



# Numerical investigation of performance, combustion and emission characteristics of various biofuels



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## ABSTRACT

Alternative fuels have been a major concern in today's era due to its promising characteristics in internal combustion engines. The computational analysis is a proven method by many researchers in determining combustion, performance and emission characteristics of an internal combustion engine, offering accurate results at par with experimental results. An attempt is being made by the author(s) in this paper for performing numerical study of nine (9) different alternative biofuels and pure diesel. Comparison has been carried out with the results of pure diesel in a single cylinder, naturally aspirated, water cooled, direct injection diesel engine with different engine load (20%, 40%, 60%, 80% and 100%), constant compression ratio (CR17.5), higher nozzle opening pressure (220 bar) and advance injection timing (23° CA before TDC). The study was performed after validating results of two experimental data using the proposed tool, which have shown that the numerical results were in good agreement with the experimental results. The numerical results of the different biofuels have depicted the possibility of using the fuels as alternative fuel for internal combustion engine.

## 1. Introduction

The increase in consumption is the reason for the increase in pollution and hence significant progress in the field of internal combustion engine research were made. Many researchers have used various biofuels as alternative and renewable source to determine the performance, emission and combustion characteristics of internal combustion engines. Pyrolysis is performed on waste plastics to obtain oil and the same is mixed with 5% and 10% with diethyl ether on single cylinder water cooled direct injection (DI) engine. The blended fuel (waste plastic and DEE) have improved the cetane rating of the fuel. It was seen that there were significant reduction in the smoke levels while using blended fuel as compared to pure waste plastic oil. The brake thermal efficiency (BTE) has increased and pollutants such as CO and NO<sub>x</sub> were decreased. Addition of oxygenates have improved the process of combustion while reducing emissions. Another study have investigated an engine fed with eight (8) different renewable diesel fuels which includes pure diesel, jet fuel, traditionally derived biodiesel (FAME), deoxygenated canola derived fatty acids (DCFA), DFCA with varying H<sub>2</sub>, continuous DCFA, deoxygenated lauric acid (DLA) and isomerize deoxygenated canola derived fatty acid alkanes. While diesel, jet fuel and FAME were used as benchmark fuels for the other new types of renewable fuels. The results have indicated lower mechanical

efficiency but increased BTE of the renewable fuels. The combustion analysis indicated short ignition delays, low peak in cylinder pressures, reduced rate in increase in cylinder pressure and low heat release rate for the proposed renewable fuels. They have an added advantage of reduction in NO<sub>x</sub>, soot and greenhouse gas emission [1,2]. Waste tire pyrolysis oil (WTPO) blended with diesel fuel is proposed to be used as fuel for investigating the performance and emission characteristics of a four stroke, four cylinder, naturally aspirated, DI diesel engine. The results shows that WTPO blended with diesel can be used as a potential fuel since it has similar characteristics like torque and power output as compared to diesel, without much modification in the engine [3,52]. Various additives were also added to improve the biodiesel blends to reduce fuel consumption and NO emission from diesel engines, such as 2-ethylhexyl nitrate (EHN), di-tertiary-butyl peroxide (DTBP) and pentanol. The additives have improved the combustion pressure (0.11–0.53 bar) and lower heat release rate (0.82–2.29 J/°CA) [4]. Some studies on diesel engine using higher alcohol/diesel fuel blend shows that without any modification, a diesel engine can run properly on 30% 1-butanol/70% diesel fuel or 25% 1-pentanol/75% diesel fuel [5]. Use of diesel-aegle marmelos oil-diethyl ether blends on various blends has been performed. 60:30:10 (diesel: aegle marmelos: diethyl ether) have increased the BTE by 4.3%, NO<sub>x</sub> emission have been reduced by 3.9% at compression ratio (CR) 17.5 and full load conditions

