



On the use of bend–twist coupling in full-scale composite marine propellers for improving hydrodynamic performance



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ARTICLE INFO

Article history:

Received 27 May 2015

Accepted 3 November 2015

Available online 23 December 2015

Keywords:

Composite propeller

Hydrodynamic efficiency

Fluid–structure interaction

Bend–twist coupling

Viscous flow

ABSTRACT

Marine propellers are designed to work for a particular operating condition. However, a propeller often requires to operate at different off-design conditions, when its hydrodynamic efficiency drops. In this paper, a comprehensive numerical study is presented on the use of bend–twist coupling of composite propeller blades for improving their hydrodynamic efficiency at off-design conditions. The analysis is carried out on a full-scale propeller of diameter 4.2 m, considering the complete viscous turbulent flow, as the loading and deformation of model propellers that have been typically studied in literature for this purpose cannot be extrapolated to a full-scale prototype propeller. The open water performance is estimated using the finite volume method employing the pressure based RANS equation for the steady, incompressible, turbulent flow. The deformation analysis is done using the finite element method based on the first order shear deformation theory for composite laminates. The fluid–structure interaction is incorporated in an iterative manner. The effect of laminate configurations on the maximum twist achieved in the blade is studied for four different composite materials. The numerical study reveals that, within the limits of material safety, the twist generated in the deformed propeller using commonly used composite materials is inadequate to create any noticeable change in the hydrodynamic efficiency. When the material failure is ignored, however, it is possible to generate sufficient deformation and twist that can cause appreciable improvement in the hydrodynamic performance.

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1. Introduction

A propeller is generally designed to operate for a particular velocity of advance and revolutions per minute, where it delivers the maximum hydrodynamic efficiency. In many situations, propellers are required to operate at off-design conditions, when its efficiency is compromised. For example, tugs and trawlers may or may not pull loads, ships may move in high sea conditions or a frigate may operate at a fast attack mode or a cruising speed. The angle of attack of the incoming flow to the propeller depends on the ratio of velocity of advance and its rotational speed. The pitch of the propeller blade is so designed that it receives flow at the optimum angle of attack. At off-design conditions, the angle of attack becomes sub-optimum and its hydrodynamic efficiency drops. The pitch can be altered in a controllable pitch propeller and it can remain efficient at different operating conditions (Das et al., 2012). It is well known that the metallic propellers (fixed pitch) are

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Nomenclature			
A_{ij}	extensional stiffness matrix	T	thrust of propeller
a	length of plate	t	thickness of propeller blade
B_{ij}	coupling stiffness matrix	U_i	i th component of velocity
C	damping	U_∞	free-stream velocity
C_p	pressure coefficient	\bar{U}	time-averaged velocity vector
C_μ	constant of $k\epsilon$ turbulence model	u_x, u_y, u_z	displacements
D	diameter of propeller	u'	fluctuating component of velocity
D_{ij}	bending stiffness matrix	V	volume of element
E_i	Young's modulus along i th direction	v_f	volume fraction of fibre
F	force	\hat{u}_z	non-dimensionalised deflection
G_{ij}	shear modulus	X_T	tensile strength in fibre direction for composite lamina
h	thickness of plate	Y_T	tensile strength in normal to fibre direction for composite lamina
J	advance ratio	α	angle of attack
k	kinetic energy of turbulence	β	angle of resultant flow with propeller plane
K_t	coefficients of thrust	ϵ	rate of dissipation of turbulent kinetic energy
K_q	coefficients of torque	$\epsilon_i, \epsilon_{ij}$	normal and shear strains
L	number of lamina in composite laminate	κ	curvature
M_l	component of moment/twist per unit length	θ	angle of orientation of fibre for composite
$[M]$	mass matrix	η	efficiency of propeller
N_i	components of force resultants per unit length	μ	coefficient of viscosity
$[N]$	shape function	ν_{ij}	Poisson's ratio
n	revolution per second for propeller	Ω	rotational velocity vector
P	pitch of propeller blade	ϕ_i	rotational displacement in i th direction for plate
p	pressure	ψ	pitch angle
\bar{p}	time averaged pressure	ρ	density of medium
Q	torque of propeller	σ_1, σ_2	normal stress in material direction in a composite lamina
Q	stiffness	σ_x, σ_y	normal stress in respective direction
\bar{Q}	stiffness for generally orthotropic condition	τ_{12}	shear stress in material direction in a composite lamina
Q^e	element stiffness matrix	τ_{xy}	shear stress component
Q_g^e	element geometrical stiffness		
\mathbf{r}	position vector		
S_{U_i}	source term in transport equation		
S	shear strength of composite lamina		

rigid enough to hold its shape under hydrodynamic loading and can only give the maximum efficiency for a particular operating condition. In layered fibre reinforced composite materials, however, the fibre orientations of the layers can be designed so as to generate bending–twisting coupling. This coupling can be used to get a twist of the propeller blade from the bending deformation under the hydrodynamic loading. The twist alters the pitch of the propeller and hence the angle of attack. This paper presents a comprehensive numerical study of the response of composite propellers at different operating conditions with a view to critically examine the effect of twisting of blades due to the bend–twist coupling on the hydrodynamic efficiency, and to explore if it can be used to improve the efficiency at off-design conditions.

The analysis of composite propellers to exploit the advantage of its flexibility and tailorability began towards the end of last century. Lin (1991) presented a numerical analysis of a three-dimensional (3-D) composite marine propeller using the commercial finite element method (FEM) software ABAQUS for the stress analysis and the PSF-2 code based on the vortex-lattice method (VLM) for computing the hydrodynamic loading. No effect of the deformation on the hydrodynamic loading was considered. The stresses and hydrodynamic performance were compared with those of a geometrically identical isotropic propeller. Lin and Lin (1996) presented a coupled 3-D FEM/VLM method for the hydroelastic analysis of metallic marine propellers, considering geometrical non-linearity for estimating the deformed shape of the propeller. Lee and Lin (2004) presented an optimum design of composite propeller blades for operating at design as well as off-design conditions, considering the effect of their deformation. The PSF-2 code was used for the fluid flow analysis and the FEM for deformation analysis. They used graphite-epoxy laminates composed of 0°, 45° and 90° plies and determined the optimum sequence of plies for maximising the twist of the blade, which in turn affects the torque requirement of the propeller. Their study, however, is made on a 0.305 m diameter model propeller. It ignores the viscosity and does not check for material failure. Extensive studies of composite propellers have been carried out by Young (2008). He used a panel based BEM method for the flow analysis, instead of the VLM used by his predecessors. The hydrodynamic pressure distribution due to rigid blade rotation is computed via the BEM, which is then applied as external normal surface traction for the FEM model to obtain the

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