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Monthly assessments of exergetic, economic and environmental criteria and optimization of a solar micro-CCHP based on DORC



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ABSTRACT

This research performs an analytical simulation of a novel solar-driven combined cooling, heating and power (CCHP) system based on a dual organic Rankine cycle (DORC) using heat transport fluids of water and Copper (II) oxide (CuO)/water nanofluid. The major design parameters are encompassed to evaluate the monthly thermodynamic efficiencies, total product cost and environmental impact (EI) rates of the system for four selected organic fluid groups, namely R134a-R245fa, R1234yf-R245fa, R1234ze-R236fa and R423A-R236fa. According to results, a considerable positive influence is observed in the energy and exergy efficiencies within 16.71% and 24.34%, respectively for R1234yf-R245fa in November. The lowest EI rate is achieved by 1.2% for R1234ze-R236fa in April as turbine 1 inlet pressure increases and the highest decrement in EI rate is calculated within 2.48% for R1234yf-R245fa in September with an increase in turbine 2 back pressure. In addition, two well-known decision making techniques comprising LINAMP and TOPSIS are applied to identify the ultimate optimal performance of the system among the solutions gained from the non-dominated sorting genetic algorithm (NSGA-II) approach. Referring to the optimization results, the maximum improvement in the exergy efficiency and product cost can be obtained within 33.49% through the LINMAP method and 23.67% through the TOPSIS procedure, respectively and the highest improvement in the EI rate is calculated by about 22.7% with 0.01788 nanoparticle volume fraction for R134a-R245fa through the LINMAP method.

1. Introduction

Global warming and depletion of fossil fuels are two major concerns for providing energy. More efficient energy systems and intensive utilization of renewable energy sources are required for the sustainability of energy systems in the future. Combined cooling, heating and power (CCHP) system is one of the most promising technologies producing multi-demand of energy for a customer, simultaneously. This ability of CCHP systems causes higher efficiency, lower fuel consumption and consequently lower emission of pollutants. In addition, incorporating the solar energy technologies to the CCHP system is an effective solution to mitigate the pollutant emissions to the environment.

Flat plate solar collectors are the assuring energy adaptation technologies in the ground of low rank heat employment. The thermal efficiency of flat plat solar collectors are considerably low due to the low convective heat transfer coefficient between the absorber and the heat transfer medium and more heat losses to the environments. One of the effective methods to increase the thermal performance is to replace the working fluid with nanofluids. Several investigations have been carried out on the effects of applying nanoparticles, i.e. Alumina (Al_2O_3), Zinc

oxide (ZnO), Silicon dioxide (SiO₂), Titanium oxide (TiO₂) and Copper (II) oxide (CuO) on the thermal efficiency, heat transfer coefficient, size reduction of the flat plat collector and its associated energy and cost saving (Alim et al., 2013; Faizal et al., 2013; Yousefi et al., 2012).

On the other hand, ORC is the beneficial subsystem that can be used in CCHPs for mechanical power production and, subsequently, electrical power production through an electrical generator. ORC is usually used when a low- or medium-temperature energy source is available (Al-Sulaiman et al., 2012). Providing cooling effect in CCHPs can be done by utilizing an absorption refrigeration cycle (Moradi and Mehrpooya, 2017; Wang et al., 2016; Wang and Yang, 2016) or an ejector refrigeration cycle (Boyaghchi and Heidarnejad, 2015; Ebrahimi and Keshavarz, 2015). The ejector refrigeration cycles are more effective systems because the ejector with the simple configuration recovers the expansion process losses and lessens the compression power (Disawas and Wongwises, 2004; Harrell and Kornhauser, 1995; Kornhauser, 1990; Sarkar, 2012; Wongwises and Disawas, 2005).

Various single loop ORC-CCHP configurations fueled by solar energy have been proposed and modeled using the energy and exergy concepts.

