

Optimisation of environmental gas cleaning routes for solid wastes cogeneration systems. Part II – Analysis of waste incineration combined gas/steam cycle

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Received 5 May 2006; received in revised form 11 December 2006; accepted 16 July 2007

Available online 4 September 2007

Abstract

In the first paper of this paper (Part I), conditions were presented for the gas cleaning technological route for environmental optimisation of a cogeneration system based in a thermal cycle with municipal solid waste incineration. In this second part, an environmental analysis is presented of a cogeneration system comprising a combined cycle composed of a gas cycle burning natural gas with a heat recovery steam generator with no supplementary burning and a steam cycle burning municipal solid wastes (MSW) to which will be added a pure back pressure steam turbine (another one) of pure condensation. This analysis aims to select, concerning some scenarios, the best atmospheric pollutant emission control routes (rc) according to the investment cost minimisation, operation and social damage criteria. In this study, a comparison is also performed with the results obtained in the Case Study presented in Part I.

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Keywords: Gas cleaning routes; Municipal solid wastes; Cogeneration system; Combined cycle; Optimisation; Environmental analysis

1. Introduction

This paper focuses on the search for the optimum exhaust gases cleaning routes for cogeneration systems with municipal solid wastes burning by environmental optimisation modelling. In Part I, the question was settled of using municipal solid waste in energy generation, as well as describing the main cogeneration systems configurations with that energetic source available in the literature. Afterward, the environmental formulation was established and the conditions analysed for optimisation of the exhaust gases cleaning routes for a cogeneration thermal central plant based on the steam cycle.

In this Part II, an environmental model is presented to perform the selection of the control route (rc) that presents

the smaller sum of yearly environment costs (that is, environment control cost + social damage cost) based on a combined cycle with commercial gas turbines in the topping cycle and an incineration unit integrated with a conventional steam generator and steam turbines in the bottoming cycle. The basis of such model is the thermoeconomic and environmental model developed by Frangopoulos [1–4] that is associated with the use of integer programming for help in ranking the best control routes. LINGO 7.0 [5], a well recognised optimisation software, was also used for performing this analysis.

2. Case Study 2: combined cycle

The flow and functional diagrams for this second case study are found in Figs. 1 and 2, respectively. For the thermodynamic analysis, the data of Table 1, Part I will be used. Also, as part of this second environmental analysis,

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Nomenclature

W_{pump}	pump consumed power	$\text{pf}_{\text{BH}}^{(\text{PM})}$	final mass flow (after BH reduction)
E_{pST}	energy produced by steam turbine	$\text{pf}_{\text{EP}}^{(\text{PM})}$	final mass flow (after EP reduction)
E_{r}	energy required by industrial site	$\text{pf}_{\text{SCR}}^{(\text{NO}_x)}$	final NO_x mass flow (after SCR control)
y_{ij}	j element of entry for unit I	$\text{pf}_{\text{SNCR}}^{(\text{NO}_x)}$	final NO_x mass flow (after SNCR control)
y_{ik}	k element of exit for unit i	b	binary variation
m_{v}	steam flow	Z	cost of investment
δ	reduction/control efficacy	FRC	capital recovery factor
$C^{\text{contr.unit.}}$	cost of unitarian control (or operational cost)	t_{m}	maintenance rate
$Z^{\text{anualizado}}$	yearly capital cost	$\$_{\text{RSM}}$	price of MSW
$\Gamma_{o,j}$	benefit of function $y_{o,j}$	TAFS	yearly time of functioning in seconds
$\Gamma_{o,k}$	cost of $y_{o,k}$ (k element of environmental resource going inside unit i)	c_{elcp}	price of electricity acquired from concessionaire
$\Gamma^{\text{op.anual}}$	cost of operation	c_{elvd}	price of electricity sold for industrial site
$\Gamma^{\text{ext.anual(pols)}}$	pollutant emission yearly external cost (SO_2 , PM and NO_x)	TAFH	yearly time of functioning in hours
Ψ	specific energy	$\$(\text{pols})$	price of environmental rate/externality for emissions (SO_2 , PM, NO_x)
v	specific volume	F	objective function
P	pressure		
m_{RSM}	MSW mass flow		
LHV _{RSM}	MSW lower heating value		
h	enthalpy		
E_{PTOTAL}	electric energy produced by cogeneration central		
$\eta_{\text{M(ST)}}$	steam turbine mechanical efficacy		
$\eta_{\text{M(pump)}}$	pump mechanical efficacy		
s	entropy		
s_{o}	water entropy in pressure and temperature of reference		
T_{o}	temperature of reference		
h_{o}	enthalpy of water in pressure and temperature of reference		
$f_{\text{e}}^{(\text{pols})}$	factor of atmospheric pollutant emission for MSW burning		
$\text{pi}^{(\text{pols})}$	atmospheric pollutant initial mass flow (prior to lessening in control routes)		
$\text{pf}_{\text{SD}}^{(\text{SO}_2)}$	final SO_2 mass flow (after SD reduction)		
$\text{pf}_{\text{WS}}^{(\text{SO}_2)}$	final SO_2 mass flow (after WS reduction)		

