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Transient stage comparison of Couette flow under step shear stress and step velocity boundary conditions☆

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ABSTRACT

Couette flow has been widely used in many industrial and research processes, such as viscosity measurement. For 15 the study on thixotropic viscosity, step-loading, which includes (1) step shear stress and (2) step velocity conditions, is widely used. Transient stages of Couette flow under both step wall shear stress and step wall velocity 17 conditions were investigated. The relative coefficient of viscosity was proposed to reflect the transient process. 18 Relative coefficients of viscosity, dimensionless velocities and dimensionless development times were derived 19 and calculated numerically. This article quantifies the relative coefficients of viscosity as functions of dimension- 20 less time and step ratios when the boundary is subjected to step changes. As expected, in the absence of step 21 changes, the expressions reduce to being functions of dimensionless time. When step wall shear stresses are 22 imposed, the relative coefficients of viscosity change from the values of the step ratios to their steady-state 23 value of 1. But with step-increasing wall velocities, the relative coefficients of viscosity decrease from positive 24 infinity to 1. The relative coefficients of viscosity increase from negative infinity to 1 under the step-decreasing 25 wall velocity condition. During the very initial stage, the relative coefficients of viscosity under step wall velocity 26 conditions are further from 1 than the one under step wall shear stress conditions but the former reaches 1 faster. 27 Dimensionless development times grow with the step ratio under the step-rising conditions and approach the 28 constant value of 1.785 under the step wall shear stress condition, and 0.537 under the step wall velocity condition respectively. The development times under the imposed step wall shear stress conditions are always larger 30 than the same under the imposed step wall velocity conditions.

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1. Introduction

In fluid dynamics, Couette flow is the laminar flow of a viscous fluid in the space between two parallel plates, one of which is moving while the other one keeps static. The transient start-up of Couette flow between two parallel plates one of which moves suddenly under imposed constant velocity has been investigated by many researchers [1,2]. Bandyopadhyay and Mazumder [3] studied the longitudinal dispersion of passive contaminant in an incompressible fluid between a parallel plate channel oscillating wall velocities. Analytical solutions of transient Couette flows of Newtonian fluid with constant and time-dependent pressure gradients have been proposed by Mendiburu et al. [4]. But the boundary condition was always a constant velocity.

There have been other researches on unsteady Couette flow of non-Newtonian fluid or under special situations. Exact solutions of unsteady

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Couette flows of generalized Maxwell fluid with fractional derivative 51 were obtained by Wang and Xu [5] and Tan et al. [6]. Theoretical analy- 52 sis of the velocity field and stress field of generalized second order fluid 53 was presented by Xu and Tan [7]. Analytic solutions for unsteady unidi- 54 rectional flows of an incompressible second grade fluid with a free sur- 55 face or a moving plate and the associated frictional forces were reported 56 by Hayat et al. [8]. Unsteady Couette flows of a second grade fluid with 57 space dependent viscosity, elasticity and density were investigated by 58 Asghar et al. [9]. Velocity distributions for transient Couette flows of 59 Oldroyed-B fluid between two infinite parallel plates have also been re- 60 ported [10,11]. Lacaze et al. [12] presented solutions to describe the 61 flows of a viscoplastic fluid subjected to constant and sinusoidal moving 62 plate velocities in a relative wide cylindrical Couette device. Solutions 63 for unsteady Couette flow through a porous medium or in a magnetic 64 or electric field have also been reported [13-15]. Unsteady magnetohy- 65 drodynamics (MHD) Couette flow of a third-grade fluid in the presence 66 of pressure gradient and Hall currents was discussed by Azram and 67 Zaman [16]. Unsteady Couette flow through of a viscoelastic fluid 68 through a parallel plate channel filled with a porous medium was stud- 69 ied by Attia et al. [17]. Makinde and Franks [18] studied transient 70

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Nomenclature

distance between the two plates (m) d t time (s) characteristic time, $t^* = d^2/\nu$ (s) ť ĩ dimensionless time, $\tilde{t} = t/t^*$ development time (s) t_d \tilde{t}_d dimensionless development time, $\tilde{t}_d = t_d/t^*$ fluid velocity (m/s) 11 bottom wall velocity (m/s) u_w dimensionless velocity distribution ĩi steady velocity distribution of stage (1) (m/s) u_s velocity after the flow field has developed (m/s) u_d bottom wall velocity at stage (1) (m/s) Uп bottom wall velocity at stage (2) (m/s) И1 U velocity during the developing period (m/s)

vertical spatial coordinate (m)

dimensionless vertical spatial coordinate

Greek symbols

χ

 $\tilde{\chi}$

eigen values β_n step ratio shear rate (s^{-1}) $\dot{\gamma}$ developed shear rate (s^{-1}) $\dot{\gamma}_d$ calculated dynamic viscosity (Pa s) actual dynamic viscosity of fluid (Pa s) μ_a shear stress (Pa) bottom wall shear stress (Pa) τ_w τ_0 bottom wall shear stress at stage(1) (Pa) bottom wall shear stress at stage(2) (Pa) τ_1 developed shear stress (Pa) τ_d eigen values ν kinematic viscosity of fluid (m²/s) relative coefficient of viscosity (dimensionless) φ φ under imposed wall shear stress condition (dimensionless) φ under imposed wall velocity condition (dimensionless) φ_u

reactive MHD Couette flow with temperature dependent viscosity and thermal conductivity. Boundary conditions of all above researches were either constant or time-dependent velocity at the moving wall. Imposed shear stress boundary conditions have never been considered.