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Nomenclature		β	collector tilt angle (°)
	2	ρ	density (kg/ m ³)
Α	area (m²)	$ ho_G$	ground albedo
b	environmental impact per unit of exergy (Pts/kJ)	ω	life cycle inventory associated with the production (mPts
\dot{B}	environmental impact associated with an exergy stream		kg)
	rate (Pts/s)	Φ	maintenance factor
c	cost per unit of exergy (\$/kJ)	η	efficiency (%)
Ċ	cost rate (\$/s)	φ	nanoparticles volume fraction
C_P	specific heat (kJ/kg K)	δ	thickness (m)
Cl	clearness index	J	there is a second of the secon
CRF	capital recovery factor	Subscript	•
di	deviation index	Subscripts	
	distance of each criterion from the ideal point	0	11
d_m^+	•	0	dead state
d_m^-	distance of each criterion from the non-ideal point	a	air
EI	environmental impact	back	back
ex	specific exergy (kJ/kg)	bf	base fluid
Ex	total exergy flow rate (kW)	C	collector
F	criterion	Con	condenser
F'	collector efficiency factor	cooling	cooling load
F_R	heat removal factor	D	destruction
Gt	total radiation rate falling on the titled collector (W/m²)	e	exiting stream
G _b	beam irradiation rate falling on the tilted collector (W/m ²)	edge	edge
G_d	diffuse irradiation rate falling on the tilted collector (W/	Ejc	ejector
O _d	m ²)		
11		Eva	evaporator
H	terrestrial average monthly irradiation on a horizontal	ex	exergy
	surface, (kWh/m² day)	F	fuel
h	specific enthalpy (kJ/kg)	glass	glass cover
hr	annual number of operation hour (h)	HE	heat exchanger
h_w	wind convection heat loss coefficient (W/m ²)	heating	heating load
i	interest rate (%)	i	entering stream
k	thermal conductivity (W/m K)	ins	insulation
L	collector length (m)	k	kth component
N	system lifetime (year)	L	loss
M	mass of equipment (kg)	m	mth solution existing on the Pareto frontier
m	mass flow rate (kg/s)		
P	pressure (kPa)	n	nth criterion
	=	net	net
Q	heat transfer rate (kW)	nf	nanofluid
R_b	beam radiation tilt factor	np	nanoparticle
S	absorbed solar radiation (W/m²)	P	product
SH	shape factor	pump	pump
T	temperature (K)	plate	absorber plate
t	time (s)	q	number of points on Pareto frontier
U	overall heat transfer coefficient (W/K m²)	r	number of criteria
V	velocity (m/s)	ST	storage tank
\dot{W}	power (kW)	sun	sun
Y	component-related EI (Pts)	th	
Ϋ́	component-related EI rate (Pts/s)		energy
Ż	capital investment cost rate (\$/s)	top	top
_		Tur	turbine
Greek l	etters	u	useful
J. COR 1		Superscri	nte
(τα)	fraction of absorbed radiation	superscri	νω
ε	emissivity		
	Stefan–Boltzman constant, $5.6697 \times 10^{-8} \text{ W/m}^2 \text{ K}^4$	•	rate
σ		tot	total
ϕ	solar zenith angle (°)	En	Euclidian non-dimensionlization procedure
θ	incident angle (°)		
τ	transitivity		

For instance, Al-Sulaiman et al. (2011) developed a new solar CCHP based on the single loop ORC and the single-effect absorption chiller to produce the cooling effect. The proposed system was examined using three modes of operation. They were: solar mode during the low-solar radiation time of the day, solar and storage mode during the high solar radiation time of the day, and storage mode during night time. The

exergy efficiencies and exergy destruction rates were examined under the variation of the ORC evaporator pinch point temperature, ORC pump inlet temperature, and turbine inlet pressure. Wang et al. (2015) used the energy analysis to study and optimize a new ORC-CCHP system equipped with the flat plat solar collector which operated in three modes, namely power mode, combined heat and power mode,

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