Subscripts

i unit
 $i = 0$ (environment)

Superscript

pols atmospheric pollutants (SO_2 , PM, NO_x)

Abbreviations

BH bag-house
 EP electrostatic precipitator
 HRSG heat recovery steam generator
 LHV lower heating value
 MSW municipal solid waste
 PM particulate matter
 SCR selective catalytic reactor
 SD spray drier
 SNCR selective non-catalytic reactor
 ST steam turbine
 WS wet scrubber

several gas turbines were considered for the analysis, but only Nuovo Pignone PGT2 was taken for the analysis [6]. Some data were drawn from manufacturer's catalogues, aiming to extract some important parameters to be used in the analysis [7].

Gas turbine selected: Nuovo Pignone PGT2

$$m_{\text{NG}} = 0.16 \text{ kg/s}$$

$$m_{\text{gases}} = 10.19 \text{ kg/s}$$

$m_{\text{v(NG)}} = 5.1$, $t/h = 1.42 \text{ kg/s}$ (for the system that works at $P = 4.2 \text{ MPa}$ and $T = 400 \text{ }^\circ\text{C}$) Exhaust temperature (T_4) = $550 \text{ }^\circ\text{C}$.

Heat rate = $14,401 \text{ kJ/kW h}$.

The set of decision variables herein adopted is the same as Case Study 1 with only m_{v} for $m_{\text{v(MSW)}}$ and m_{p} (whose range of values remains the same, that is, $1.0 \leq m_{\text{v(MSW)}} \leq 1.42$ and $1 \leq m_{\text{p}} \leq 1.42$) being changed.

$$\begin{aligned} \text{Min } F = & Z_{\text{GT}}^{\text{anualizado}} + Z_{\text{HRSG}}^{\text{anualizado}} + Z_{\text{boiler}}^{\text{anualizado}} + Z_{\text{ST1}}^{\text{anualizado}} \\ & + Z_{\text{ST2}}^{\text{anualizado}} + Z_{\text{condenser}}^{\text{anualizado}} + Z_{\text{pump}}^{\text{anualizado}} + \Gamma_{0.1} \\ & + \Gamma_{0.2} - \Gamma_{0.1} - \Gamma_{0.2} + \sum_{n=1}^8 Z_{\text{rcn}}^{\text{anualizado}} \cdot b_{\text{rcn}} \\ & + \sum_{n=1}^8 \Gamma_{\text{rcn}}^{\text{op.anual}} \cdot b_{\text{rcn}} + \sum_{n=1}^8 \Gamma_{\text{rcn}}^{\text{ext.anual(pols=SO}_2, \text{PM, NO}_x)} \cdot b_{\text{rcn}} \end{aligned} \quad (1)$$

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