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### Long-term performance of air-side heat transfer and pressure drop for finned tube evaporators of air conditioners under intermittent operation conditions

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#### ABSTRACT

In this study, the effects of long-term intermittent operations on the air-side heat transfer and pressure drop performance of finned tube evaporators of air conditioners were investigated by experiments on an aluminum-fin evaporator and a copper-fin evaporator. In order to simulate intermittent operations of on–off controlled air conditioners, the temperatures of the two evaporators changed to  $5\pm0.5\,^{\circ}\text{C}$  at first and then to  $27\pm0.5\,^{\circ}\text{C}$  repeatedly. The repetition number was up to 4800, and the air-side heat transfer and pressure drop of the two evaporators were tested after every 300 repetitions. The test results indicate that after long-term intermittent operations, the air-side heat transfer coefficient decreases and the pressure drop increases. The variations of the heat transfer coefficient and the pressure drop are more obvious at lower inlet air velocity, and the influence of long-term intermittent operations on the aluminum-fin evaporator is greater than that on the copper-fin evaporator.

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# Evaporateurs à tubes ailetés des systèmes de conditionnement d'air sous des conditions de fonctionnement intermittent : performance dans le long terme sous l'angle du transfert de chaleur côté air et de la chute de pression

Mots clés : Conditionnement d'air ; Évaporateur ; Tube aileté ; Expérimentation ; Transfert de chaleur ; Chute de pression ; Paramètre

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Nomenclature
                                                                                         maximum velocity across heat exchanger (m s<sup>-1</sup>)
                                                                              u_{\text{max}}
                                                                                         overall heat transfer coefficient (W m<sup>-2</sup> K<sup>-1</sup>)
                                                                             U_{o.w}
Α
           area (m<sup>2</sup>)
                                                                                         thickness of tube wall (m)
                                                                             \chi_{p}
b'
           slop of the air saturation curve
                                                                                         thickness of condensate water film (m)
                                                                             y_{\rm w}
C_p
           specific heat (J Kg<sup>-1</sup> K<sup>-1</sup>)
D
           diameter (m)
                                                                             Greek symbols
D_c
           fin collar outside diameter (m)
                                                                                         wet fin efficiency
                                                                             \eta_{f.wet}
           friction factor
                                                                              \eta_{f,\text{fullywet}} fin efficiency of a fully wet fin
f
                                                                              \eta_{f,partiallywet} fin efficiency of a partially wet fin
F
           correction factor
           Fin pitch (m)
F_{p}
                                                                                         wavy angle (°)
           heat transfer coefficient (W m<sup>-2</sup> K<sup>-1</sup>)
                                                                                         dynamic viscosity of air (Pas)
h
                                                                             μ
i
           enthalpy (J kg<sup>-1</sup>)
                                                                                         density (kg m^{-3})
                                                                             ρ
\Delta i_m
           logarithmic-mean enthalpy difference (J kg<sup>-1</sup>)
                                                                             Subscripts
           Colburn factor
                                                                                         air
                                                                             а
k
           thermal conductivity, (W m<sup>-1</sup> K<sup>-1</sup>)
                                                                                         actual
                                                                             act
           mass flow rate (kg s<sup>-1</sup>)
m
                                                                                         average value
                                                                             ave
Ν
           number of intermittent operations
                                                                                         aluminum
                                                                              Al
N_t
           number of longitudinal tube rows
                                                                             C_{11}
                                                                                         copper
Δр
           pressure drop (Pa)
                                                                                         experiment
                                                                             exp
P_1
           longitudinal tube pitch (m)
                                                                             f
                                                                                         fin
Pr
           Prandtl number
                                                                             i
                                                                                         inner
P_t
           transverse tube pitch (m)
                                                                             in
                                                                                         inlet
           heat transfer rate (W)
Q
                                                                                         mean value
                                                                             m
           Reynolds number based on tube collar diameter
Re_{Dc}
                                                                             0
                                                                                         outer
           Reynolds number based on tube inside diameter
Re<sub>Di</sub>
                                                                                         outlet
                                                                             Out
T
           temperature (K)
                                                                                         tube
                                                                              р
           velocity (m s<sup>-1</sup>)
и
                                                                              w
                                                                                         water
```

#### 1. Introduction

Room air conditioners with finned tube evaporators are widely used and can be used over a long period (e.g. 10 years). The performance of such air conditioners may obviously deteriorate in its life cycle, and the deterioration may mainly result from the degradation of evaporator performance. The refrigerant circuit of an air conditioner is closed and the performance variation of the refrigerant-side is very small compared with that of the air-side. Therefore, the investigation on the long-term performance of finned tube evaporators should be focused on the variation of the air-side performance, including heat transfer and pressure drop.

Most of room air conditioners are on-off controlled and operated intermittently. After long-term intermittent operations, the air-side heat transfer and pressure drop performance of a finned tube evaporator changes obviously. The air-side heat transfer performance is affected by the thermal contact resistance between tubes and fins, and the thermal resistance between the airflow and fins. For a new and well made evaporator, the thermal contact resistance between tubes and fins is quite limited (Jeong et al., 2004, 2006), and it is much smaller than the thermal resistance between airflow and fins. However, for an intermittently operated air conditioner, the thermal contact resistance between tubes and fins becomes more and more important with the increase of operation time. Therefore, both the variation of the thermal contact resistance between the airflow and fins and the

thermal resistance between tubes and fins contribute to the long-term air-side heat transfer performance of a finned tube evaporator under intermittent operations. At the same time, the hydrophilicity of fin surfaces may degrade after the long-term intermittent operations, resulting in a pronounced change of the air-side heat transfer and pressure drop performance (Min et al., 2000).

There is no published research directly focusing on the long-term air-side heat transfer performance of finned tube heat exchangers. However, there are researches on the contact angle of fins after long-term operations (Hong, 1996; Min et al., 2000; Min and Webb, 2002). These researches show that the contact angle is affected by dry/wet cycles under intermittent operation conditions, and the effect of dry/wet cycles on contact angle of fins with hydrophilic coating is smaller than that without hydrophilic coating. Considering the relationship between the contact angle and the air-side heat transfer performance, it can be concluded from these researches that the air-side heat transfer performance of finned tube heat exchangers will change after long-term intermittent operations.

The effects of plasma treatment (Kim et al., 2007), dust deposition (Yang et al., 2007) and microorganism growth (Pu et al., 2009) on the long-term pressure drop performance of finned tube heat exchangers have been studied. However, these studies do not reflect the influence of intermittent operations.

The purpose of this study is to experimentally investigate the variation of air-side heat transfer and pressure drop of

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