## Accepted Manuscript

Experimental Investigations and Correlation Development of Convective Heat Transfer in a Rotating Smooth Channel

S.M. Hosseinalipour, H.R. Shahbazian, Bengt Sunden

PII:	S0894-1777(18)30257-7
DOI:	https://doi.org/10.1016/j.expthermflusci.2018.02.020
Reference:	ETF 9381
To appear in:	Experimental Thermal and Fluid Science
Received Date:	10 December 2017
Revised Date:	18 February 2018
Accepted Date:	20 February 2018



Please cite this article as: S.M. Hosseinalipour, H.R. Shahbazian, B. Sunden, Experimental Investigations and Correlation Development of Convective Heat Transfer in a Rotating Smooth Channel, *Experimental Thermal and Fluid Science* (2018), doi: https://doi.org/10.1016/j.expthermflusci.2018.02.020

This is a PDF file of an unedited manuscript that has been accepted for publication. As a service to our customers we are providing this early version of the manuscript. The manuscript will undergo copyediting, typesetting, and review of the resulting proof before it is published in its final form. Please note that during the production process errors may be discovered which could affect the content, and all legal disclaimers that apply to the journal pertain.

## ACCEPTED MANUSCRIPT

### **Experimental Investigations and Correlation Development of Convective Heat Transfer in a Rotating Smooth Channel**

S.M. Hosseinalipour<sup>1</sup><sup>†</sup>, H.R. Shahbazian<sup>1</sup> and Bengt Sunden<sup>2</sup>

<sup>1</sup>Department of Mechanical Engineering, Iran University of Science and Technology, Narmak, Tehran, Iran <sup>2</sup>Department of Energy Sciences, Lund University, Sweden † Corresponding author: Seyed.m.alipour@gmail.com

#### **ABSTRACT:**

An experimental study has been carried out to investigate the effects of Coriolis force and centrifugal buoyancy force on convective heat transfer in a smooth cooling channel under rotating condition in a new rotation facility. The main motivation behind this work is to obtain required data for developing heat transfer correlations based on rotating state. In accordance with the open literature, four parameters, i.e.s Reynolds number, Rotation number, coolant Density Ratio and main flow direction were selected to study the influence on local heat transfer coefficient. Therefore, the investigation considered Reynolds number (Re) ranging from 5,000 to 12,000, Rotation number (Ro) in the range of 0 to 0.15, Inlet Density Ratio (DR) between 0.04 to 0.1 and at the entry developing region of a square smooth channel in a new rotation test bench. Comprehensive experiment have been conducted to collect data on the local Nusselt number at the leading and trailing walls. The results show that the local heat transfer rate is enhanced by rotation on the trailing surface and reduced on the leading surface in outward flow due to symmetrical secondary vortex generated by Coriolis force. The Centrifugal Buoyancy effect is favorable to the heat transfer enhancement on both walls. Also, by utilizing the extensive experimental data, correlations for the local Nusselt number have been updated for the leading and trailing walls in ranges of the mentioned flow parameters, which can be used as an internal cooling boundary condition for numerical analysis of thermal stresses and cooling efficiency maximization for gas turbine blades.

Keywords: Rotating heat transfer, Coriolis force, Centrifugal Buoyancy, Square channel, Correlation development

#### NOMENCLATURE

	A	area (m <sup>2</sup> )	Х	streamwise coordinate
	Bu	rotational Buoyancy number	у	widthwise coordinate
	$C_p$	specific heat (J kg <sup>-1</sup> K <sup>-1</sup> )	Х	span-wise coordinate
	$egin{array}{c} C_p \ D_h \end{array}$	channel hydraulic diameter (m)	V	Voltage (V)
	DR	density ratio		Greek symbols
>	f	friction factor	α, β, γ,	$\theta$ correlation coefficient
	$f_0$	friction factor from Blasius equation	μ	fluid dynamic viscosity (Pa.s)
	h	heat transfer coefficient (W $m^{-2} K^{-1}$ )	ρ	fluid density (kg/m <sup>3</sup> )
	Ι	electrical current	λ	Thermal conductivity (W/m.K)
	L	channel length (m)	Ω	Rotational speed $(r.s^{-1})$
	Nu	local Nusselt number		Subscripts
	$Nu_0$	Nusselt number from Dittus-Boelter equation	amb	Ambient
	$\Delta P$	pressure drop (Pa)	b	Bulk
	Pr	Prandtl number	i	the number of the measured point in x direction
	Q	volumetric Flow Rate (m <sup>3</sup> /hr)	W	wall
	q	heat energy (W)	in	inlet
	r	rotating radius (m)	loss	loss
	Re	Reynolds number	out	Outlet
	Ro	Rotation number		Abbreviations
	Т	temperature (K)	LE	Leading Edge
	$T_w$	air bulk temperature (K)	Stn	Stationary Condition
	$T_b$	wall temperature (K)	TE	Trailing Edge

Download English Version:

# https://daneshyari.com/en/article/7051771

Download Persian Version:

https://daneshyari.com/article/7051771

Daneshyari.com