



## Wick type solar stills: A review

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### ABSTRACT

Solar distillation is one of the water purification techniques that produce ultrapure water which is superior to most of the commercial bottled water sources. Though solar distillation is a simple method, productivity seems to be low due to the large thermal capacity and consumption of time. Researchers have taken efforts to make different designs of solar still for higher distillate yield and inferred that wick-type solar stills are effective and efficient. In this review, we are making an attempt to study the present status of different designs of wick type solar stills.

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### 1. Introduction

People prefer technologies whenever they find something lacking in their life. Solar distillation is one such technology, which they found useful as a solution for their drinking water shortages. Though the availability of global water reserves is about 1.4 billion km<sup>3</sup>, the sea water constitutes about 97.5%. Only the remaining 2.5% is gifted as fresh water for human beings and other living organisms. The available amount of fresh water in the

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**Nomenclature**

$\Delta Y$	thickness of water layer, (m)	$T_g, T_{w1}, T_{wf}$	temperature of glass, tilted, floating-wick, ambient ( $^{\circ}\text{C}$ )
$\rho_w$	density of water, ( $\text{kg}/\text{m}^3$ )	$\alpha_{w1}$	absorptivity of blackened tilted-wick surface
$c_{pw}, c_w$	specific heat capacity of water ( $\text{J}/\text{kg K}$ )	$m_{w1}$	mass flow rate along the tilted-wick along the length $L_1$ ( $\text{kg}/\text{s}$ )
$\Delta\tau$	time interval selected (2 min), (m/s)	$\bar{T}_g, \bar{T}_{w1}, \bar{T}_{wf}$	average temperature of the glass, tilted and floating-wick surfaces ( $^{\circ}\text{C}$ )
$U_2$	overall heat transfer coefficient between bottom wall and surrounding ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$L_1, L_2$	length of the tilted and floating-wick stills (m)
$A_s, A$	area of the still (m)	$h_4$	convective and radiative heat transfer coefficient from flowing water to ambient ( $\text{W}/\text{m}^2$ )
$A_m$	Area of the mirror ( $\text{m}^2$ )	$\theta_{evap}$	evaporative heat transfer from flowing water to the ambient ( $\text{W}/\text{m}^2$ )
$m_i$	hourly productivity (l)	$m_{fw}$	mass of the flowing water over the glass cover ( $\text{kg}/\text{s}$ )
$P_d$	daily productivity ( $\text{l}/\text{m}^2$ day)	$T_{fw}$	temperature of the flowing water over the glass cover ( $^{\circ}\text{C}$ )
$kw$	thermal conductivity of water ( $\text{w}/\text{m K}$ )	$A_{wf}, A_{w1}$	area of floating and tilted-wick surfaces ( $\text{m}_2$ )
$h_1$	total heat transfer from water to glass cover ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$h_{ewfa}$	evaporative heat transfer coefficient form floating-wick to ambient ( $\text{W}/\text{m}^2$ )
$h_2$	heat transfer from floating-water to ambient ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$Q_{sun, re}, Q_{sun, re}, Q_{sun, df}$	rate of absorption of reflected, direct and diffused solar radiation (W)
$h'_2$	heat transfer coefficient from glass cover to flowing water ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$G_{dr}, G_{df}$	direct and diffused solar radiation on a horizontal surface ( $\text{W}/\text{m}^2$ )
$T_g, T_w, T_a$	glass, water and ambient temperatures ( $^{\circ}\text{C}$ )	$Q_{sun, g}, Q_{sun, w}$	absorption of solar radiation on glass and wick surfaces ( $\text{W}/\text{m}^2$ )
$\bar{T}_g$	average water temperature ( $^{\circ}\text{C}$ )	$\tau_g$	transmittance of glass cover
$\bar{T}_g$	average glass temperature ( $^{\circ}\text{C}$ )	$\beta$	incident angle of sunrays to glass cover ( $^{\circ}$ )
