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A simple method for estimating transition locations on blade surface of model propellers to be used for calculating viscous force

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Abstract

Effects of inflow Reynolds number (*Re*), turbulence intensity (*I*) and pressure gradient on the transition flow over a blade section were studied using the γ -Re θ transition model (STAR-CCM+). Results show that the Re_T (transition Re) at the transition location (P_T) varies strongly with *Re*, *I* and the magnitude of pressure gradient. The Re_T increases significantly with the increase of the magnitude of favorable pressure gradient. It demonstrates that the Re_T on different blade sections of a rotating propeller are different. More importantly, when there is strong adverse pressure gradient, the P_T is always close to the minimum pressure point. Based on these conclusions, the P_T on model propeller blade surface can be estimated. Numerical investigations of pressure distribution and transition flow on a propeller blade section prove these findings. Last, a simple method was proposed to estimate the P_T only based on the propeller geometry and the advance coefficient.

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Keywords: Transition flow; Viscous force; Strip method; Propeller performance

1. Introduction

So far, the performance prediction of propulsion systems is mainly based on the results of open water tests in model scale. Due to the limitation of experimental conditions, model propellers cannot be as large as the full scale propeller, and the rotation rate is also limited. The Reynolds number (Re) of model propellers (based on 0.75R blade section, where R is the radius of the propeller) is much smaller than that of the full scale propellers, which makes the boundary layer flow of the model propeller very different from that of the full scale propeller. The different boundary layer flows make different viscous forces and different propulsion performances, known as scale effects. To obtain accurate performance of the full

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scale propeller, the measured data of the model propeller need to be scaled.

The study of propeller scale effects has been carried out for decades. The key is to accurately predict the viscous force of the model propeller with complex boundary layer flows because of the low Re. There are several scaling methods to scale the measured model propeller data to full scale propeller performance, and four of them are frequently used today: no scaling, the Lerbs-Meyne method (Meyne, 1968), the 1978 ITTC scaling method (ITTC, 1978) and the strip method (Praefke, 1994; Streckwall et al., 2013). A brief introduction about the four scaling methods is available in Helma (2015). In the strip method, the vector sum of contributions of each radial section (strip) towards the friction resistance is calculated to get the friction resistance of the whole blade. Theoretically, the strip method is a relatively accurate approach of the existing scaling methods (the 1978 ITTC scaling method is only based on the friction of one blade section). However, the calculation of the viscous force needs the transition location

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Nomenclature

C and $C_{0.75R}$ chord length of the airfoil and the 0.75R	
propeller blade section	
C_f local friction coefficient C_p local pressure coefficient C_T length from the leading edge to the transition	
C_p local pressure coefficient	
C_T length from the leading edge to the transition point P_T	
<i>D</i> diameter of the propeller	
<i>I</i> turbulence intensity = $\sqrt{(2/3)k/\overline{v}}$	
J advance coefficient	
<i>k</i> turbulent kinetic energy	
K_T propeller thrust coefficient	
K_Q propeller torque coefficient	
L^{\sim} length of the flat plate	
<i>n</i> rotational rate of the propeller	
<i>P</i> pitch of a blade section	
P_T transition location, where the flow changes from	l
laminar to turbulent region	
<i>R</i> radius of the propeller	
<i>r</i> radius of a blade section, $r = \eta R$, η is between 0 and 1	
<i>Re</i> inflow Reynolds number to the flat plate and the	
propeller	,
$Re_{0.75R}$ inflow Reynolds number to the 0.75R propeller	•
blade section	
Re_T local Reynolds number at the transition location	
P_T	
T turbulent time scale	
u_a and u_t induced axial velocity and circumferential ve-	
locity by the propeller	
Z blade number	
$\overline{\nu}$ the mean reference velocity	
V_A advance velocity for the propeller	
V_A and V_{θ} axial velocity and tangential velocity in pro-	
v_x and v_θ axial velocity and tangential velocity in pro-	
V_n relative circumferential velocity = $2\pi nr$	
V_R resultant velocity of u_a , u_t , V_A and V_n	
α_K and α'_K angle of attack and approximated angle of	?
attack	
θ pitch angle	
β advance angle	
β_i hydrodynamic pitch angle	
ρ density of water	
μ_t turbulent viscosity	
μ natural molecular viscosity of water	
ν kinematic viscosity	
ε and ω turbulent dissipation rate in turbulence models	
γ intermittency, for trigging the transition	
$Re\theta$ momentum thickness Reynolds number	
$\varepsilon(K_T)$ and $\varepsilon(10K_Q) \varepsilon(K_T) = (K_{T,\text{Fine}} - K_{T, \text{Middle}})/K_T$,
	,
$\lambda(K_T)$ and $\lambda(10K_Q)$ $\lambda(K_T) = (K_{T,\text{Num}} - K_{T,\text{Exp}})/(K_{T,\text{Exp}})/(K_T,\text{Exp})$,
$\lambda(10K_Q) = (10K_{Q,\text{Num}} - 10K_{Q,\text{Exp}})/10K_{Q,\text{Exp}}$ <i>M-P-G</i> magnitude of pressure gradient = $\Delta C_p/\Delta x$	

the difference of local pressure coefficients of
two adjacent cells
the distance between two adjacent cells
turbulence viscosity ratio = μ_t/μ
adverse pressure gradient
favorable pressure gradient
experimental data
numerical results
shear stress transport
a turbulence model; applying the $k \boldsymbol{\epsilon}$ model in
the far-field for high speed flow region and the
$k\omega$ model near the wall for low speed flow
region
ble k ϵ a turbulence model; based on the standard
ke model; suitable for boundary layer flow
simulation
transition model; based on the SST-k ω model;
two additional transport equations need to be

 (P_T) on each radial section of the whole blade surface. Since the boundary layer transition flow on the blade surface is very complex because of too many factors affecting the transition, the transition locations on all sections of the whole blade calculated only based on two assumed transition Reynolds numbers (Re_T) (one for the blade face and the other for the blade back) remains questionable, even though two other assumed Re_T were applied considering the effect of turbulence intensity (Streckwall et al., 2013).

solved for predicting transition

Recently, with the development of the γ -Re θ transition model within the RANS code (Menter et al., 2006), the numerical study of the transition flow of model propellers was beginning to emerge in the past few years. In 2010, Müller et al. (2009) and Müller (2013) applied the transition model for studying the transition flow of a model propeller. In 2014, Sánchez-Caja et al. (2014) studied the scale effects on tip loaded propeller performance. In 2016, Bhattacharyya et al. (2015, 2016a, 2016b) used the transition model for studying the transition flow of a ducted propeller, and a scaling approach was proposed by way of regression. Generally, the γ -Re θ transition model gives us a powerful tool for a deep insight to the transition flow on the model propeller blade surface.

In this paper, the γ -Re θ transition model was applied for studying the transition flow of a model propeller, especially the transition Reynolds number (Re_T) and the P_T on the blade surface. The primary work was introduced as follows. First, the accuracy of the transition model was validated by simulating the transition flow of a flat plate and a two-dimensional airfoil. Numerical results were compared with available experimental data. Second, to simplify the research, the transition flow of a marine propeller was studied first by simulating that on a propeller blade section. The effects of Re, I(turbulence intensity) and the magnitude of pressure gradient on the transition flow were studied. Major conclusions about

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