

Energy decay of a microbeam model with a locally distributed nonlinear feedback control

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Abstract

In this paper we address the problem of internal stabilization of the deflection of a microbeam, which is modeled by a sixth-order hyperbolic equation. Employing multiplier techniques and an integral inequality, we prove that a locally distributed nonlinear feedback control forces the energy associated to the deflection to decay exponentially or polynomially to zero. As a consequence of this, the deflection goes to the rest position as the time goes to infinity.

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1 Introduction

A microbeam is a beam whose dimensions are in the order of a few microns. According to [26, Chapter 6], the microbeams are perhaps the most common structural component used in micro-electro-mechanical systems (MEMS) such as actuators, filters, resonators and sensors.

The deflection $z = z(t, x)$ of a clamped microbeam of density $\rho > 0$, cross-sectional area $A > 0$, Young's modulus $E > 0$, area moment of inertia $I > 0$, shear modulus $G > 0$ and length $L > 0$ being subjected to a distributed load $f = f(t, x)$ can be modeled by

$$\left\{ \begin{array}{l} \rho A z_{tt} + M_1 z_{xxxxx} - M_2 z_{xxxxxx} = f, \quad (t, x) \in (0, \infty) \times (0, L), \\ z(t, 0) = z_x(t, 0) = z_{xx}(t, 0) = 0, \quad t \in (0, \infty), \\ z(t, L) = z_x(t, L) = z_{xx}(t, L) = 0, \quad t \in (0, \infty), \\ z(0, x) = z_0(x), \quad x \in (0, L), \\ z_t(0, x) = z_1(x), \quad x \in (0, L). \end{array} \right. \quad (1.1)$$

This model has been derived in [12, Section 3] and [17, Section 4] by using the modified strain gradient elasticity theory developed in [19, Section 2] together with Hamilton's Principle. Here

$$M_1 = EI + GA \left(2l_0^2 + \frac{8}{15}l_1^2 + l_2^2 \right) \quad \text{and} \quad M_2 = GA \left(2l_0^2 + \frac{4}{5}l_1^2 \right),$$

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