$T_{w1}$	temperature of the flowing water over the glass cover ( $^{\circ}\text{C}$ )	$\beta'$	incident angle of reflected sunrays to glass cover ( $^{\circ}$ )
$m_w$	mass flow rate of water over the glass cover ( $\text{kg}/\text{s}$ )	$\rho_m$	reflectance of reflector
$\tau$	fraction of energy absorbed by water in still	$\alpha_w$	absorptance of the wick
$L, l$	latent heat of evaporation ( $\text{J}/\text{kg }^{\circ}\text{C}$ )	$w$	width (m)
$\epsilon_{\omega}$	emissivity of water surface	$m_{cp}$	heat capacity ( $\text{J}/\text{K}$ )
$\sigma$	Stefan–Boltzmann constant, $5.669 \times 10^{-8}$ ( $\text{w}/\text{m}^2\text{ }^{\circ}\text{C}^4$ )	$l_s$	length of the still (m)
$m_{ew}$	amount of water distillate per unit time per unit basin area ( $\text{kg}/\text{m}^2\text{ s}$ )	$Q_{ci}, Q_{ei}, Q_{ri}$	convective, evaporative and radiative heat transfer from water to glass cover ( $\text{W}/\text{m}^2$ )
$q_u$	the rate of heat loss due to water flow	$h_{ci}, h_{ri}$	convective and radiative heat transfer coefficient from glass cover to ambient ( $\text{W}/\text{m}^2$ )
$P_w, P_g$	saturated vapour pressure of water and glass ( $\text{N}/\text{m}^2$ )	$Si$	computation ratio (heat of evaporation to total heat transferred)
$h_{rw}, h_{ew}, h_{cw}$	radiative, evaporative and convective heat transfer coefficient ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$Q_{ac}$	heat transferred from ambient to glass cover ( $\text{W}/\text{m}^2$ )
$\eta_o$	overall efficiency of the distillate output	$Q_{be}$	conduction heat transfer through base ( $\text{W}/\text{m}^2$ )
$\alpha_g$	absorptance–transmittance product of glass cover	$h_{ca}$	heat transferred from glass cover to atmosphere by convection ( $\text{W}/\text{m}^2$ )
$b_3, b_3, b_1$	breadth of the glass cover, tilted, floating-wick surfaces (m)	$h_{\theta}$	latent heat ( $\text{kJ}/\text{kg}$ )
$dx_3, dx_2, dx_1$	length of the glass cover, tilted, floating-wick surfaces (m)	$H_s, I(t), S$	incident solar radiation on glass cover per unit area per unit time ( $\text{W}/\text{m}^2$ )
$h_1$	total heat transfer coefficient from tilted-wick surface to glass ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )	$M$	mass of the water (kg)
$h_2$	total heat transfer coefficient from floating-wick surface to glass ( $\text{W}/\text{m}^2\text{ }^{\circ}\text{C}$ )		
$h_3$	total heat transfer from glass surface to ambient ( $\text{W}/\text{m}^2$ )		

form of rivers, lakes, surface water, polar ice, ground water etc., cannot match the water requirement for the entire globe. Hence solar distillation can be considered as a practical alternative for the production of drinking water.

**2. Working of a wick type solar still**

The solar radiation falling on the glass cover transmits through it and reaches the wick surface, where it is absorbed. A part of the energy is utilized for heating the water flowing through the wick due to capillary action. A large amount of heat gets trapped inside the still, and transfer of energy takes place from the wick surface

to the glass cover and to the ambient air. Heat transfers in the distillation system are governed by external and internal modes. The external heat transfer mode occurs due to convection and radiation, which are independent of each other and occurs outside the still. Heat transfer within the solar distillation unit is referred to as the internal heat transfer mode, which occurs due to radiation, convection and evaporation. In internal heat transfer mode, the mass transfer accompanied with radiative and convective heat transfer. Water flowing through the wick surface gets heated and evaporated into vapors. The saturated water vapour condenses in the lower surface of the glass cover after releasing the latent heat of vaporization. The condensed water droplets trickle down due to gravity and get collected in the drainage channel.

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