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## Nonlinear dynamic analysis and robust controller design for Francis hydraulic turbine regulating system with a straight-tube surge tank



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#### ABSTRACT

The safety and stability of hydraulic turbine regulating system (HTRS) in hydropower plants become increasingly important since the rapid development and the broad application of hydro energy technology. In this paper, a novel mathematical model of Francis hydraulic turbine regulating system with a straight-tube surge tank based on a few state-space equations is introduced to study the dynamic behaviors of the HTRS system, where the existence of possible unstable oscillations of this model is studied extensively and presented in the forms of the bifurcation diagram, time waveform plot, phase trajectories, and power spectrum. To eliminate these undesirable behaviors, a specified fuzzy sliding mode controller is designed. In this hybrid controller, the sliding mode control law makes full use of the proposed model to guarantee the robust control in the presence of system uncertainties, while the fuzzy system is applied to approximate the proper gains of the switching control in sliding mode technique to reduce the chattering effect, and particle swarm optimization is developed to search the optimal gains of the controller. Numerical simulations are presented to verify the effectiveness of the designed controller, and the results show that the performances of the nonlinear HTRS system assisted with the proposed controller is much better than that with the commonly used optimal PID controller.

#### 1. Introduction

Many hydropower plants have been built worldwide to utilize the energy of falling water or flowing water for electricity goal by various turbines, such as Francis turbine [1], Kaplan turbines [2], and pump-turbine [3]. An important part of a hydropower plant is the hydraulic turbine regulating system (HTRS), which is used to maintain the safety and stability of hydropower generating unit (HGU) [4]. The HTRS system is in nature a complex nonlinear, time-varying and non-minimum phase system, which involves the interactions among hydraulic system, mechanical system, and electrical system. More specifically, some important inner parameters in the HTRS system are dependent on the water head, and a number of certain nonlinear features like coupled characters among different devices and operational uncertainties have been observed in hydro system [5–7]. As these complicated properties, the control performances of the HTRS influence the working conditions of a hydro plant significantly. For instance, the oscillatory

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Nomenclature		δ	generator rotor angle relative deviation	
		$\omega_0$	generator based angular speed	
Francis	hydraulic turbine regulating system with a	$T_a$	generator mechanical time constant	
straight-tube surge tank		D	generator damping constant	
		$m_e$	generator electromagnetic torque	
$h_1$	the water head at upstream penstock	$P_e$	generator electromagnetic power	
$h_2$	the water head at surge tank outlet	$E_q'$	generator transient voltage of quadrature axis	
$h_3$	the water head at downstream penstock	$V_s$	generator voltage of infinite bus system	
$h_{f1}$	the water head loss at upstream penstock	$x'_{d\sum}$	generator direct axis transient reactance	
$q_1$	the water flow rate at upstream penstock	$x_d$	generator direct axis reactance	
$q_2$	the water flow rate at surge tank outlet	$x_T$	short-circuit reactance of generator transformer	
$q_3$	the water flow rate at downstream penstock	$x_L$	the reactance of transmission lines	
$T_{w1}$	the water inertia time constant of downstream	$\chi_{q\Sigma}$	generator quadrature axis transient reactance	
	penstock	$x_q$	generator quadrature axis reactance	
$T_{j}$	the water storage time constant of surge tank	$d_1 \sim d_6$	the uncertain system state perturbation	
$T_{w3}$	the water inertia time constant of downstream			
	penstock		FSMC controller and PID controller	
$T_p$	the pilot actuator time constant of servo system			
$T_{y}$	the main gate time constant of servo system	f(x), g	g(x), d the composite equations in Eq. (27) of	
$m_t$	the incremental torque deviation		parameters of HTRS system	
q	the incremental turbine flow deviation	$u_{eq}$	the equivalent control law (approaching item)	
$x_t$	the incremental turbine speed deviation	$u_{sw}$	the switching control law (reaching item)	
y	the incremental guide vane/wicket gate position	S	the designed sliding mode surface	
	deviation	c	a positive constant	
h	the incremental turbine head deviation	η	a positive constant lager than $D_r$	
$e_h$ , $e_y$ ,	$e_x$ partial derivatives of the hydraulic turbine tor-	sgn(s)	sign function	
	que with respect to head, guide vane and turbine	sat(s)	a high-slope saturation function	
	speed	$\mu_{fuzzy}$	the normalization value of fuzzy system	
$e_{qh}$ , $e_{qy}$ , $e_{qx}$ partial derivatives of the hydraulic turbine		$k_p$	the proportional adjustment coefficient	
	flow with respect to head, guide vane and	$k_i$	the integral adjustment coefficient	
	turbine speed	$k_d$	the differential adjustment coefficient	

problems in HGU were reported to be closely related to the possible Hopf bifurcation and chaotic oscillation occurs in HTRS system [8]. Hence building a mathematical model of the HTRS system and carrying out the stability analysis of it are a necessary work.

The literatures review reveals that considerable research efforts have been devoted to the modeling of hydraulic system and its stability criteria. In [9], the authors studied the small disturbance stability of hydropower plants with complex conduits and a linear turbine model. In [10], there developed an interesting nonlinear and liberalized turbine models during the transient stability analysis in the free source Power System Analysis Toolbox (PSAT) software. In [11], the authors modeled the hydraulic turbine system in accordance with the response of an actual Francis turbine. Meanwhile, there are plenty of published papers about the models of some individual parts with respect to the HTRS system [12-14]. For instance, an elastic model and non-elastic model based on first-order differential equations for the conduit system have been established in [12,13]. As the capacity of the hydrogenerator system becomes larger, and the water head becomes higher, especially the widely use of various surge tanks, the robustness and the stability of the nonlinear HTRS system meets some new challenges. Consequently, a number of improved mathematical models with partial nonlinear terms have been proposed in recent years [15-17]. These models described the characters of the HTRS system in different ways and tried to approximate the prototype system as much as possible, but it is noted that most of them are focused on the traditional regulation without concerning the system uncertainties, which are commonplaces for the nonlinear systems. With the development of nonlinear science and improvement of computer capacity in data processing, nonlinear dynamic analysis has gradually been used in many fields, such as vehicle system [18], rotor system [19], and power system [20], which has an advantage to the other stability analytical methods [19]. Therefore, it is natural to apply the nonlinear dynamic analysis method into the HTRS system with uncertainties.

On the other hand, to guarantee the unconditional stability, an innovative control method should be proposed when the system is operating in the undesirable states. The tracking control of nonlinear systems has been a well-established area in the last few decades. A widely applied tracking control approach is the sliding mode control because of its high robustness and availability [21]. However, after sustained researches on the sliding mode control, there found that the major drawback of this technique is the chattering effect, due to the use of discontinuous sign functions. Various techniques have been suggested to minimize this bad effect, such as selecting smooth approximations like *sat*, *tanh* and other continuous control laws instead of the sign function [22]. However, the practical implementations show that this simple-instead type sliding mode controller (SMC) is easy to excite the unexpected high frequency responses of the control systems [23]. To tackle this problem, fuzzy logic controller (FLC) is often used. Integration of SMC and FLC methods overcomes the weakness of one method over the other so that the guaranteed performance can be achieved. With the practical application of this hybrid fuzzy sliding mode controller (FSMC), many researchers have suggested different

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