustness of the system is mainly dominated 149 by its primary out-of-plane excitation mode as well as by the high 150 aspect ratio of the micro machined Si structures. 151

The stress distribution of the parallel spiral springs is further 152 investigated by ANSYS when the proof mass is displaced of 153 300 μ m, as presented in [Fig. 3\(](#page--1-0)b). The simulation results show 154 the maximum stress is only about 18.6 MPa occurring around the 155 beam anchors. Although single-crystal Si displays high fracture 156 limit (up to 7 GPa) below a ductile–brittle transition temperature 157 of \sim 500 °C [\[20\],](#page--1-0) its fracture strength is more likely to be affected 158 by its intrinsic imperfections such as the size and population of 159 surface defects $[21]$ and extrinsic conditions such as the stress 160 intensity and high-cycle fatigue loading $[22]$. Previous research 161 has reported that the fracture strength of silicon beam could be 162 diminished by 33–75% by using different grade polish paste $[23]$. 163 Since the calculated maximum stress is only in MPa level, which 164 would still be substantial less than the high fracture strength 165 reported to be of the order of GPa. One can therefore infer 166 that the device should work adequately under the current 167 circumstances. 168

2.3. Electro-mechanical dynamic model 169

[Fig. 4](#page--1-0) illustrates schematic model of an EVEH where both the 170 mechanical and the electrical parts are incorporated. The behavior 171 of the harvester is governed by a basic nonlinear differential equa- 172 tion and is expressed as:

 $m\ddot{x} + b_m\dot{x} + kx = -m\ddot{y} + Fe$ (1) 176

where k is the spring stiffness, b_m is the mechanical damping, m is 177 the seismic mass, Fe is the electrostatic force and y is the external 178 vibration excitations ($y = Y\sin(\omega t)$), respectively. As depicted in 179 [Fig. 4](#page--1-0), $C(t)$ is the overall capacitance between the fixed and movable 180 electrodes, which is consist of three serial capacitances, the variable 181 capacitance $C_1(t)$ and the two fixed electret capacitances C_2 and C_3 , $g = 182$ and *d* denote the thickness of air gap and electret thin film, 183

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