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**Nomenclature**

$A_0, A_1, A_2$	empirical factors	SD	standard deviation
A/F	air/fuel	RME	rapeseed methyl ester
BMEP	brake mean effective pressure (bar)	SFC	specific fuel consumption (g/kWh)
BN	Bosch number	SME	soybean methyl ester
BSN	Bosch smoke number	SOI	start of injection (degree)
BT	brake power (kW)	SFC	specific fuel consumption (kg/kWh)
BTE	brake thermal efficiency (%)	SFR	soot formation rate (1/deg)
$b_m$	depth of the spray forward front (m)	SS	speed sensor
CA	crank angle (degrees)	STP	spray tip penetration (mm)
CAE	crank angle encoder	$V_p$	mean piston velocity (m/s)
CI	compression ignition	$\dot{S}_g$	net generation rate of the <i>i</i> th species (kg/s)
CN	cetane number of fuel	T	temperature (K)
CPP	cylinder peak pressure (bar)	TDC	top dead centre
CR	compression ratio	TIS	temperature indicator sensor
CPT	cylinder peak temperature (K)	$T_b$	temperature in a burnt gas zone (K)
CSOBD	cotton seed oil biodiesel	TME	tallow methyl ester
DI	direct injection	V	volume of cylinder ( $\text{cm}^3$ )
ED	error deviation (%)	$V_1$	current velocity of the EFM (m/s)
EGR	exhaust gas recirculation	$V_0$	initial velocity of the EFM at the nozzle of the injector (m/s)
EGT	exhaust gas temperature (K)	$V_m$	fuel spray evaluation process in a medium speed diesel engine (m/s)
EEFO	ethyl ester fish oil	$V_k$	swept volume ( $\text{cm}^3$ )
$E_a$	apparent activation energy for the auto ignition process (kJ/kmole)	$V_i$ & $V_c$	cylinder volumes at injection timing and top dead centre ( $\text{cm}^3$ )
FMEP	friction mean effective pressure (bar)	x	fraction of fuel burnt
HSL	Hartridge Smoke Level	$X_0$	fraction of burnt fuel during ignition delay
$h_{wfr}$	height of the NWF forward front	$Y_i$	mass fraction
JME	jatropha methyl ester	$[N_2]_e$	equilibrium concentrations of an molecular nitrogen
$K_T$	evaporation constant	$[NO]_e$	equilibrium concentrations of an oxide of nitrogen
l	current distance between the injector's nozzle and the location of the EFM (m)	$[O_2]_e$	equilibrium concentrations of molecular oxygen
LCS	load cell sensor	$[O_2]_e$	equilibrium concentrations of atomic oxygen
LFR	liquid flow rate	$r_{H_2O}$	volume fraction of water vapor in a combustion chamber
LME	linseed methyl ester	$Y_i^{cy1}$	stoichiometric coefficients on the reactant side
$L_c$	cycle work done (kJ)	$Y_i^j$	stoichiometric coefficients on the product side
$l_m$	EFM's penetration distance (m)	$\alpha, \beta, \lambda$	constants
MAOME	microalgae oil methyl esters	$\alpha_1$	air-fuel equivalence ratio
m	total mass (kg)	$\tau$	time (second)
$m_f$	fuel mass per cycle (kg/h)	$\tau_k$	travel time for the EFM to reach a distance l from the injector's nozzle
NOP	nozzle opening pressure (bar)	$\rho$	density ( $\text{kg}/\text{m}^3$ )
NWF	near wall flow	$\nu$	specific volume ( $\text{m}^3/\text{kg}$ )
n	engine speed (rpm)	$\phi$	crank angle (degree)
P	pressure (bar)	$\omega$	angular crank velocity (rpm)
PD	pure diesel	$\varepsilon$	compression ratio
PM	particular matter (g/kWh)	$\xi_b$	cylinder air charge usage efficiency
POU	percentage of uncertainty (%)	$\sigma_{ud}, \sigma_u$	fuel fractions evaporated during ignition delay period and up
PSBD	palm stearin biodiesel	$\Omega_i$	molar rate of production (mol/s)
PTS	pressure transducer sensor	$\frac{dt}{dx}$	heat release rate (J/deg)
$P_{max}$	maximum cylinder pressure (bar)	VCR	variable compression ratio
$P_b$	brake power (kW)	VE	volumetric efficiency (%)
$q_c$	cycle fuel mass (kg)		
R	gas constant (J/(mol·k))		
SE	standard error		

[6]. A diesel ethanol blend D95E5 (95% diesel and 5% ethanol) have been found to have good potential for use as fuel for diesel engines. The results emphasized the potential of ethanol in reducing  $\text{NO}_x$  emission, smoke capacity and brake specific energy consumption (BSEC) [7].

Vegetable oil such as pongamia biodiesel was also evaluated as an alternative fuel in the transport, power and agricultural sector. Pongamia biodiesel have reduced CO emission by 8.2%, HC emission by 8.9%, BSFC increased by 4.2% and the thermal efficiency is reduced by 2.4% [8]. Dual fuel mode is used on a single cylinder diesel engine with

fuel biodiesel/n-butanol, biodiesel/ethanol and biodiesel/2,5-dimethylfuran. Then results indicated that 20% blend ratio have overall advantages as compared to other blends and diesel. Fatty acid methyl ester (FAME) using Jatropha curcas oil (JCO) and methanol through homogenous catalyst NaOH and KOH is investigated. Thermo-gravimetric and differential thermal analysis (TGDTA) is performed on the blends. It was observed that JCO have the highest lowest activation energy (49.97 kJ/mol) and thus suitable for combustion. Cotton seed oil as a possible fuel on a diesel engine is analyzed and it was found that

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