There have been some researchers paying their attention on boundary conditions of controlled tangential surface force or torque. Ting [19] discussed unsteady Couette flows of second-order fluids with constant tangential surface force. Considering inertia of rotation, Ravey et al. [20] studied the transient motions of rotor based on an air-bearing viscometer filled with Newtonian fluids. Transient Couette flow of an Oldroyd fluid between two circular cylinders was investigated by Bernardin and Nouar [21]. The flow was induced by applying a constant torque to the inner cylinder with the outer cylinder kept stationary. Asymptotic analysis on acceleration and relaxation process near the inner cylinder was carried out to determine the time for inner cylinder and fluid to reach steady state.

Muzychka and Yovanovich [22] investigated the transient stage of Couette flow between two parallel plates and proposed a compact expression to describe the time-dependent shear stress on the internal surface of moving plate. Emin Erdoğan [2] concluded that the time required to attain the asymptotic values of the skin friction or volume flux of a Couette flow is d^2/ν , where d indicates the gap between the two parallel plates and ν represents the kinematic viscosity of the fluid. There have been several reasonable time scales for Taylor vortex flow to reach fully developed in cylindrical Couette flow, such as $t^*=$

 d^2/ν , where d indicates radial gap [23,24], $t^* = H^2/\nu$, where H indicates 96 gap length [25,26], and a mixed time scale based on both H and d 97 [27,28], $t^* = Hd/\nu$. Another description of viscous time scale incorporating both the gap width and the distance between the endwalls of the 99 system was recommended by Czarny and Lueptow [29].

Couette flow has been used as the fundamental method for the 101 measurement of viscosity [22]. Viscosity of time-dependent viscous 102 fluid, such as thixotropic fluid, needs to be measured and recorded 103 continuously. Step-loadings, either step shear stress or step shear rate 104 are widely used to study the viscosity of thixotropic fluid [30–33]. The 105 velocity distribution is not steady immediately after step-changing of 106 boundary conditions. Only when the velocity gradient of fluid becomes 107 time-invariant, will the measured viscosity represent the actual one.

In this article, transient Couette flow of an incompressible Newtoni- 109 an fluid between two infinite parallel plates subjected to both step wall 110 shear stress and step wall velocity boundary conditions are analyzed. 111 The times needed for the velocity fields to allow for proper viscosity 112 measurements called the "development times" are presented. For quan- 113 titative analysis on transient stage, we define the relative coefficient of 114 viscosity $\varphi(t)$ and dimensionless development time \tilde{t}_d . The potential 115 errors on viscosity measurements under different loading conditions 116 are quantified and compared by examining these two parameters. 117

The remainder of this article is divided into seven sections. The 118 problem analyzed in this article is outlined in the next section. This is 119 followed by the definitions of the dimensionless parameters used to 120 identify the development times. Section 4 presents the governing equa-121 tions, the initial condition and the boundary conditions for the problem 122 described in Section 2. The velocity distributions are then listed in 123 Section 5. The relative coefficient of viscosity and the dimensionless ve-124 locity are then obtained in Section 6. This is followed by discussion of 125 the results. Some concluding remarks are given to conclude the article. 126

2. Problem description

When viscosity is measured in coaxial cylinder viscometer, fluid is 128 kept in concentric cylindrical cylinders. One cylinder is kept stationary 129 while the other rotates. This leads to an axisymmetric velocity distribu- 130 tion. Neglecting end effects, the velocity is a function of the radial coordi- 131 nate. For the study on thixotropic viscosity, step-loading is widely used 132 which includes (1) step shear stress and (2) step shear rate conditions. 133 In a rheometer, shear rate is imposed by setting the rotational speeds 134 of the measuring system [34]. So imposed shear rate condition can be an- 135 alyzed as the imposed velocity condition. The flow in the annulus can be 136 modeled using flow between two infinite parallel plates as the ratio of 137 the annulus gap between inner and outer cylinders to the radius of the 138 inner cylinder is small. The remainder of this article presents analyses 139 of flow between two flat plates used to calculate viscosity.

Fig. 1 shows the schematic of the problems considered in this article. 141 Couette flows between two parallel plates separated by a distance d are 142 analyzed. Transient laminar flows of an incompressible Newtonian fluid 143 are induced by imposing different conditions on the bottom wall while 144 the top wall remains stationary. The fluid is restricted to flow parallel to 145 the two plates leading to a velocity field which is a function of vertical 146 spatial coordinate x and time t. The flow is steady under a specific wall 147 shear stress τ_0 ($\tau_0 \neq 0$) or velocity u_0 ($u_0 \neq 0$) in stage (1) and the velocity 148 distribution could be described by

$$u_s(x) = \frac{\tau_0}{\mu}(d - x) \tag{1}$$

under the imposed wall shear stress condition and 151

$$u_s(x) = \frac{u_0}{d}(d-x) \tag{2}$$

under the imposed wall velocity condition. Here u_s is the steady velocity of stage (1) and also the initial velocity of stage (2). At the beginning of 152

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