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Experimental investigation of surface roughness 4 effects on flow behavior and heat transfer characteristics for circular microchannels

Yuan Xing, Tao Zhi, Li Haiwang^{*}, Tian Yitu 7

National Key Laboratory of Science and Technology on Aero-Engine, Beihang University, Beijing 100191, China 8 Collaborative Innovation Center for Advanced Aero-Engine of China of Aerodynamics, Beihang University, Beijing 100191, China 0

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Abstract This paper experimentally investigates the effect of surface roughness on flow and heat transfer characteristics in circular microchannels. All test pieces include 44 identical, parallel circular microchannels with diameters of 0.4 mm and 10 mm in length. The surface roughness of the microchannels is $R_a = 0.86, 0.92, 1.02 \,\mu\text{m}$, and the Reynolds number ranges from 150 to 2800. Results show that the surface roughness of the circular microchannels has remarkable effects on the performance of flow behavior and heat transfer. It is found that the Poiseuille and Nusselt numbers are higher when the relative surface roughness is larger. For flow behavior, the friction factor increases consistently with the increasing Reynolds number, and it is larger than the constant theoretical value for macrochannels. The Reynolds number for the transition from laminar to turbulent flow is about 1500, which is lower than the value for macrochannels. For the heat transfer property, Nusselt number also increases with increasing Reynolds number, and larger roughness contributes to higher Nusselt number.

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With continuing development of aircraft engines, turbine inlet

temperatures become increasingly higher, often far exceeding

the melting point of turbine blade materials. Efficient cooling

techniques are among the most important methods to ensure

safe operation of turbines. Many cooling techniques, e.g. film

cooling and impingement cooling, have been applied to aero

engines. Microchannels, which have superior heat transfer

characteristics with higher surface to volume ratios, are

attracting more and more attention for application in cooling

techniques. After the landmark work of Tuckerman and

1. Introduction

Corresponding author at: National Key Laboratory of Science and Technology on Aero-Engine, Beihang University, Beijing 100191, China

E-mail address: 19820912@sina.com (H. Li).

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Pease,¹ many researchers in the last decade have investigated the flow and heat transfer behavior of microchannels (< 1 mm), which differ from those in the macro-scale.

Early in their investigation, Peng et al.^{2,3} experimented with 35 the behaviors of flowing fluid and heat transfer in rectangular 36 microchannels with diameters ranging from 0.1 mm to 0.3 mm. 37 Water was used as the working fluid. It was noted that the 38 overall hydrodynamic performance of microchannels was dif-39 ferent from conventional theories. Based on their experimental 40 results, the friction factor in laminar and turbulence flow 41 regimes was inversely proportional to $Re^{1.98}$ and to $Re^{1.72}$, 42 respectively. This is in stark contrast to what is expected from 43 44 the conventional theories on laminar and turbulent regimes according to Filonenko.⁴ What's more, the authors found a 45 transition occurred at Reynolds numbers 300-700, which was 46 47 considered smaller than the transitional critical Reynolds number 2300. 48

Mala and Li⁵ completed experiments in microtubes with 49 50 diameters ranging from 50 µm to 254 µm. The results also indicated departure of flow characteristics from conventional the-51 ory of microtubes. In their experiment, the friction factor in 52 laminar regime is higher than that predicted by conventional 53 theory. The experimental results indicated the transition from 54 laminar to turbulent flow mode at Reynolds numbers between 55 300 and 900. Yang and Lin⁶ investigated the heat transfer and 56 57 friction characteristics of water flow in microtubes. Experi-58 mental results reveal that there is no significant size effect for 59 water flow in tubes with diameters ranging from 123 µm to 962 µm. Zhao et al.⁷ investigated the characteristics of nitrogen 60 flow in microtubes with diameters of 2.05, 5.03, and 10.10 μ m. 61 The results indicated that the flow characteristics had signifi-62 cant discrepancies between the experimental data and predic-63 tions from the classical Poiseuille's theory. Lelea et al.⁸ 64 found that the conventional theories were applicable in lami-65 nar flow regimes. Hasan et al.⁹ employed numerical simulation 66 67 and concluded that Reynolds number, thermal conductivity ratio and hydraulic diameter all affect the behavior of axial 68 69 heat conduction. The authors explained that porous fin 70 enhanced heat transfer performance along with the mechanism 71 of water in microchannels reducing the pressure drop it affects.

