

Available online at www.sciencedirect.com

## **ScienceDirect**

journal homepage: www.elsevier.com/locate/issn/15375110



## Research Paper

## Development of stability indicators for dynamic Phase I overturn of conventional farm tractors with front axle pivot



Zhen Li <sup>a</sup>, Muneshi Mitsuoka <sup>b,\*</sup>, Eiji Inoue <sup>b</sup>, Takashi Okayasu <sup>b</sup>, Yasumaru Hirai <sup>b</sup>

#### ARTICLE INFO

Article history:
Received 24 October 2014
Received in revised form
26 March 2015
Accepted 30 March 2015
Published online

#### Keywords:

Conventional farm tractor Front axle pivot 3D mathematical model Dynamic Phase I overturn Tractor stability indicator Critical speed Tractor overturns are serious potential hazards for operators. While rollover protective structures (ROPS) protect operators passively, greater protection can be achieved through theoretical prediction of a potential overturn. Given effective warning, an operator can act to correct a tractor's motion when a tyre is about to lose contact with the ground. Such a loss of contact is associated with the initiation of a Phase I tractor overturn. However, it remains unclear how the initiation of tractor overturn is influenced by certain factors. Furthermore, the current mathematical models for tractors should be further extended for general utilisation. This study was conducted to develop stability indicators based on a more general model for dynamic Phase I tractor overturn. We considered practical tractor configurations and motion characteristics in a threedimensional (3D) reference frame in formulating the mathematical model. Tractor stability indicators for overturn and sideslip were derived from force calculations. A parametric study was conducted using an example tractor. The tractor speed and slope angle were found to affect the overturning stability significantly. The coefficient of maximum static friction was found to be the main factor contributing to tractor sideslip. Critical tractor speeds for various ground conditions were identified by considering the zero values of the tractor stability indicators. The critical tractor speed was determined as a function of the maximum static friction and the slope angle. By providing a display device based on ergonomics principles, the results of this study can be further implemented in the form of guidance to operators.

© 2015 IAgrE. Published by Elsevier Ltd. All rights reserved.

<sup>&</sup>lt;sup>a</sup> Graduate School of Bioresources and Bioenvironmental Sciences, Kyushu University, 6-10-1 Hakozaki, Higashi-ku, Fukuoka 812-8581, Japan

<sup>&</sup>lt;sup>b</sup> Faculty of Agriculture, Kyushu University, 6-10-1 Hakozaki, Higashi-ku, Fukuoka 812-8581, Japan

<sup>\*</sup> Corresponding author. Tel.: +81 926422929.

