



ELSEVIER

Contents lists available at ScienceDirect

Tribology International

journal homepage: [www.elsevier.com/locate/triboint](http://www.elsevier.com/locate/triboint)

# Influence of the normal force, abrasive slurry concentration and abrasive wear modes on the coefficient of friction in ball-cratering wear tests

Ronaldo Câmara Cozza<sup>a,b,\*</sup>

<sup>a</sup> Centro Universitário da FEI – Fundação Educacional Inaciana “Padre Sabóia de Medeiros”, Departamento de Engenharia Mecânica, Av. Humberto de Alencar Castelo Branco, 3972, São Bernardo do Campo, SP 09850-901, Brazil

<sup>b</sup> CEETEPS – Centro Estadual de Educação Tecnológica “Paula Souza” – FATEC-Mauá, Av. Antônia Rosa Fioravante, 804, Mauá, SP 09390-120, Brazil

## ARTICLE INFO

### Article history:

Received 24 September 2012

Received in revised form

16 January 2013

Accepted 13 September 2013

Available online 8 October 2013

### Keywords:

Ball-cratering

Abrasive wear

Coefficient of friction

## ABSTRACT

The purpose of this work is to study the influence of the normal force ( $N$ ), abrasive slurry concentration ( $C$ ) and abrasive wear modes on the coefficient of friction in ball-cratering wear tests. Experiments were conducted with balls of AISI 52100 steel, an AISI H10 tool-steel specimen and abrasive slurries prepared with black silicon carbide (SiC) particles+distilled water. The tangential ( $T$ ) and normal loads were monitored throughout the tests and the results have shown that: (i) the coefficient of friction behavior was independent of the normal force and (ii) both the concentrations of abrasive slurries and the subsequent action of the abrasive wear modes, generally, did not affect the behavior or magnitude of the coefficient of friction.

© 2013 Elsevier Ltd. All rights reserved.

## 1. Introduction

Recently, the micro-scale abrasive wear test has gained large acceptance at universities and research centers and is widely used in studies focusing on the abrasive wear of materials. Fig. 1a [1] presents a schematic diagram of the operating principles of the abrasive wear test, where a rotating ball is forced against the specimen being tested in the presence of an abrasive slurry. There are two main equipment configurations that can be used to conduct this type of test: the “free-ball” and “fixed-ball” configurations. Fig. 1b [2] and 1c [3,4] show examples of these equipment configurations.

The aim of the micro-abrasive wear test is to generate “wear craters” on the specimen being tested. Fig. 2 presents representative images of such craters, together with an indication of the crater diameter ( $b$ ) [5] (Fig. 2a), the crater depth ( $h$ ) (schematic illustration) (Fig. 2b) and the wear volume ( $V$ ) [2] (Fig. 2c).

The diameter of the wear crater is commonly measured by optical microscopy, but other methods are available. For example, computer aided design (CAD) software [3] has been used for this purpose. The crater depth and the wear volume may be

determined as a function of  $b$ , using Eqs. (1) and (2) [6], respectively, where  $R$  is the radius of the ball.

$$h \cong \frac{b^2}{8R} \quad \text{for } b \ll R \quad (1)$$

$$V \cong \frac{\pi b^4}{64R} \quad \text{for } b \ll R \quad (2)$$

Two abrasive wear modes are usually observed on the surface of the worn crater: “grooving abrasion” results when the abrasive particles slide on the specimen (Fig. 3a – [7]), while “rolling abrasion” is observed when the abrasive particles roll on the surface of the specimen (Fig. 3b – [7]). Depending on test conditions, “rolling abrasion” and “grooving abrasion” can occur simultaneously in a given crater [8]. Fig. 3c [1], 3d [8] and 3e [3] presents images of grooving abrasion, rolling abrasion and the simultaneous action of rolling and grooving abrasion, respectively. In a previous work [3],  $A_t$  was defined as the total projected area of the crater and  $A_g$  as the projected area with grooving abrasion. The projected area with rolling abrasion ( $A_r$ ) may be defined as  $A_r = A_t - A_g$  [9].

The micro-abrasive wear test has been applied toward studying the abrasive wear of metallic [2,3,8] and non-metallic [3,4,10] materials where, depending on the equipment configuration, it is possible to apply normal loads ( $N$ ) from 0.01 N [11] to 10 N [12] and ball rotational speeds ( $n$ ) up to 80 rpm [13].

The wear behavior of different materials can be analyzed based on the dimensions of the wear crater and/or on the wear mode.

\* Correspondence address: Centro Universitário da FEI – Fundação Educacional Inaciana “Padre Sabóia de Medeiros”, Departamento de Engenharia Mecânica, Av. Humberto de Alencar Castelo Branco, 3972, São Bernardo do Campo, SP 09850-901, Brazil. Tel.: +55 11 4353 2900; fax: +55 11 4109 5994.