72 As indicated above, most investigations show that thermal 73 and hydrodynamic performance in microchannels is different from that of macrochannels. However, researchers have 74 reached various conclusions about the cause of this divergence. 75 Some investigators consider that the discrepancies might be 76 caused by factors that are ignored in macrochannels. One of 77 78 the most important of these, surface roughness, has attracted more and more attention. Tang et al.¹⁰ investigated flow char-79 acteristics for nitrogen and helium in stainless steel microtubes, 80 fused silica microtubes and fused silica square microchannels. 81 The data in fused silica microtubes (D_h range: 50–201 µm, D_h 82 is hydraulic diameter) are consistent with conventional predic-83 84 tions; however, the friction factors in stainless steel tubes $(D_{\rm h})$ 85 range: 119-300 µm) are much higher than theoretical values. 86 The authors believed that such significant deviations can be attributed to large surface relative roughness. Yang et al.¹¹ 87 investigated the pressure drop and heat transfer performance 88 of air flow in microtubes with inner diameters of 86, 308, 89 and 920 µm. The surface roughness of the microtubes is 90 $R_a = 0.704, 0.685, 0.135 \,\mu\text{m}$, respectively, and are all less than 91 1.5% of the diameter. The experimental friction factor shows a 92 good agreement with the conventional theory. Lorenzini et al.¹² 93

later conducted an investigation about compressible flow of 94 nitrogen through circular microchannels from 26 µm to 95 508 µm with different surface roughness parameters. They 96 found that, for both smooth and rough microtubes, the fric-97 tion factor agrees well with conventional theory. It should be 98 noted that when the Reynolds number was larger than 1300, 99 the friction factor of smaller microchannels ($< 100 \mu m$) varies 100 from the Poiseuille law. Liu et al.¹³ studied the flow behaviors 101 for air flow in rectangular microchannels with relative rough-102 ness of $R_a = 0.58, 0.82, 1.26$. The experimental results indi-103 cated that a larger roughness tends toward larger Poiseuille 104 numbers; thus, the effect of roughness cannot be ignored in 105 experiments. Kharati-Koopaee and Zare¹⁴ numerically studied 106 the flow and heat transfer characteristics of air and water with 107 aligned and offset roughness patterns in rectangular 108 microchannels. The results indicated that the offset arrange-109 ment leads to lower pressure loss for both fluids and also a 110 lower heat transfer rate for water than the aligned pattern. It 111 should be noted that both roughness patterns contribute to 112 better thermal performance. Zhang et al.¹⁵ numerically investi-113 gated gas slip flow characteristics affected by rough surfaces. 114 The results revealed that gas flow behavior in rough 115 microchannels is affected by the statistical roughness height 116 and rarefaction. Kandlikar et al.¹⁶ investigated heat transfer 117 behaviors for microtubes of different diameters, 0.62 mm and 118 1.032 mm, and roughness ranging from $R_a = 1.0 \,\mu\text{m}$ to 119 $R_a = 3.0 \,\mu\text{m}$. This study finds that relative surface roughness 120 bears no or little effect on the heat transfer characteristics 121 for larger-diameter cases. However, the roughness effect is sig-122 nificant for the smaller diameters. Lin et al.¹⁷ studied the effect 123 of roughness on flowing fluid and heat transfer performance of 124 air and CO₂ for microtubes with a 1 mm diameter. In the 125 experiment, four different roughness features were generated. 126 The results showed that the effect of roughness is different in 127 laminar and turbulence flow regimes: in laminar flow, there 128 was no difference of heat transfer between smooth and rough 129 channels. However, in turbulent flow, rough channels have 130 better thermal behavior. Koo and Kleinstreuer¹⁸ numerically 131 investigated the effect of surface roughness on heat transfer 132 in micro-channels. They concluded that the Nusselt number 133 increases with the increase in relative surface roughness in lam-134 inar flow. For turbulent flow, the Nusselt number becomes 135 higher when the relative surface roughness of the tubes was lar-136 ger. Guo et al.¹⁹ built a model to investigate the influence of 137 wall roughness on fluid flow and heat transfer in microchan-138 nels. The results showed that roughness plays a positive role 139 in thermal performance as well as flow resistance. 140

Although the effects of roughness on flow and heat transfer behavior of microchannels have been studied by some, there is a lack of research on the behavior of roughness in undeveloped sections with air. What's more, owing to the difficulties in measuring pressure, temperature and roughness, data about flow and heat transfer behavior in circular microchannels are lacking. Thus, considering the applications for turbine blades, heat transfer and flow behavior of air flow affected by surface roughness in circular microchannels are examined in this paper. 141

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Experimental and theoretical investigation of the flow and heat transfer behavior in circular microchannels of 0.4 mm diameter and 10 mm length with various surface roughness follows. Based on experimental data, the corresponding empiric equations for Poiseuille and Nusselt numbers were developed. Download English Version:

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