

Chinese Society of Aeronautics and Astronautics & Beihang University

## **Chinese Journal of Aeronautics**

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JOURNAL 

# Variable load failure mechanism for high-speed load sensing electro-hydrostatic actuator pump of aircraft

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Received 14 March 2017; revised 7 June 2017; accepted 30 September 2017

#### **KEYWORDS**

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Coupling lubrication model; 13 Electro-Hydrostatic Actua-14 15 tor (EHA); 16 High-speed pump; 17 Partial abrasion; Slipper pair; 18 Variable load 19

Abstract This paper presents a novel transient lubrication model for the analysis of the variable load failure mechanism of high-speed pump used in Load Sensing Electro-Hydrostatic Actuator (LS-EHA). Focusing on the slipper/swashplate pair partial abrasion, which is considered as the dominant failure mode in the high-speed condition, slipper dynamic models are established. A forth sliding motion of the slipper on the swashplate surface is presented under the fact that the slipper center of mass will rotate around the center of piston ball when the swashplate angle is dynamically adjusted. Besides, extra inertial tilting moments will be produced for the slipper based on the theorem on translation of force, which will increase rapidly when LS-EHA pump operates under highspeed condition. Then, a dynamic lubricating model coupling with fluid film thickness field, temperature field and pressure field is proposed. The deformation effects caused by thermal deflection and hydrostatic pressure are considered. A numerical simulation model is established to validate the effectiveness and accuracy of the proposed model. Finally, based on the load spectrum of aircraft flight profile, the variable load conditions and the oil film characteristics are analyzed, and series of variable load rules of oil film thickness with variable speed/variable pressure/variable displacement are concluded.

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More Electric Aircraft (MEA) is the future development trend

for general aircraft, which partially replace the conventional

central hydraulic system by local electrically Power-By-Wire

(PBW) system. The successful application of PBW technology

brings less energy loss and higher efficiency. Among them,

Electro-Hydrostatic Actuator (EHA) is the key component

of the PBW system, which has already been applied in the large

civil aircraft, such as A380 and A350.1-3 However, because of

#### 1. Introduction

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ELSEVIER Production and hosting by Elsevier

#### https://doi.org/10.1016/j.cja.2018.01.005

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Please cite this article in press as: SHI C et al. Variable load failure mechanism for high-speed load sensing electro-hydrostatic actuator pump of aircraft, Chin J Aeronaut (2018), https://doi.org/10.1016/j.cja.2018.01.005

### **ARTICLE IN PRESS**

### Nomenclature

- $v_{\rm p}, a_{\rm p}$  linear reciprocating motion velocity and axial acceleration
- $r, \theta$  polar radius and polar angle
- $F_g$  slipper gravity
- $F_{sa}$  slipper inertial force of linear reciprocating motion  $F_{a_{A\delta}}$  slipper inertial force of sliding motion
- $M_{g_{\Lambda}}, M_{g_{V}}$  extra tilt moments produced by slipper Gravity
- $M_{gx}, M_{gy}$  extra tilt moments produced by slipper Gravity  $M_{sax}, M_{say}$  extra tilt moments produced by slipper Inertial force of linear reciprocating motion
- $M_{a_{\Delta\delta}x}, M_{a_{\Delta\delta}x}$  extra tilt moments produced by slipper Inertial force of sliding motion
- $F_{Ts}(p,h)$  friction force between the slipper and swash plate
- $T_{\rm s}, T_{\rm tx}, T_{\rm ty}$  friction moments between the piston and the slipper socket along the x-, y- and z-
- $f_{\rm s}$  friction coefficient of piston ball

- $F_1$  bearing force between the slipper and the swash plate along *z*-axis
- $M_2, M_3, M_4$  anti-overturning torques of the slipper on x-, y- and z-
- $\rho$  oil density
- $\mu_0$  oil viscosity at the initial condition
- $\alpha_{\rm p}, \alpha_{\rm T}$  pressure coefficient, temperature coefficient
- $T_0$  reference temperature
- $d_1, l_1$  length and diameter of the damping orifices of piston
- $d_2, l_2$  length and diameter of the damping orifices of slipper
- $\dot{\mathbf{h}}^{(k)}$  oil film thickness shifting rates vector in kth iteration
- $\mathbf{F}'(\dot{\mathbf{h}})$  jacobi matrix of  $\mathbf{F}(\dot{\mathbf{h}})$
- $\mathbf{E}_k$  diagonal matrix to revise Newton iterative method

the heating problem, they are just used as standby systems.<sup>4</sup> To 30 solve the motor heating problem in EHA, a Load Sensing 31 Electro-Hydrostatic Actuator (LS-EHA) scheme has been pro-32 33 posed, which can solve the heating problem and the dynamic problem simultaneously. The schematic diagram of the LS-34 EHA is shown in Fig. 1. When the LS-EHA works in high load 35 and slow rate conditions, the pump displacement is reduced 36 37 through decreasing the angle of the swashplate, and the motor speed is improved at the same time to maintain a stable output 38 of the actuator. Hence, the armature current will also be 39 decreased due to the reduction of the motor torque. Conse-40 41 quently, the energy loss can be effectively reduced and the system heating can be limited greatly.<sup>5,6</sup> Based on the insightful 42 advantages, LS-EHA should be the future development trend 43 for the next generation of actuation system of aircraft. 44

45 The LS-EHA system consists of a brushless DC motor, a LS-EHA pump, a Load Sensing Mechanism (LSM), etc. 46 47 Among them, the LS-EHA pump plays a core role for convert-48 ing mechanical power into hydraulic power, which directly determines the service life and reliability of the LS-EHA. 49 Fig. 2 shows the cross section of the high-speed pump used 50 51 in the LS-EHA system, which comprises nine pistons mounted within the cylinder at an equal angular interval around the cen-52



Fig. 2 Cross section of high-speed pump used in LS-EHA.

terline of the cylinder, the cylinder is pushed towards the fixed 53 valve plate by a compressed cylinder spring, and the com-54



Fig. 1 Schematic of LS-EHA system.

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