E-mail addresses: [rcamara@fei.edu.br](mailto:rcamara@fei.edu.br), [ronaldo.cozza@fatec.sp.gov.br](mailto:ronaldo.cozza@fatec.sp.gov.br)

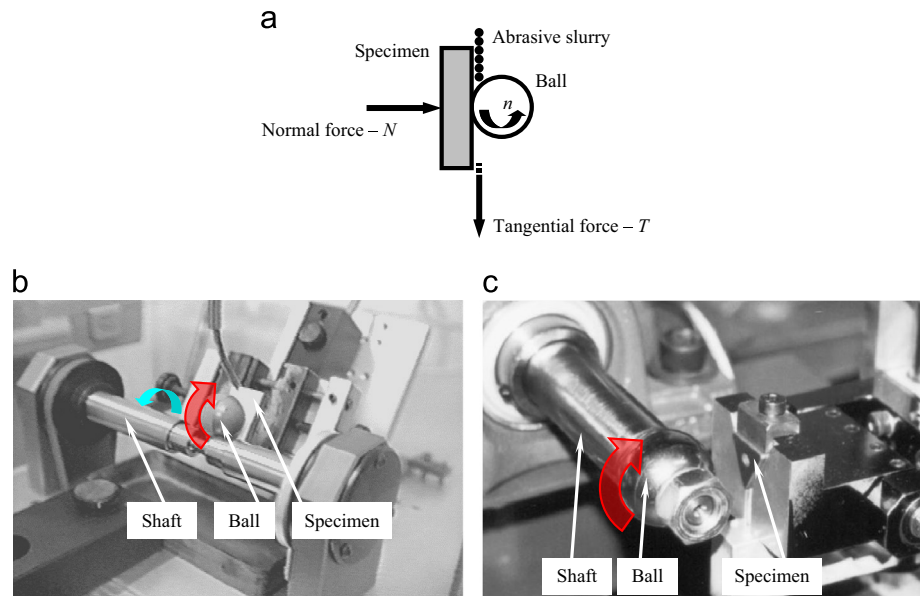
**Nomenclature**

$A_g$	projected area with grooving abrasion [mm <sup>2</sup> ]
$A_r$	projected area with rolling abrasion [mm <sup>2</sup> ]
$A_t$	total projected area of the wear crater [mm <sup>2</sup> ]
$b$	diameter of the wear crater [mm]
$C$	concentration of the abrasive slurry
CAD	Computer Aided Design
$D$	diameter of the ball [mm]
DSRW	Dry Sand Rubber Wheel
DW	distilled water
$h$	depth of the wear crater [mm]
$k$	wear coefficient [mm <sup>3</sup> /(N m)]
$n$	ball rotational speed [rpm]
$N$	normal force [N]
$R$	radius of the ball [mm]
$S$	sliding distance [m]

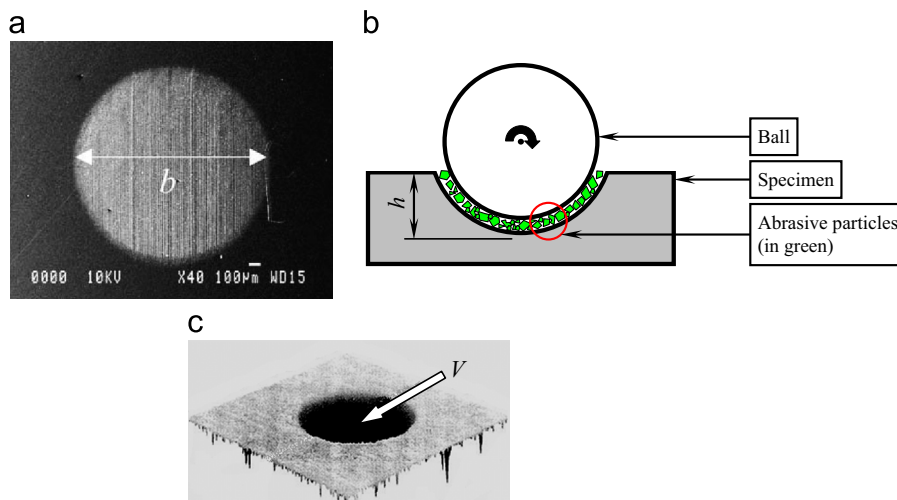
$t$	test time [s]
$T$	tangential force (friction force) [N]
$T_{AS}$	tangential force (friction force) between abrasive particles and specimen [N]
$T_{BA}$	tangential force (friction force) between ball and abrasive particles [N]
$v$	tangential sliding velocity [m/s]
$V$	wear volume (volume of the wear crater) [mm <sup>3</sup> ]

*Greek letters*

$\mu$	coefficient of friction
$\mu_{AS}$	coefficient of friction between abrasive particles and specimen
$\mu_{BA}$	coefficient of friction between ball and abrasive particles



**Fig. 1.** Micro-abrasive wear testing using the rotating ball method: (a) schematic diagram of the operating principle [1], (b) “free-ball” configuration [2] and (c) “fixed-ball” configuration [3,4].



**Fig. 2.** Representative images of wear craters: (a) diameter –  $b$  [5], (b) crater depth –  $h$  (schematic illustration) and (c) wear volume –  $V$  [2].

Download English Version:

<https://daneshyari.com/en/article/7003358>

Download Persian Version:

<https://daneshyari.com/article/7003358>

[Daneshyari.com](https://daneshyari.com)