Nomeno	lature	$f_{f\_limit}$	maximum static friction force acting on the front tyres in the Y direction [N]
Symbols		$F_{py}$	reaction force between the two main tractor parts
α	lateral slope angle [rad]	РУ	in the Y direction [N]
$\theta$	pitch angle in local coordinates [rad]	$F_{pz}$	reaction force between the two main tractor parts
$\theta_{\mathtt{1}}$	initial angle between the horizontal line and the	P2	in the Z direction [N]
-	line connecting the COG <sub>r</sub> and the front axle pivot,	$f_r$	lateral friction force acting on the rear tyres in the
	from side view [rad]	<i>J.</i>	Y direction [N]
$\theta_2$	initial angle between the horizontal line and the	$f_{r\_limit}$	maximum static friction force acting on the rear
_	line connecting the COG <sub>r</sub> and a modelled rear tyre	J	tyres in the Y direction [N]
	(spring–damper unit), from side view [rad]	$F_{si}$	static ground supporting force on tyre i in the Z
μ	coefficient of maximum static friction between		direction for non-sloping conditions [N]
,	the tyre and the ground in a direction parallel to	$F_{zi}$	dynamic ground supporting force on tyre i in the Z
	the rotational axis of the tyre [–]	2.	direction [N]
π	circumference ratio, Pi [–]	g	acceleration due to gravity [m s <sup>-2</sup> ]
Φ	roll angle in global coordinates [rad]	H <sub>f</sub>	height of the COG <sub>f</sub> [m]
$arphi_1$	initial angle between the front axle and the line	$h_f$	length of the line segment between the COG <sub>r</sub> and
, ,	connecting the front axle pivot and a modelled	J	the tractor front axle pivot, from side view [m]
	front tyre (spring-damper unit), from back view	$H_r$	height of the COG <sub>r</sub> [m]
	[rad]	$h_r$	length of the line segment between the COG <sub>r</sub> and
$\varphi_2$	initial angle between the horizontal line and the	,	the point connecting the tractor main body (rear
12	line connecting the COG <sub>r</sub> and a modelled rear tyre,		end) and a modelled rear tyre, from side view [m]
	from back view [rad]	$i_o$	tractor stability indicator against overturn [–]
$arphi_f$	roll angle of the tractor anterior part in local	is	tractor stability indicator against sideslip [–]
. ,	coordinates [rad]	$I_{xf}$	tractor moment of inertia about the $x_f$ axis, i.e., the
$\varphi_r$	roll angle of the tractor posterior part in local	λ)	longitudinal axis of the front end of a tractor
	coordinates [rad]		[kg m <sup>2</sup> ]
$\psi$	yaw angle in local coordinates [rad]	$I_{xr}$	tractor moment of inertia about the $x_r$ axis, i.e., the
$\omega_{\mathrm{X}}$	component of tractor angular velocity about the X	Al .	longitudinal axis of the main body (rear end) of a
21	axis [rad s <sup>-1</sup> ]		tractor [kg m <sup>2</sup> ]
$\omega_{ m Y}$	component of tractor angular velocity about the Y	$I_{yr}$	tractor moment of inertia about the $y_r$ axis, i.e., the
	axis [rad s <sup>-1</sup> ]	,	axis transversal to the motion direction of a
$\omega_{\mathrm{Z}}$	component of tractor angular velocity about the Z		tractor, passing through the COG of its main (rear)
	axis [rad s <sup>-1</sup> ]		end [kg m²]
Α	coordinate transformation matrix from local to	$j_f$	perpendicular distance from the point connecting
	global coordinates [–]		the tractor front end and a modelled front tyre to
$A^{T}$	transpose of the matrix A [–]		the longitudinal axis through the front axle pivot
$A_X$	transformation matrix due to rotation about the X		[m]
	axis [–]	j <sub>r</sub>	perpendicular distance from the point connecting
$A_{Y}$	transformation matrix due to rotation about the Y		the tractor main body and a modelled rear tyre to
	axis [–]		the longitudinal axis through the front axle pivot
$A_Z$	transformation matrix due to rotation about the Z		[m]
	axis [–]	$k_i$	vertical spring constant of tyre i [N m <sup>-1</sup> ]
$c_i$	vertical viscous damping coefficient of tyre i	L	length of the obstacle [m]
	$[{ m N \ s \ m^{-1}}]$	$L_1$	effective obstacle length during the obstacle-
D	height of the obstacle [m]		surmounting passage of the front wheel(s) [m]
$e_{fy}$	lateral distance between the $COG_f$ and the pivot	$L_2$	effective obstacle length during the obstacle-
	axis [m]		surmounting passage of the rear wheel(s) [m]
$e_{fz}$	vertical distance between the COG <sub>f</sub> and the pivot	$\mathbf{L}_f$	horizontal distance between the COG and the
	axis [m]		front wheel [m]
$e_{\mathrm{G}}$	vertical distance between the COG and the pivot	$l_f$	horizontal distance between the $COG_r$ and the
	axis [m]		front wheel [m]
$e_{ry}$	lateral distance between the COG <sub>r</sub> and the pivot	$L_r$	horizontal distance between the COG and the rear
	axis [m]		wheel [m]
$e_{rz}$	vertical distance between the COG <sub>r</sub> and the pivot	$l_r$	horizontal distance between the $COG_r$ and the rear
	axis [m]		wheel [m]
$f_f$	lateral friction force acting on the front tyres in the	m	total mass of the tractor [kg]
	Y direction [N]	$m_f$	mass of the tractor front end [kg]

### Download English Version:

# https://daneshyari.com/en/article/8055165

Download Persian Version:

https://daneshyari.com/article/8055165

**Daneshyari.com**